



**RUPTURE OF MATERIALS AND STRUCTURES**  
**Mechanisms and modelling VS. industrial applications**  
**MECAMAT, AUSSOIS 2019**

**René BILLARDON — Safran Landing Systems**



# OUTLINE

- **Industrial environment**
  - ❑ Structures, systems, loadings
  - ❑ Certification procedures
  - ❑ High strength metallic materials
  - ❑ Prediction of the rupture of a structure under monotonic loading
- **Application to structures made of High Strength steels**
  - ❑ Constitutive model and ductile damage model
  - ❑ Prediction of the rupture
    - Local material rupture
    - Global instability
- **Other typical applications**
- **Some conclusions**

# 1



## INDUSTRIAL ENVIRONMENT

# AERONAUTICAL SYSTEMS



➤ **Engines**



- **Landing Gears, Wheels & Brakes**
- **Systems and Equipment**

# CERTIFICATION PROCEDURES

## Different strength requirements – Static (monotonic) loading

- **Landing Gear, Wheels & Brakes, Systems and Equipment**
  - ❑ **Fatigue loads**
  - ❑ **Limit loads:** maximal loads expected in service
    - No detrimental permanent deformation of the structure
  - ❑ **Ultimate loads:** each limit load case multiplied by a safety factor of 1.5
    - No rupture
  
- **Engines**
  - ❑ **Fatigue loads**
  - ❑ **A few limit loads** expected in service
  - ❑ **A few ultimate loading cases** due to particular adverse conditions
    - e.g. Fan Blade Out, turbine over-speed

# CERTIFICATION PROCEDURES

## Different strength requirements – Fatigue loading

- **Safe life approach**
  - No (detectable) crack in the structure in service
  
- **Damage tolerant approach**
  - To cope with anomalies occurring in-service or/and during maintenance operations
  - To assess the fatigue life of non Critical Structural Elements
  - Regular inspections so that the residual strength of the structure is capable of the limit loads

# CERTIFICATION PROCEDURES

## Different requirements – Tests and analyses

- **Landing Gear**
  - ❑ Certification by Type Certificate Holder (aircraft manufacturer) **by analysis supported by tests**
    - Hundreds of non-proportional load cases
- **Wheels & Brakes**
  - ❑ Qualification and Technical Standard Order **by tests supported by analyses**
  - ❑ Certification by Type Certificate Holder (aircraft manufacturer) **after flight tests**
- **Systems and Equipment**
  - ❑ All components certified **by tests supported by analyses** except for Critical Structural Elements (e.g. retraction actuator)
  - ❑ Integrated systems certified **by tests and analyses** mainly at aircraft level
- **Engines**
  - ❑ Certification **by analysis supported by tests**

# CERTIFICATION PROCEDURES

## Analysis vs. tests

### ➤ Analyses

- To deal with numerous complex load cases
- To deal with the scatter of material properties and the scatter of parts geometry

### ➤ Tests

- Tests at different scales to support analysis methods
- Full scale –detail or complete- tests to certify a particular system

### ➤ Past experience

- Positive lessons learnt from in-service systems
  - validate the methods that were used for their design
  - whatever their theoretical background
- Authorities strongly support the use of “classical” methods



# CERTIFICATION PROCEDURES

## Analysis vs. tests

- **A need for new analysis methods and tools**
  - ❑ To introduce new designs and new materials
  - ❑ To take best benefit of 3D analysis tools
  - ❑ To better control conservatisms to be decreased because of weight reductions
- **New analysis tools must not lead to a regression**  
**when applied to a system that has been already certified**
- **Models used to better understand mechanisms**  
**vs. models to design structures**

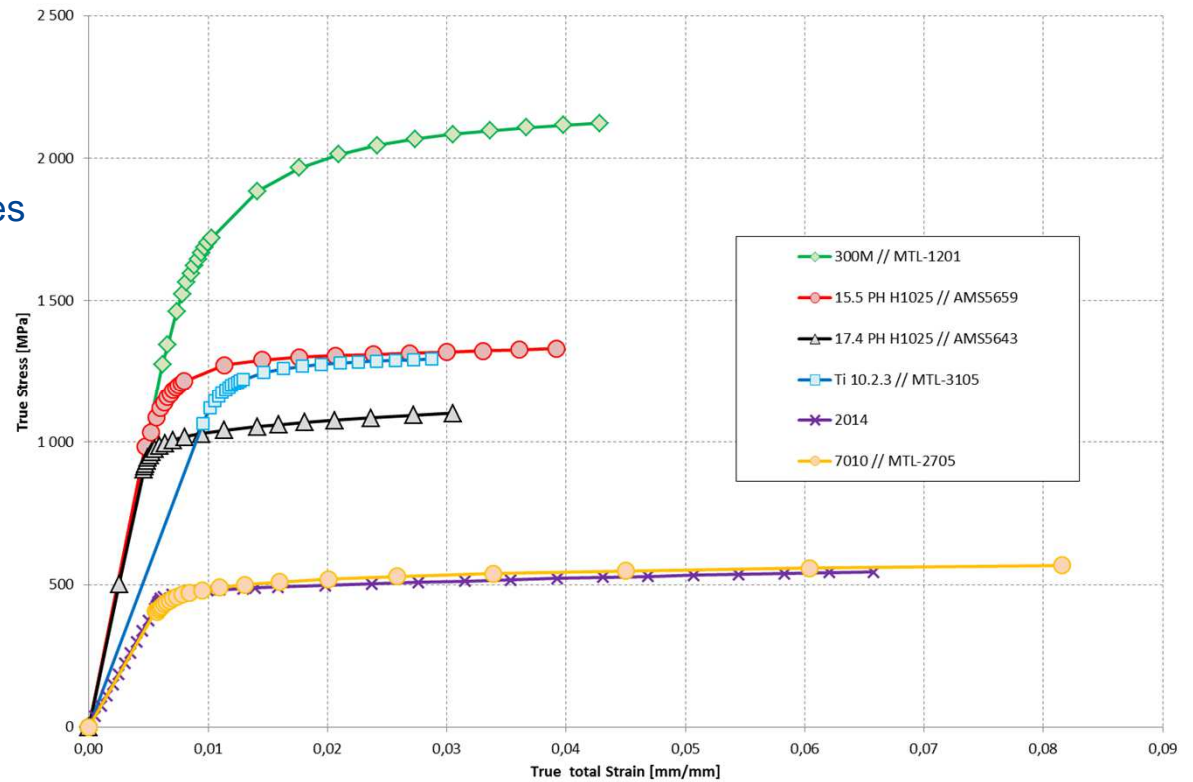
# MATERIALS

## High strength materials

### ➤ Tensile tests

- Typical

true stress-strain curves  
up to UTS



# MATERIALS

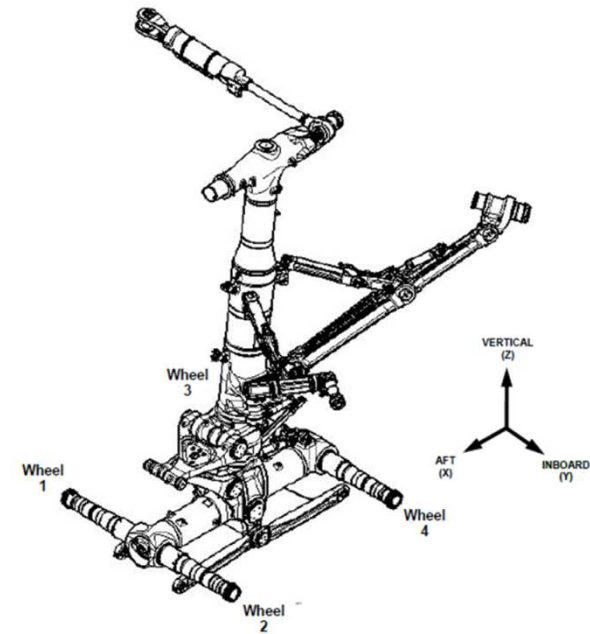
## High strength materials

- **High yield stresses**
  - Large elastic strains
- **Low Hardening**
  - $UTS / YS < 1.x$
- **Different forms of ductility depending on the materials**
  - For several materials: small strain to UTS  $e_u$
  - No correlation between  $e_u$  and Elongation or Reduction of Area
- **Toughness**
  - For several materials: small

