

SC-Consultants

Replace **trial & error** with numerical simulation
to save **time & money**





Simulation du chauffage dans un four

Mecamat – Aussois 2024

Sacha El Aouad, Giulia Lissoni, Laurent Ratte et [Lucas Sardo](#)





WHO ARE WE?

SCIENCES COMPUTERS CONSULTANTS edits and markets numerical simulation software dedicated to industrial processes.

In a context where raw materials and energies are becoming more and more expensive, halving conventional methods «trials/errors» is a considerable argument.

OUR FOCUS AREAS



ABOUT US

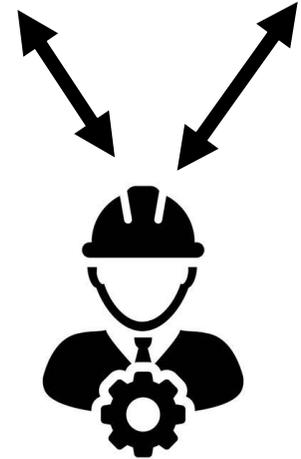
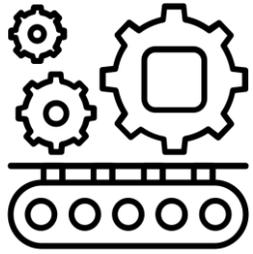
02



INDUSTRY
RESEARCH

Materials transformation processes

R&D



Factories Technical Support



WHAT WE DO ?