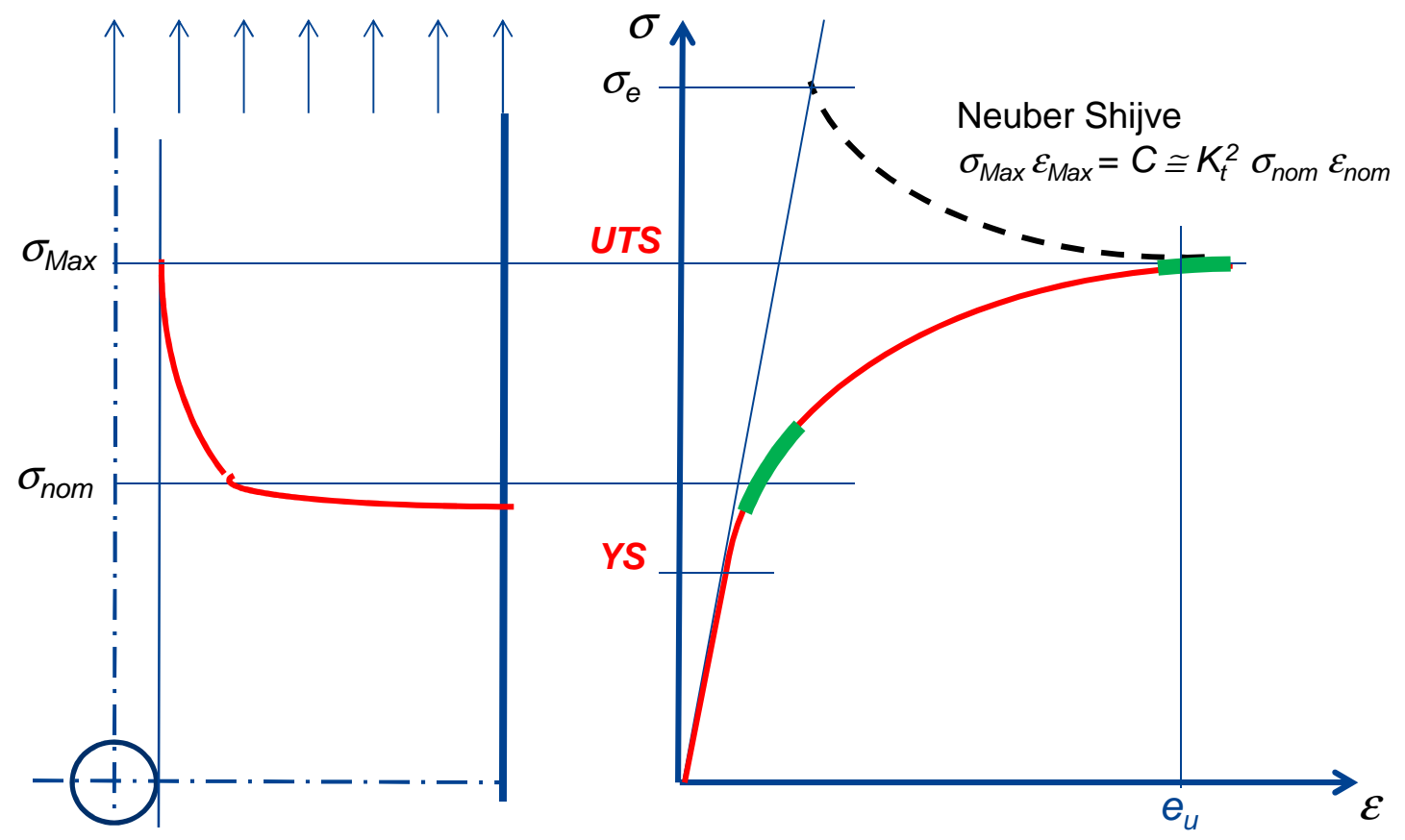
# STANDARD STATIC ANALYSIS

## General approach

- **Specifications**
  - ❑ Loading cases
  
- **Structural analysis**
  - ❑ Inputs:
    - Geometry
    - Boundary Conditions, loads
    - Material strain behaviour
  
- **Strength assessment**
  - ❑ Material rupture criterion
  - ❑ Geometrical instability



# STANDARD STATIC ANALYSIS – SIMPLE APPLICATION



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# 2

## APPLICATION TO STRUCTURES MADE OF HIGH STRENGTH STEELS



## Ductile rupture criterion for ML340 steel (from C. Defaisse PhD 2018)

### ML340 steel

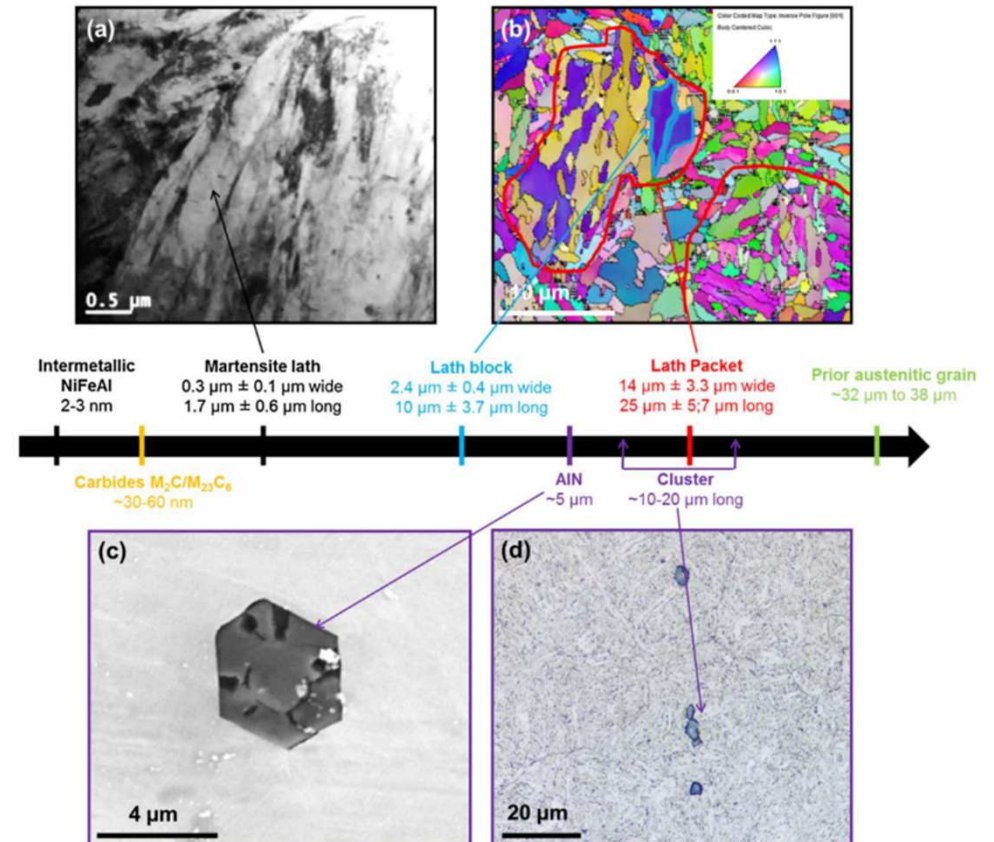
➤ **Complex microstructure**

- ❑ Typical composition (wt%)

Fe	C	Ni	Cr	Mo	Al	Co	V
Base	0.23	13.	3.3	1.5	1.5	5.8	0.25

( from Aubert & Duval 2015)

- ❑ Martensite + 1wt% austenite
- ❑ Carbides and intermetallics (<1µm)
- ❑ AlN inclusions



## Ductile rupture criterion for ML340 steel (from C. Defaisse PhD 2018)

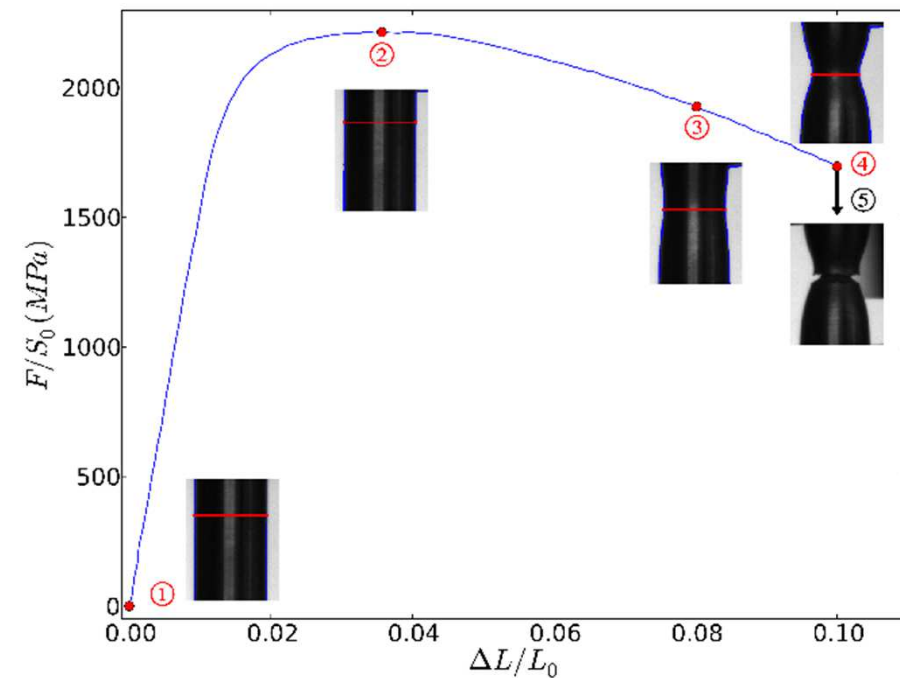
### ML340 steel

#### ➤ Very high strength

Material	UTS (MPa)	YS (Mpa)
Maraging 250	1758	1724
300M	1979	1682
ML340	2200	1900

#### ➤ Low ductility

- Strain to UTS  $e_u = 3\%$
- Red. of  $A = 40\%$

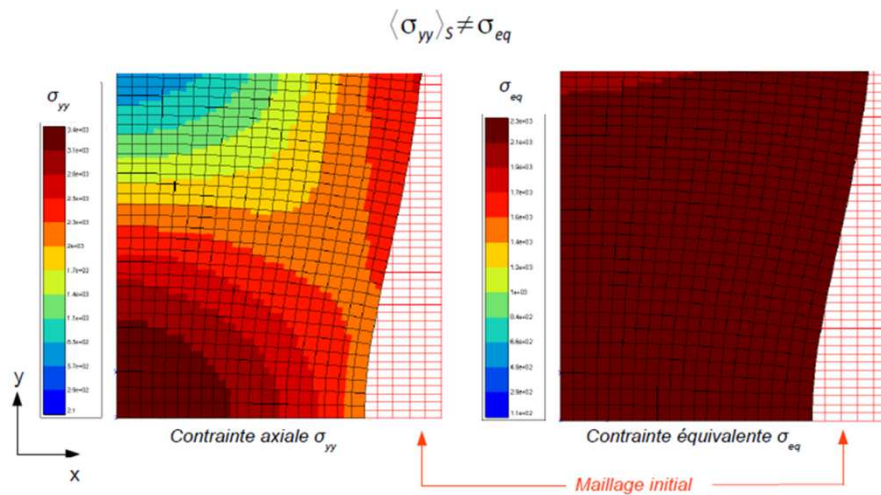




# Ductile rupture criterion for ML340 steel (from C. Defaisse PhD 2018)

## ML340 steel

- **Elastoplastic stress-strain behaviour**
  - von Mises criterion, isotropic hardening
  - Inverse identification using necking geometry