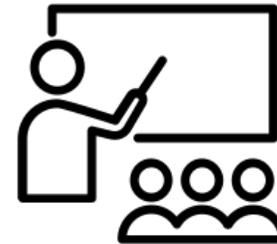
SOFTWARE LICENSE



CONSULTING



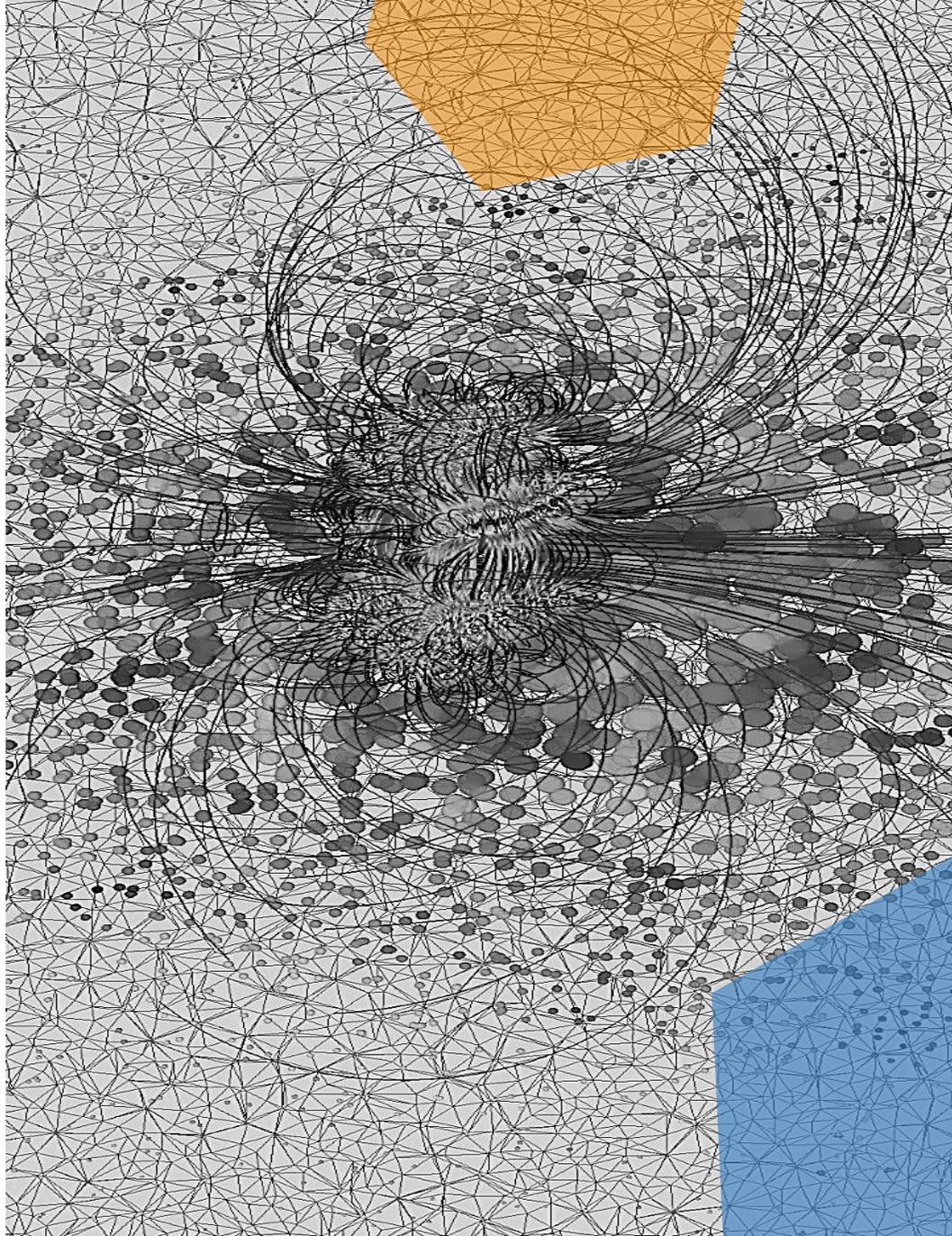
R&D Support



TRAINING



OUR SOFTWARE



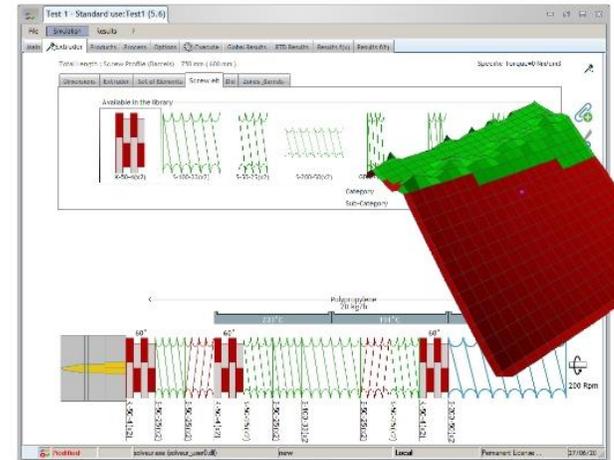


Twin Screw



OUR SOFTWARE 01

Opening the TSE black box



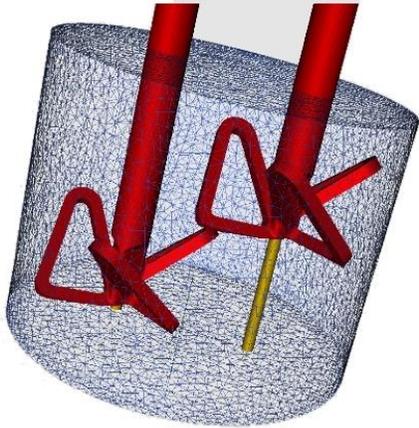


Mixing & Extrusion

OUR SOFTWARE 02



Master mixing with precision and control



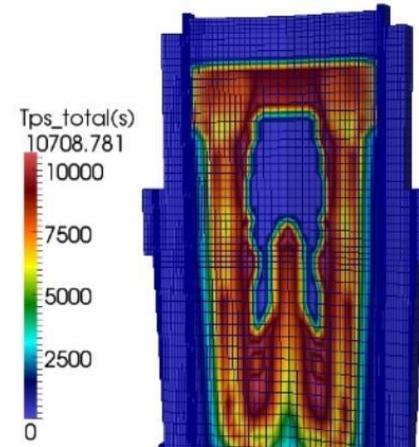


Ingot Casting



OUR SOFTWARE 03

Optimize casting processes by accurately predicting segregation.





Heating
&
Cooling

OUR SOFTWARE 04

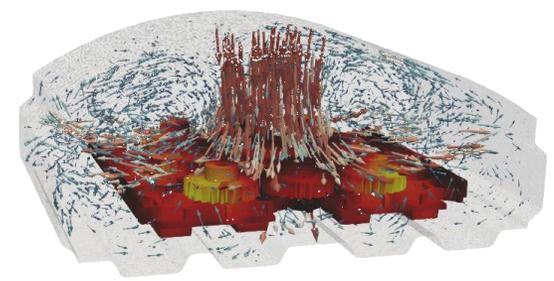
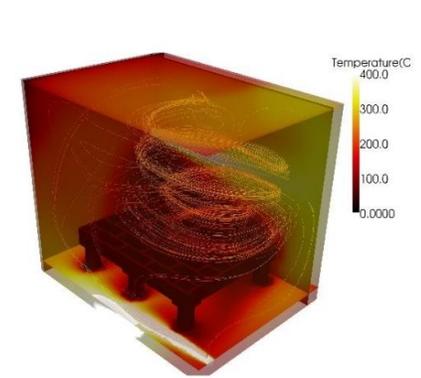
[qobeo]
Thermal Optimisation Solution