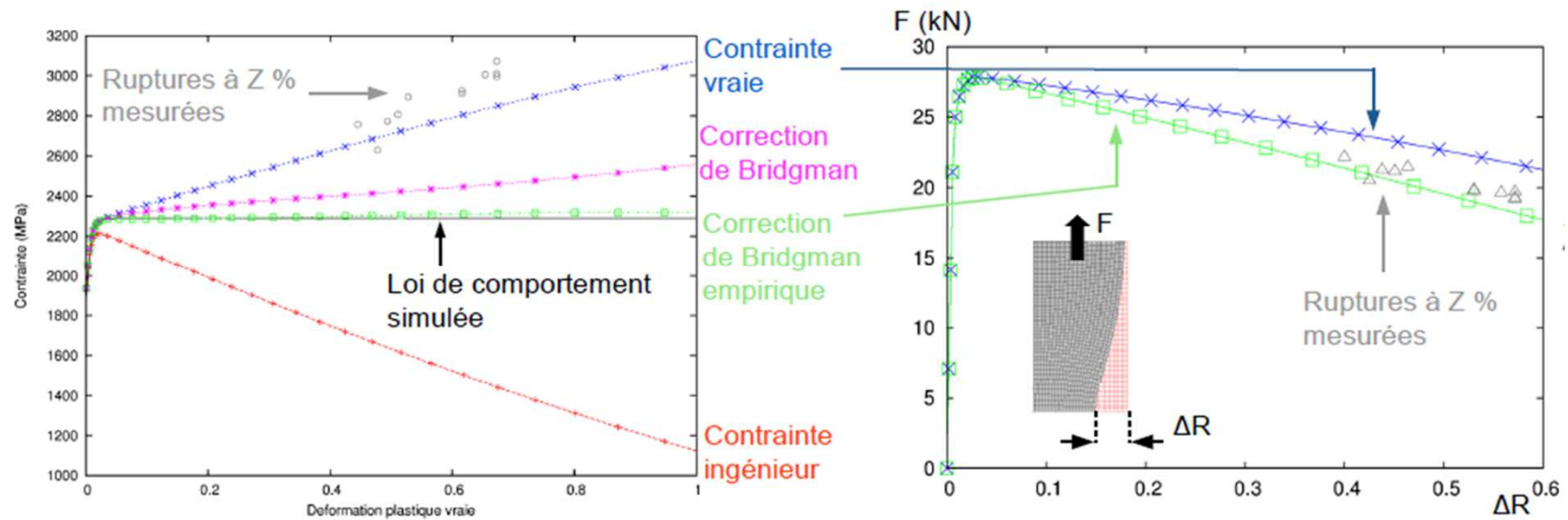


# Ductile rupture criterion for ML340 steel (from C. Defaisse PhD 2018)

## ML340 steel

### ➤ Elastoplastic stress-strain behaviour

- ❑ Inverse identification using necking geometry



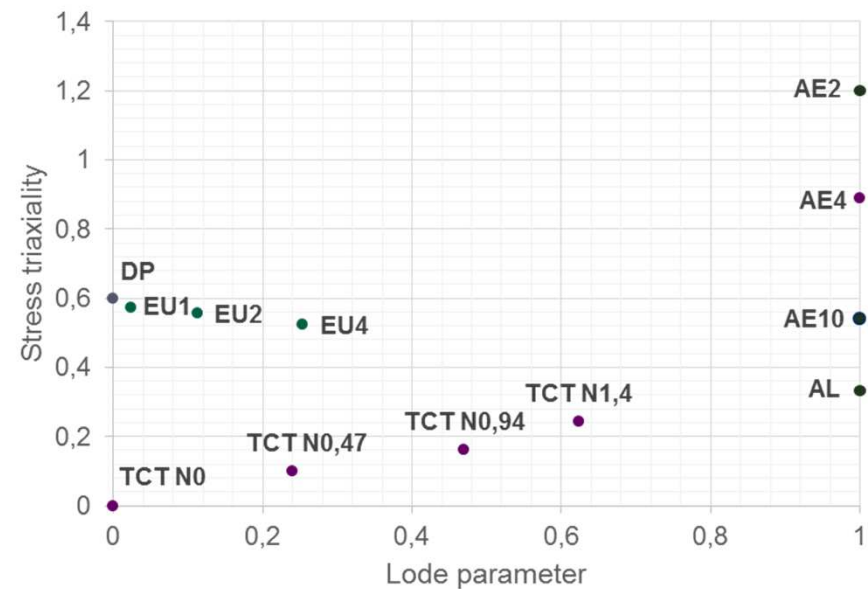
## Ductile rupture criterion for ML340 steel (from C. Defaisse PhD 2018)

### ML340 steel

#### ➤ Ductile damage model

- ❑ Experimental campaign
  - TCT: tension-compression-torsion
  - DP: plane deformation
  - AL: standard tension
  - AE: cylindrical notched tension
  - EU: plane notched tension
- ❑ Model proposed

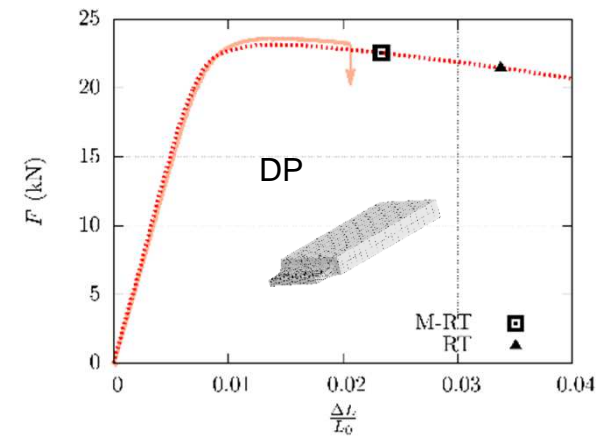
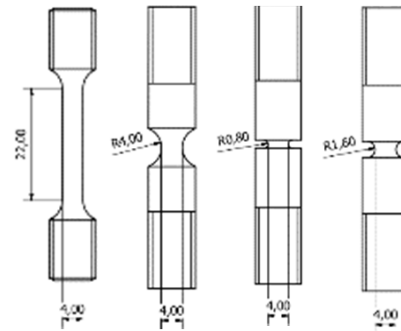
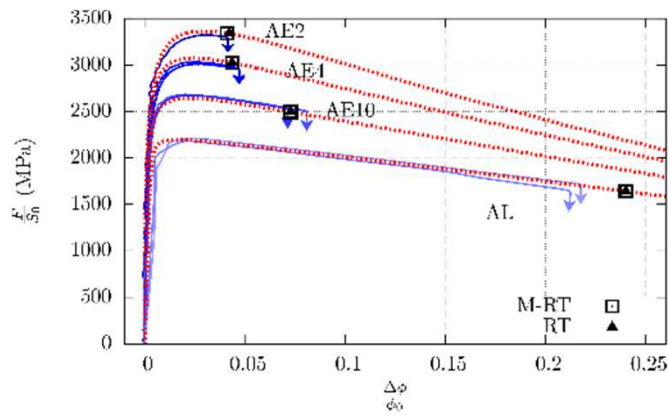
$$D = \int_0^p \underbrace{(ae^{-bT})}_{\text{Rice \& Tracey}} + \underbrace{c(1 - |L|^d)}_{\text{Lode correction}} d\bar{\epsilon}$$



# Ductile rupture criterion for ML340 steel (from C. Defaisse PhD 2018)

## ML340 steel

- Numerical simulation of all tests and inverse identification of damage model
  - Cylindrical notched tension (AE) and Plane tension (DP)

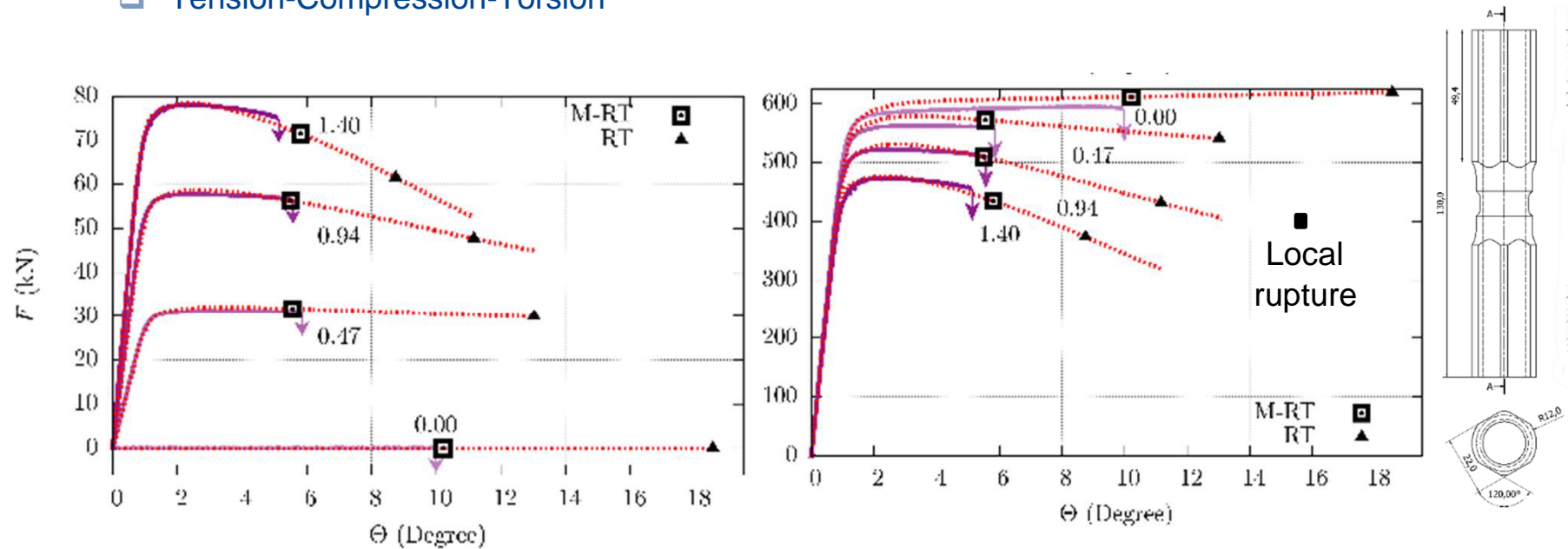


### ■ Local rupture

# Ductile rupture criterion for ML340 steel (from C. Defaisse PhD 2018)

## ML340 steel

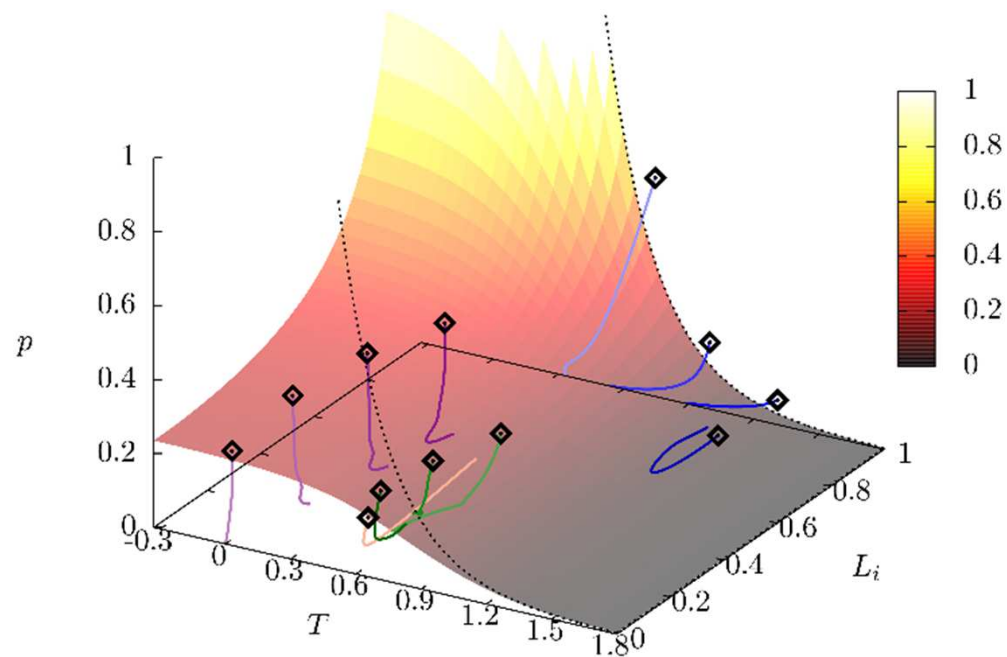
- Numerical simulation of all tests and inverse identification of damage model
  - Tension-Compression-Torsion



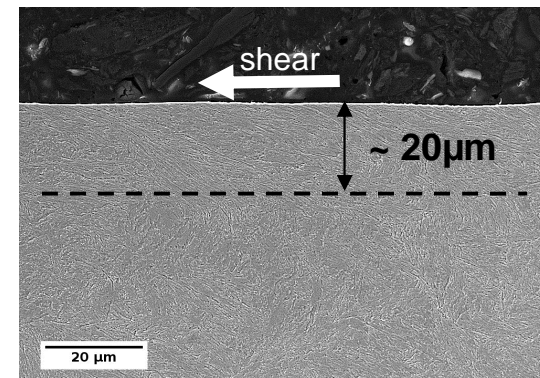
# Ductile rupture criterion for ML340 steel (from C. Defaisse PhD 2018)

## ML340 steel

### ➤ Identified damage model



□ But... at which scale?



Fracture surface of a torsion coupon

# TORSION MODULUS OF RUPTURE – $TM_{oR}$ ALLOWABLE

“Classical” approach to size tubes submitted to torsion loading

- **Typical curve (from MMPDS)**
  - ❑ Different failure mechanisms
    - Material failure
      - Shear stress criterion
      - “Identified” as  $UTS / \sqrt{3}$
      - Through the whole thickness of the tube
    - Buckling analysis
      - Deformation theory
      - Minimization of the total potential energy
      - No local buckling

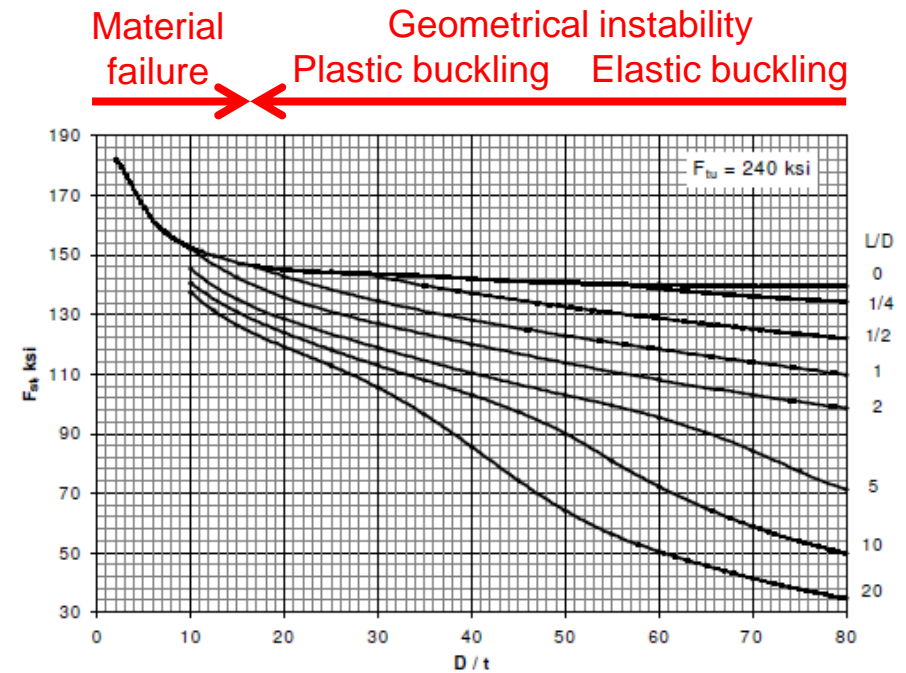


Figure 2.8.3.2(i). Torsional modulus of rupture - plain carbon steels,  $F_{tR} = 240$  ksi.



# TMoR ALLOWABLE – NUMERICAL PREDICTIONS (from M. Al Kotob PhD 2019)

## Competition between local and global instabilities

### ➤ Approaches

- Material failure by strain localization
  - Loss of ellipticity : Rice criterion
- Hill loss of stability analysis
- Finite strain formalism
- von Mises criterion
- Isotropic hardening

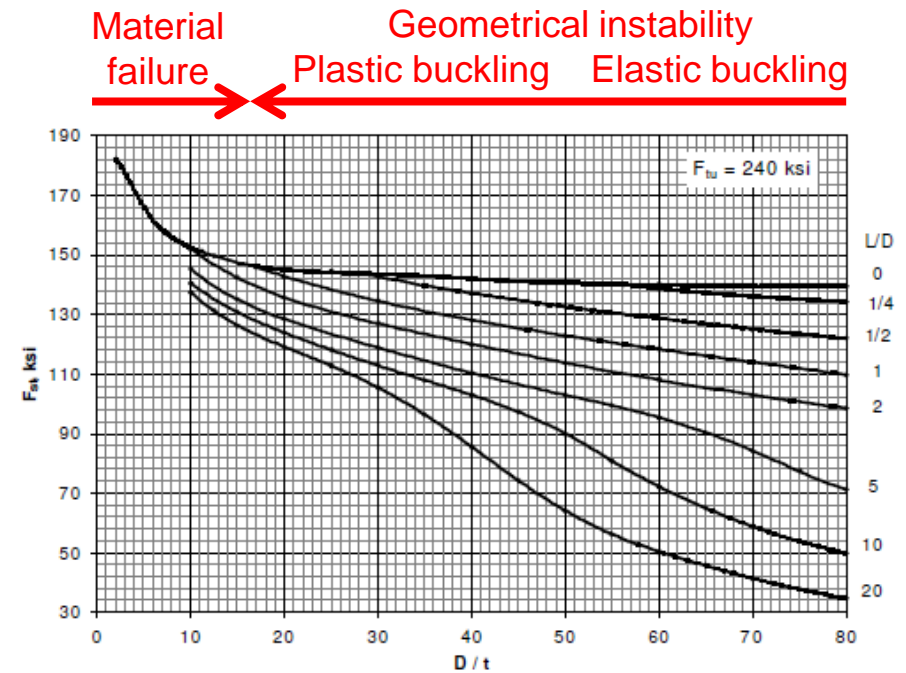
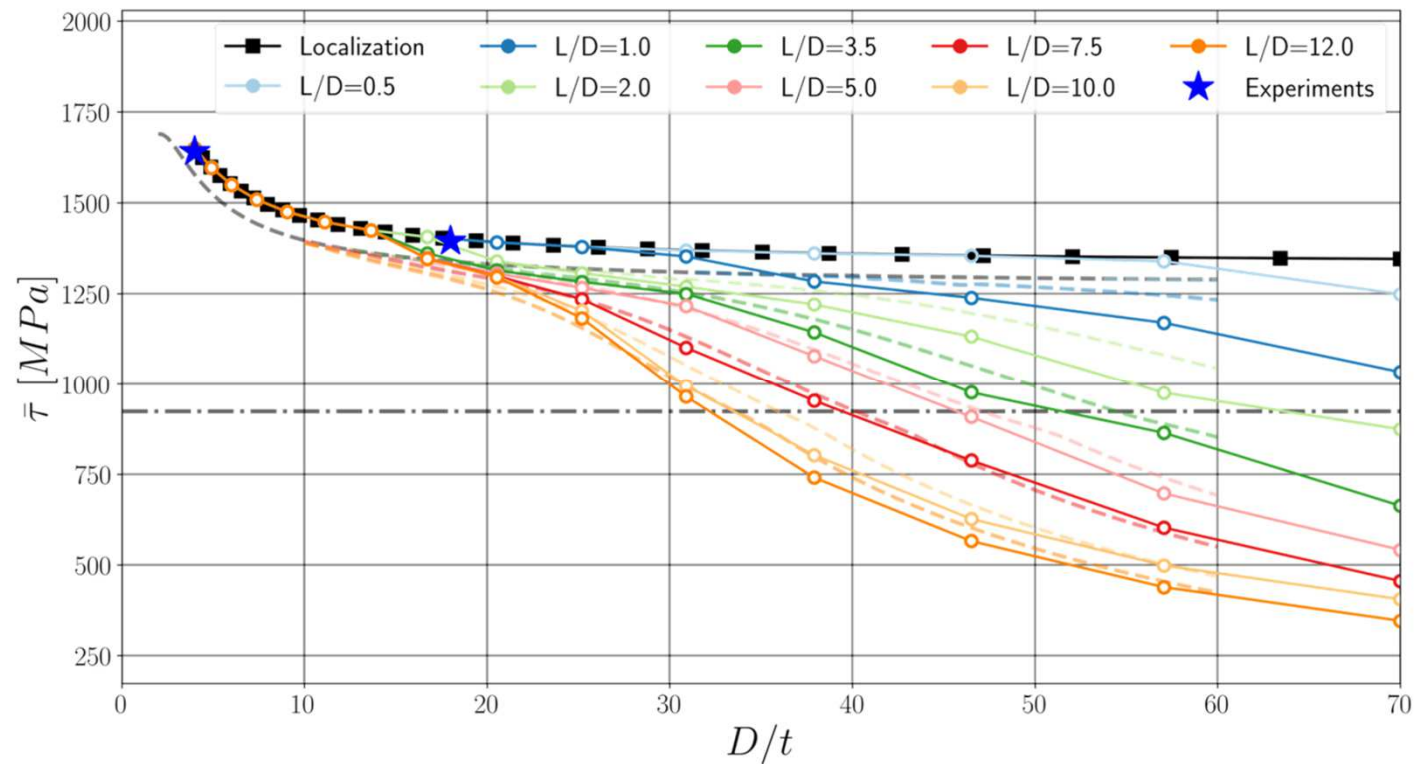


Figure 2.8.3.2(i). Torsional modulus of rupture - plain carbon steels,  $F_{tr} = 240$  ksi.



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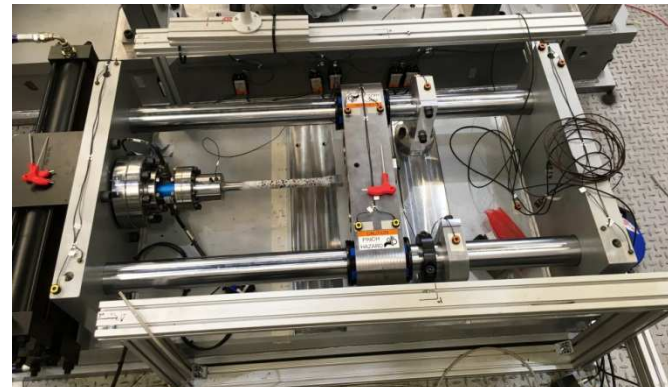
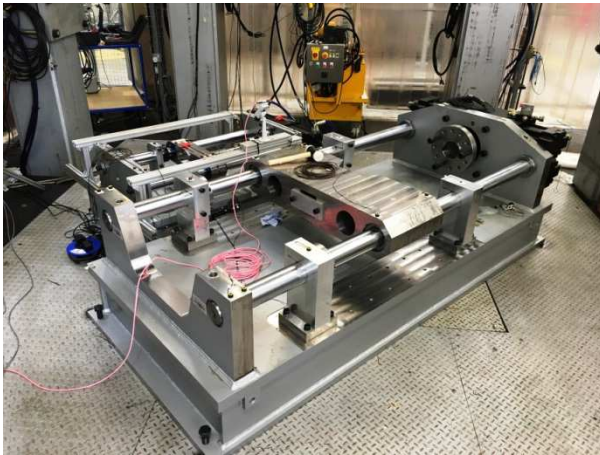
## Comparison with test results for ML340 steel



# TMoR ALLOWABLE – EXPERIMENTAL RESULTS

Test rigs at TestEng lab Safran Landing Systems Gloucester

- **First experimental campaign on 300M martensitic steel**
  - ❑ 3 specimens for each size ( $L/D$ ,  $D/t$ )
  - ❑ Torque, rotation, strain gauges, DIC on specimens, optical markers on test rig

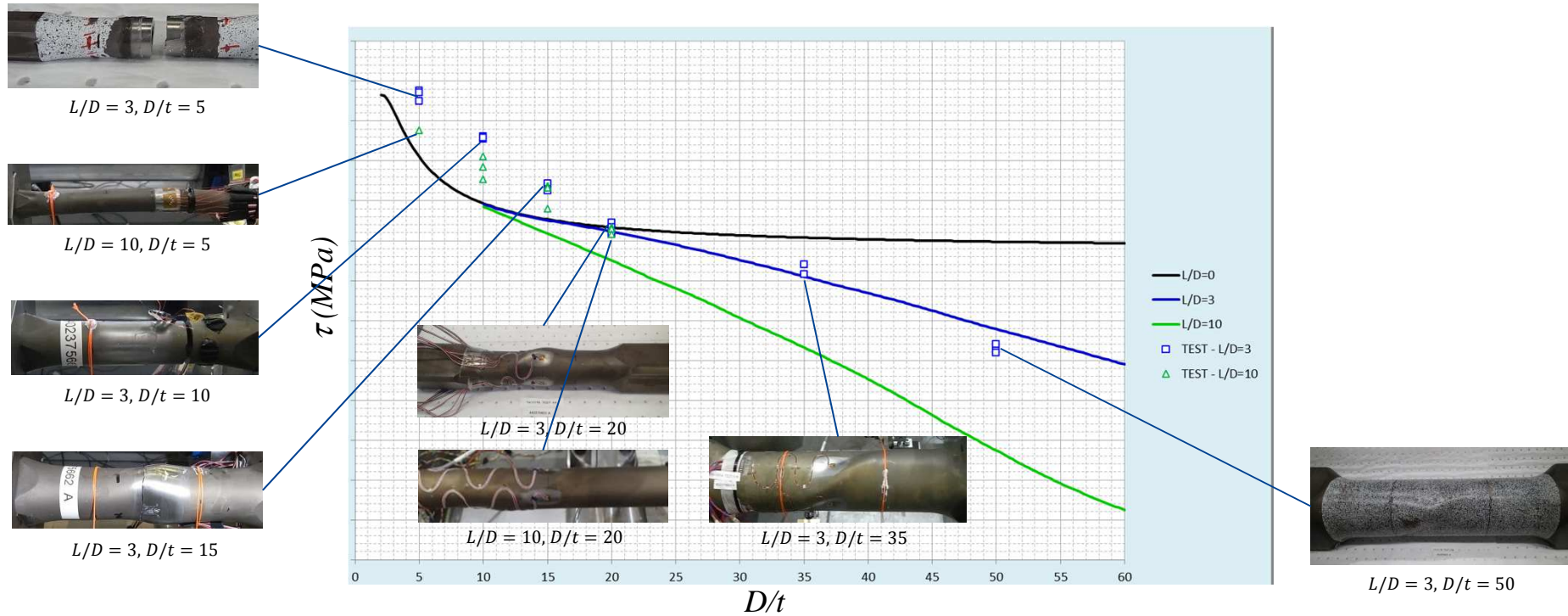


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# TMoR ALLOWABLE – EXPERIMENTAL RESULTS

Comparison of test results for 300M steel with Lee & Ades predictions (from N. Antoni)



# TMoR ALLOWABLE – FINITE ELEMENT SIMULATIONS

## Comparison with test results for 300M steel for stable structures

### ➤ Constitutive model

- von Mises criterion
- Isotropic hardening
- Identification from coupons cut from the same material

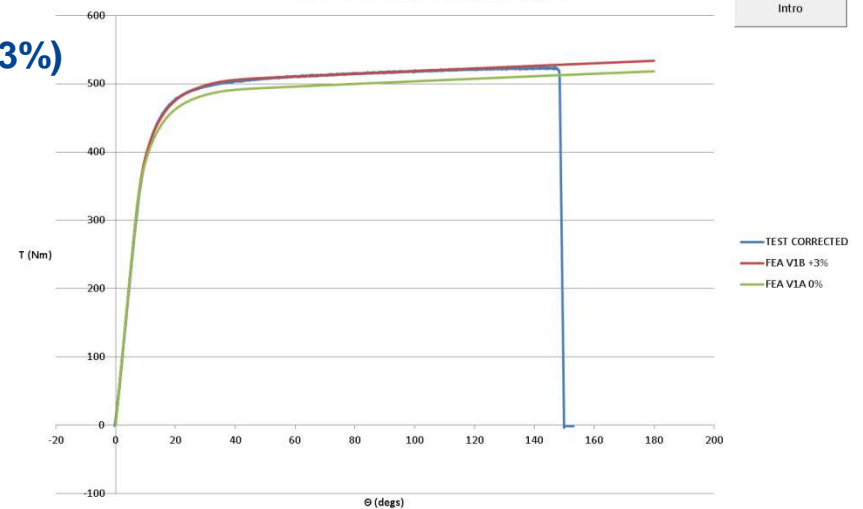
### ➤ Systematic error on predicted global response (~3%)

- Constitutive model to be revisited
  - Kinematic hardening to be added?
  - Yield criterion to be refined  
Hosford, Hill, 3<sup>rd</sup> invariant, ...?
- Further material tests in progress



$$L/D = 3, D/t = 5$$

Test vs FEA (2) - Torque Output



# TMoR ALLOWABLE – FINITE ELEMENT SIMULATIONS

## Comparison with test results for 300M steel for geometrical instabilities

### ➤ Geometry of the specimens

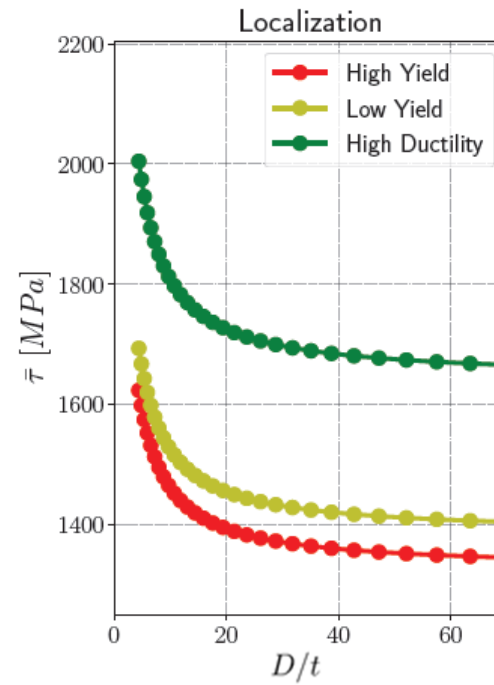
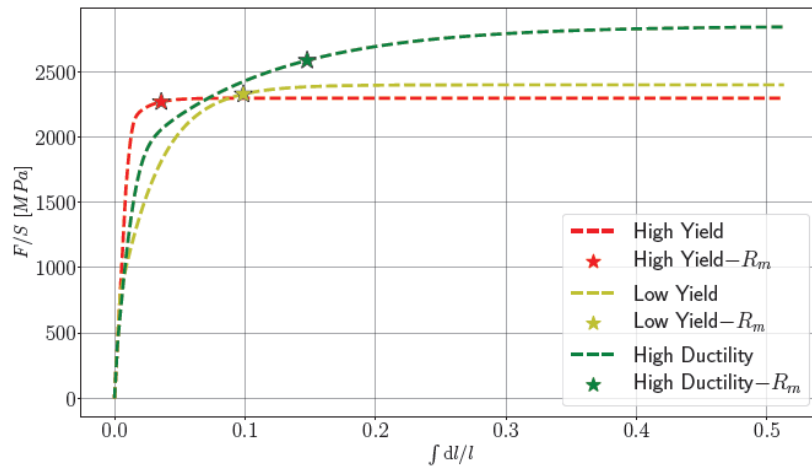
- Small geometrical imperfections measured on each specimens
  - In general different for the 3 specimens of the same geometry
- No obvious relationship between instability onset and instability mode

### ➤ Sensitivity of the predicted global response

- No obvious relationship between geometrical imperfections and the predicted instability mode
- Further investigations on the influence of Boundary Conditions in progress

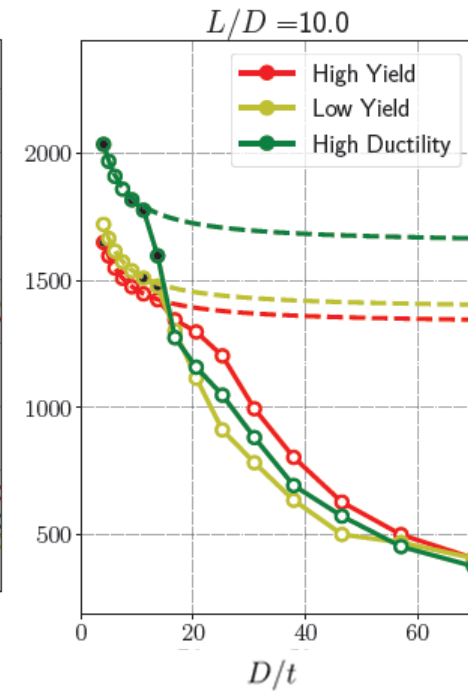
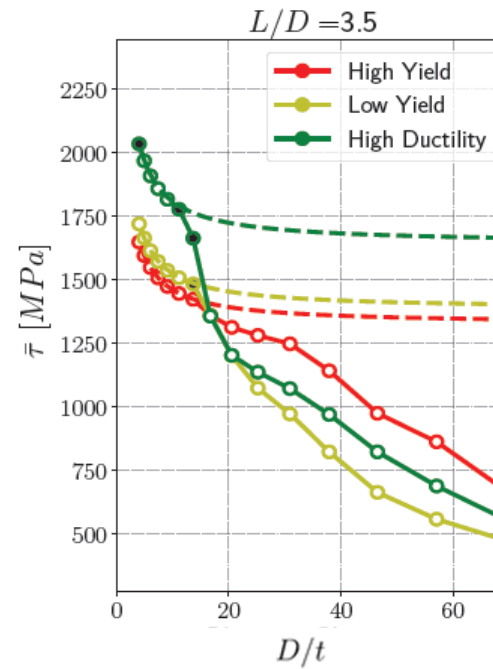
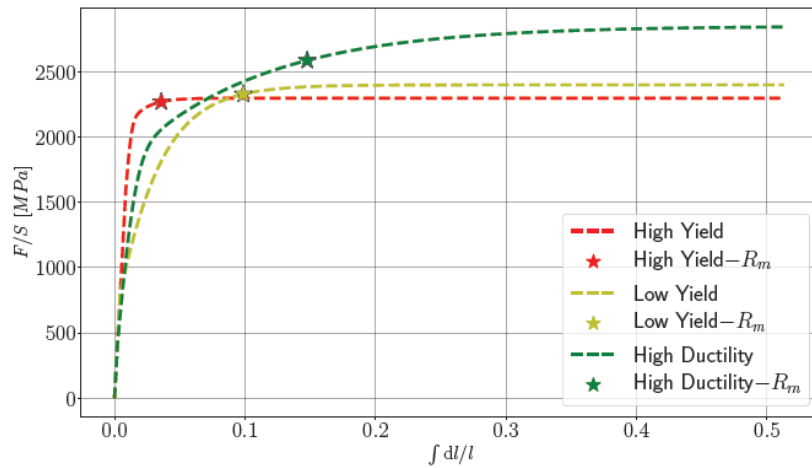
# TMoR ALLOWABLE – NUMERICAL PREDICTIONS (from M. Al Kotob PhD 2019)

## Sensitivity to material hardening – Prediction of strain localization



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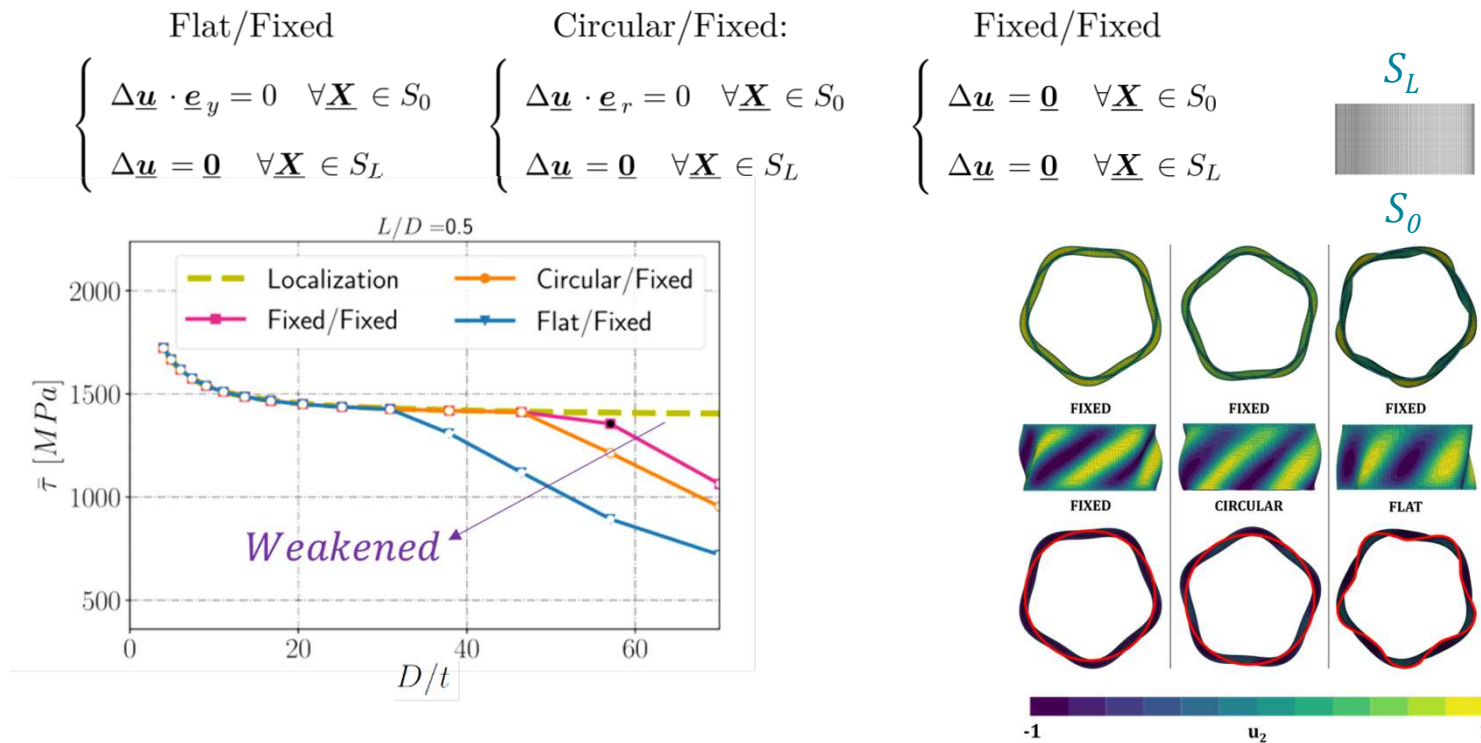
## Sensitivity to material hardening – Prediction of geometrical instabilities





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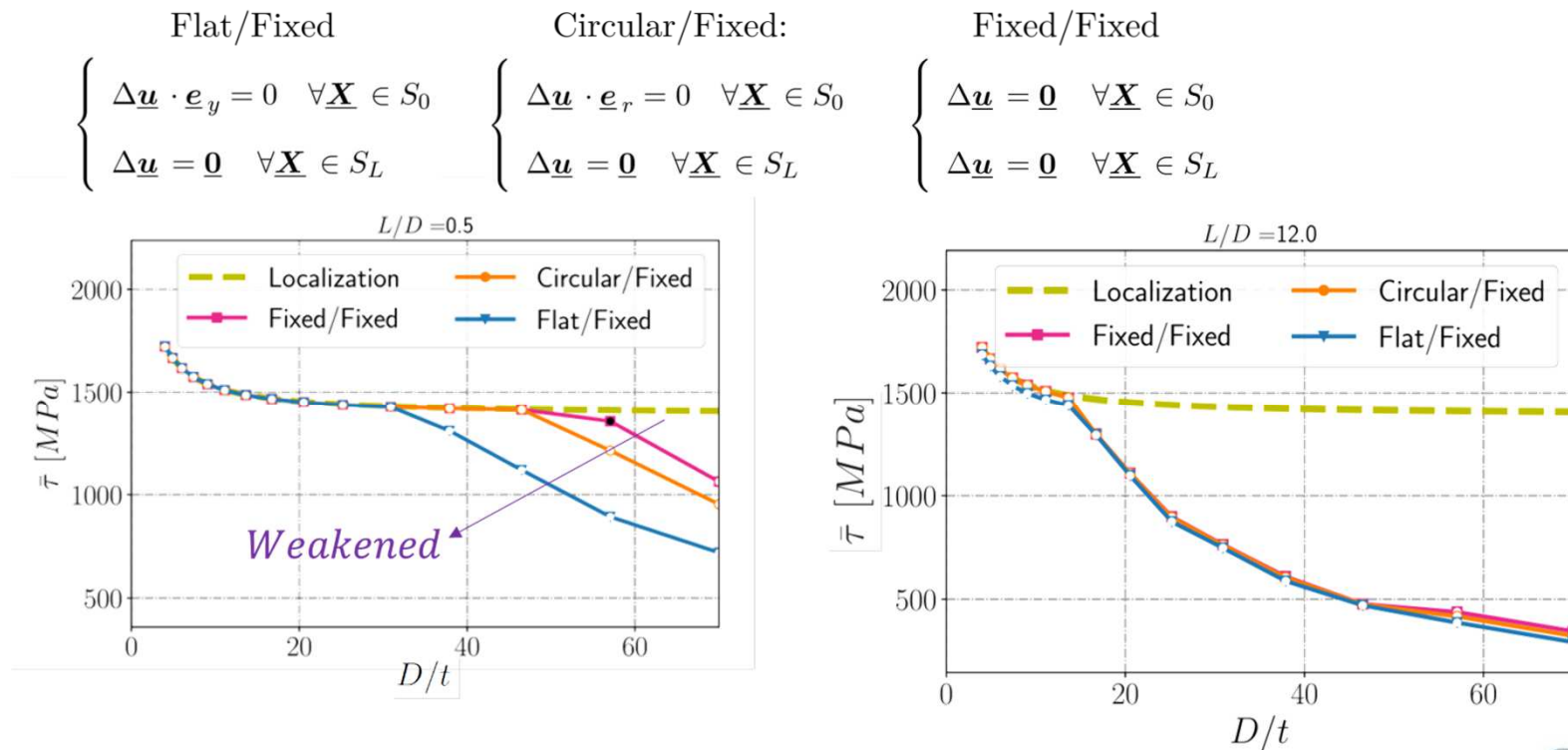
## Sensitivity to Boundary Conditions – Prediction of geometrical instabilities ( $L/D = 0.5$ )





# TMoR ALLOWABLE – NUMERICAL PREDICTIONS (from M. Al Kotob PhD 2019)

## Sensitivity to Boundary Conditions – Prediction of geometrical instabilities ( $L/D = 12$ )



# 3

## OTHER APPLICATIONS



# FUSE PINS

## Design and analysis

- Designed to provide a safe and controlled separation of the landing gear from the aircraft
  - ❑ Special care to fatigue life assessment



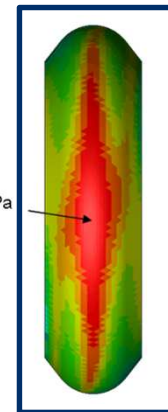
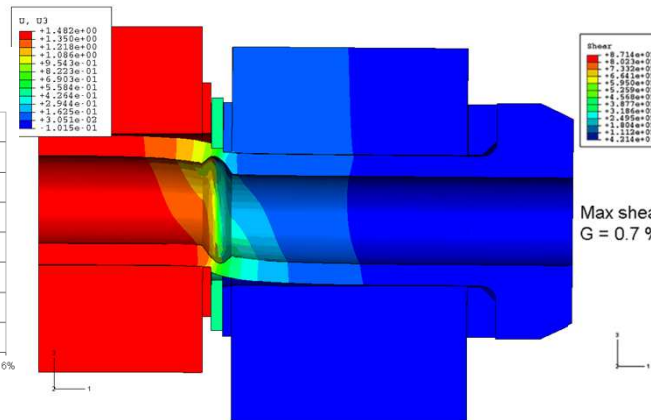
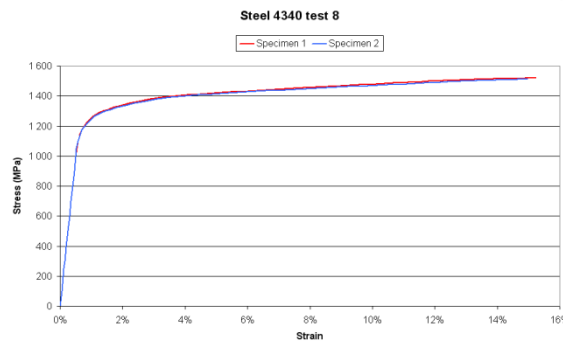
	Slick fuse pins	Groove fuse pins
Design type		
Static strength	<p><b>Cons:</b> Initiation &lt;&lt; Total rupture                      Shear Fracture criterion:  <math>\frac{P}{2S} = F_{su} = K_r F_{tu}</math>                      Requires low <math>D/t</math> ratio</p>	<p><b>Pros:</b> Initiation <math>\approx</math> Total rupture                      Shear Fracture criterion:  <math>\frac{P}{2S} = \frac{1}{\sqrt{3}} F_{tu}</math></p>
Fatigue	<p><b>Pros:</b> higher fatigue strength</p>	<p><b>Cons:</b> high stress concentration area unfavourable against fatigue and stress corrosion</p>

# FUSE PINS

## Design and analysis

### ➤ Analysis supported by test

- ❑ Rupture at a load slightly larger than the ultimate load
- ❑ Shear design curves derived from dedicated tests
  - Fine tuning of the geometry to meet strength requirement and material variability



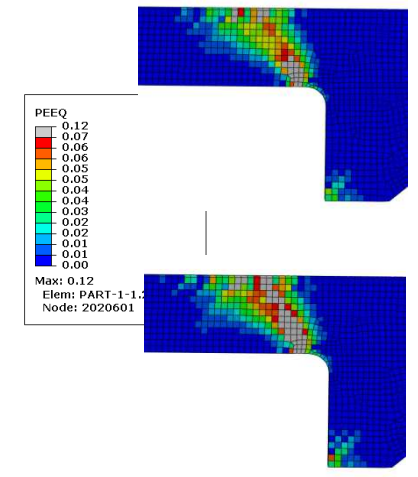
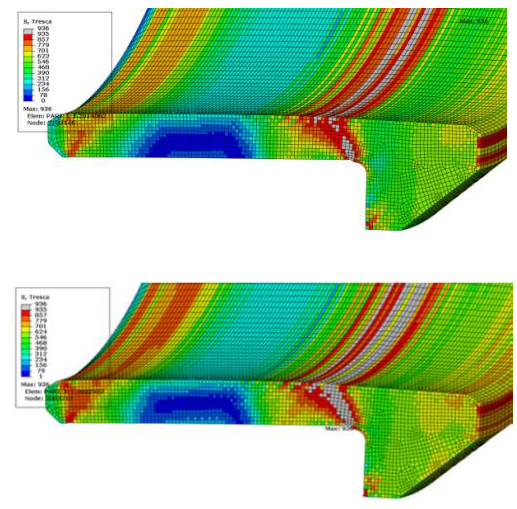
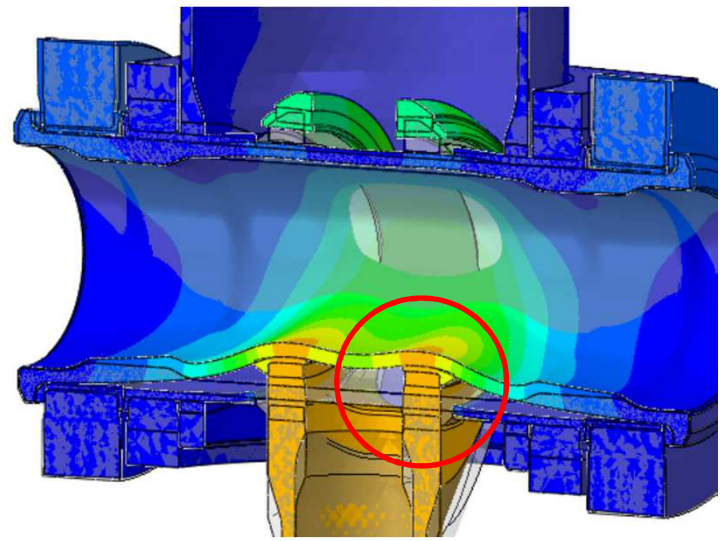
# BUSH SUBJECTED TO EXTREME LOAD

## Design and analysis

- Prediction of the load leading to the fracture of the bronze bush

Tresca stress

Cumulated plastic strain



# 4

## CONCLUSIONS



# CONCLUSIONS

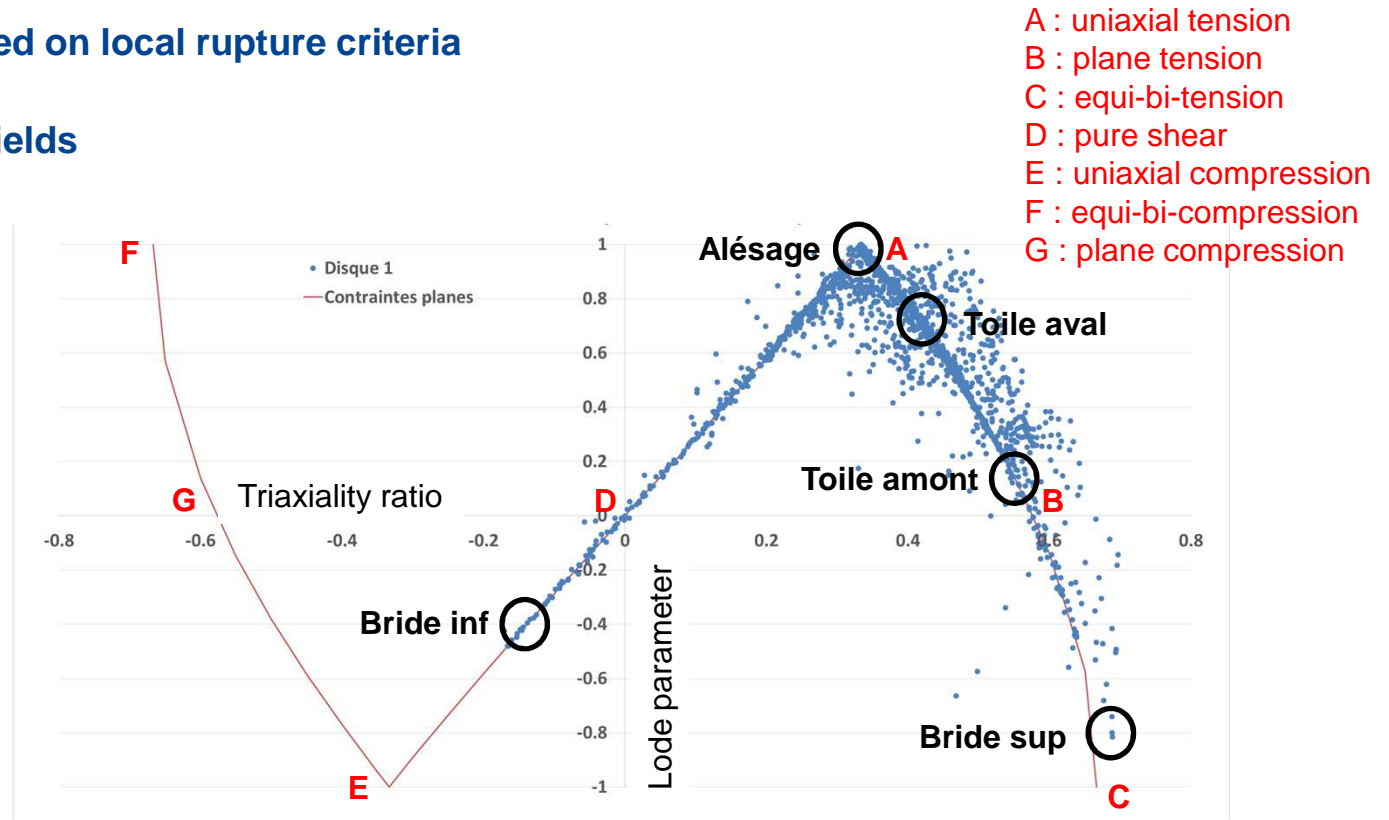
## More efforts are needed on local rupture criteria

- **Rupture initiation in contact zones**
  - ❑ Lugs, assemblies
  
- **Validation for different materials and loadings**
  - ❑ Effect of the stress-strain behaviour
  - ❑ Effect of stress 3<sup>rd</sup> invariant
    - Shear + positive or negative stress 1<sup>st</sup> invariant
  - ❑ Complex multiaxial non-proportional loadings
    - Complex tests to be designed

# CONCLUSIONS

More efforts are needed on local rupture criteria

➤ Complex stress fields

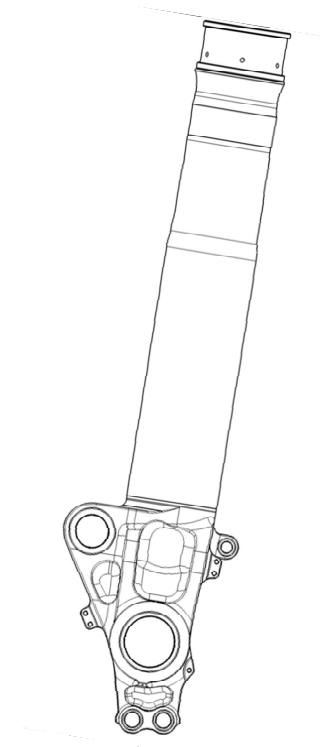
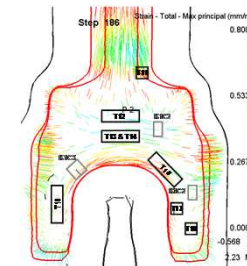
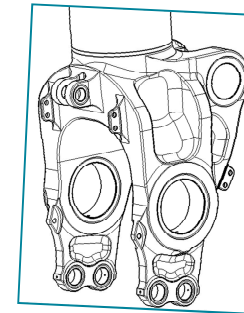




# CONCLUSIONS

More efforts are needed on local rupture criteria

- How to cope with the variability of material properties?
    - Anisotropy
    - Isotropic and kinematic hardening
    - Behaviour post-localization,...
- via numerical sensitivity analyses with validated tools?



# CONCLUSIONS

## More efforts are needed on global instability analyses

### ➤ How to cope with the variability of key factors?

- Variability of material behaviour post-localization
- Variability of components geometry
- Uncertainty on Boundary Conditions

→ via numerical sensitivity analyses with validated tools?

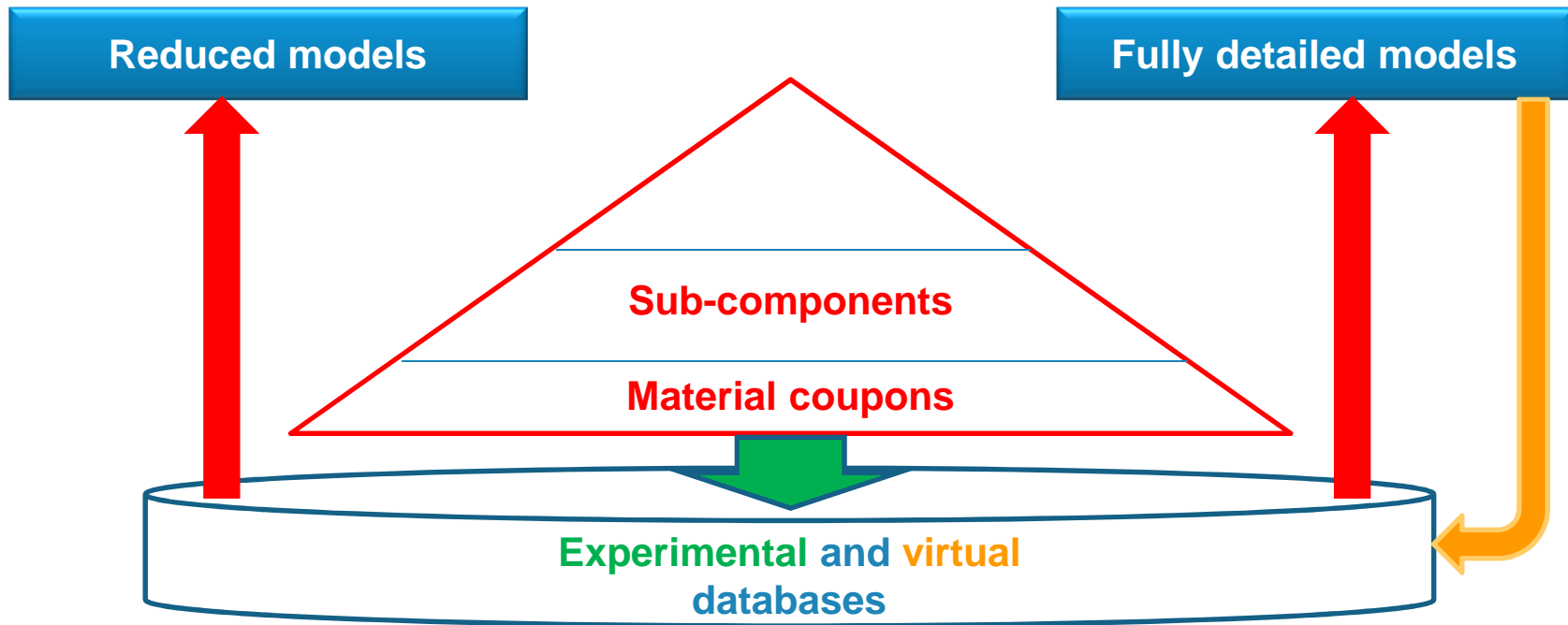
# CONCLUSIONS

## Design allowables

- **Requirements for certification by analysis supported by tests**
  - ❑ Material allowables: material standard properties
  - ❑ Design allowables: strength of particular features (lugs, tubes,...)
- **Tests to support analysis methods**
  - ❑ Good knowledge of mean values and scatter of material and design allowables
    - To validate analysis methods at different scales
    - To substitute some tests at sub-component level by numerical simulations
    - To derive Reserve Factors taking account of material variability
- **Mature and documented analysis methods supported by tests at sub-components scale**
  - ❑ To decrease the number and the complexity of full-scale tests

# CONCLUSIONS

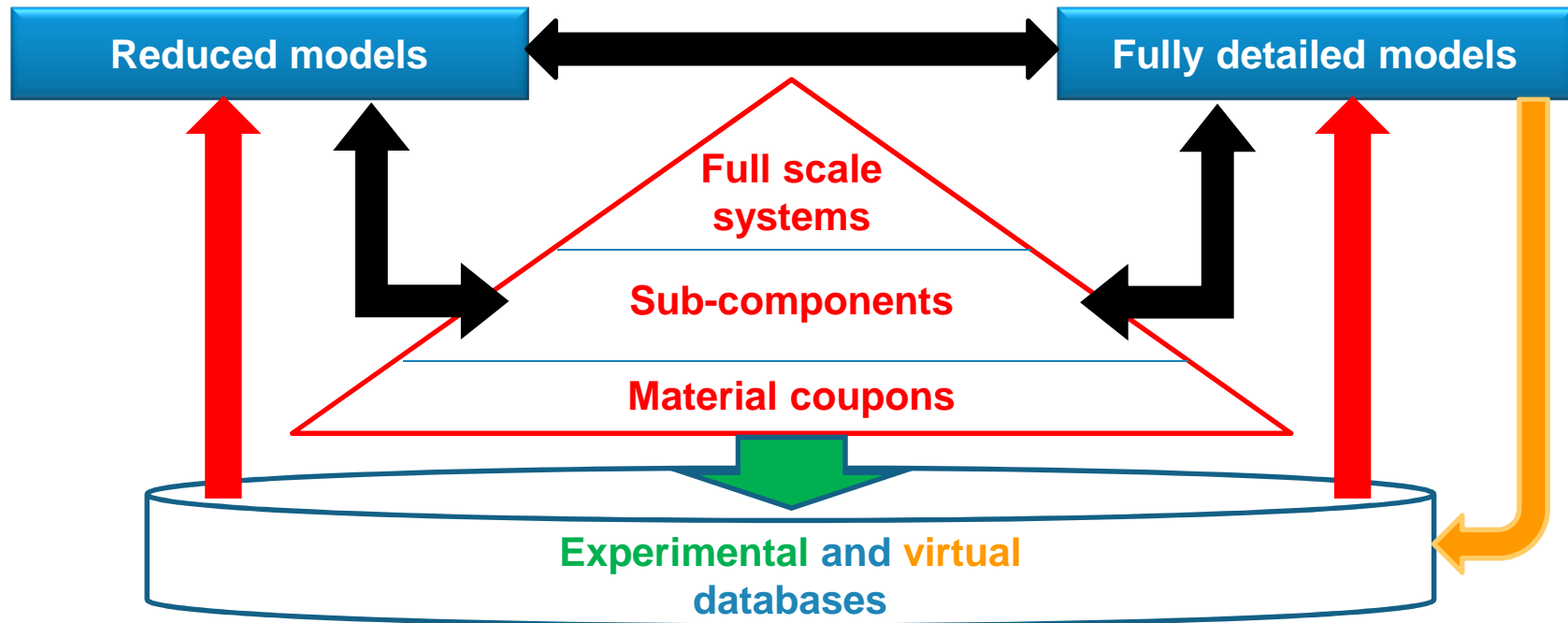
## Testing pyramid



The maturity of both sets of modelling tools is strongly dependent on **reliable databases on the entire field of application**

# CONCLUSIONS

## Testing pyramid



Validation of both sets of modelling tools via a refined exploitation of **highly instrumented tests mainly at sub-component scale**

# CONCLUSIONS

## Testing pyramid

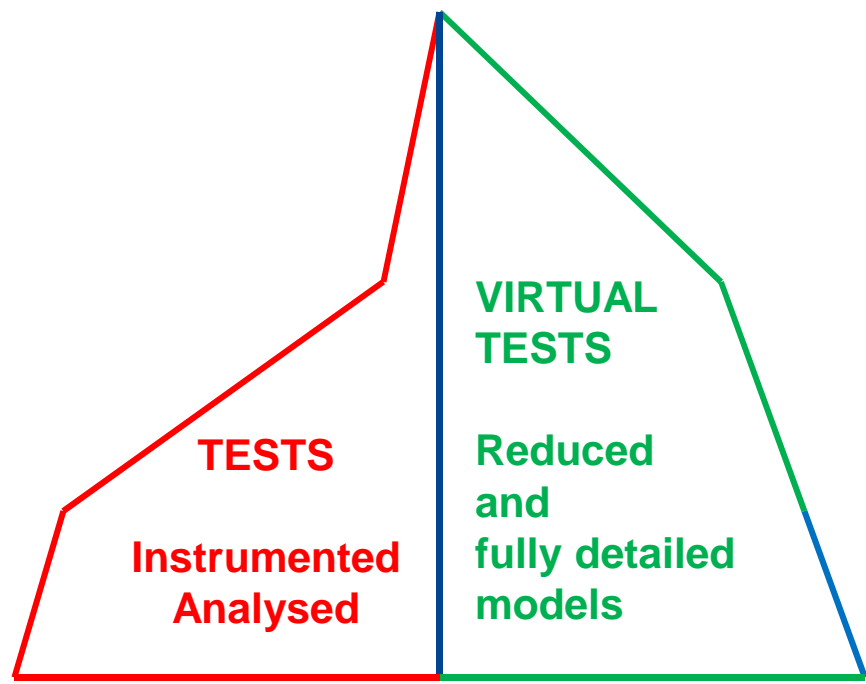
Full scale systems

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Sub-components

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Material coupons



C2 - Restricted



**POWERED  
BY TRUST**