S2S

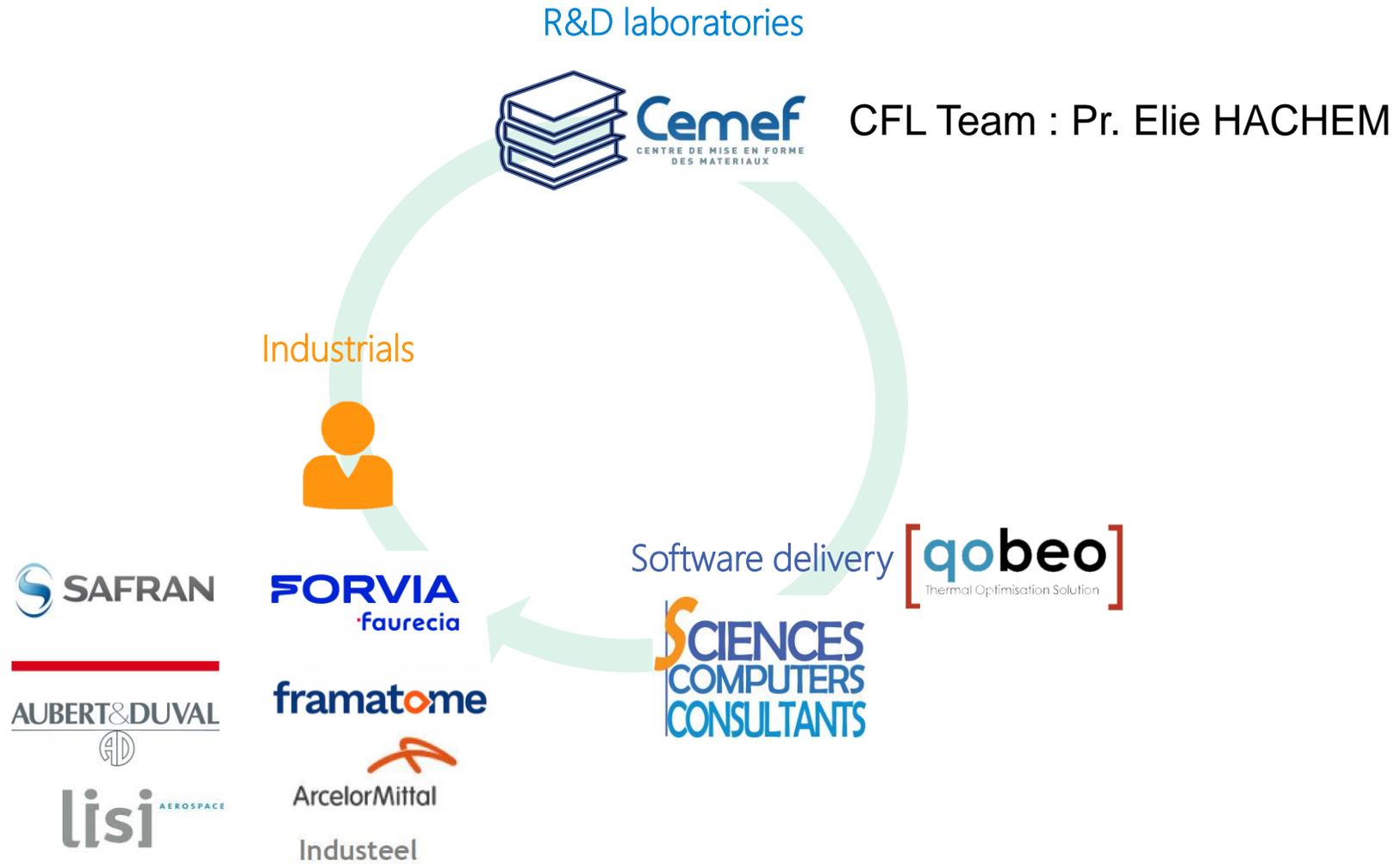
AIR

BUB

Optimize furnace efficiency: Simulate. Save. Succeed.

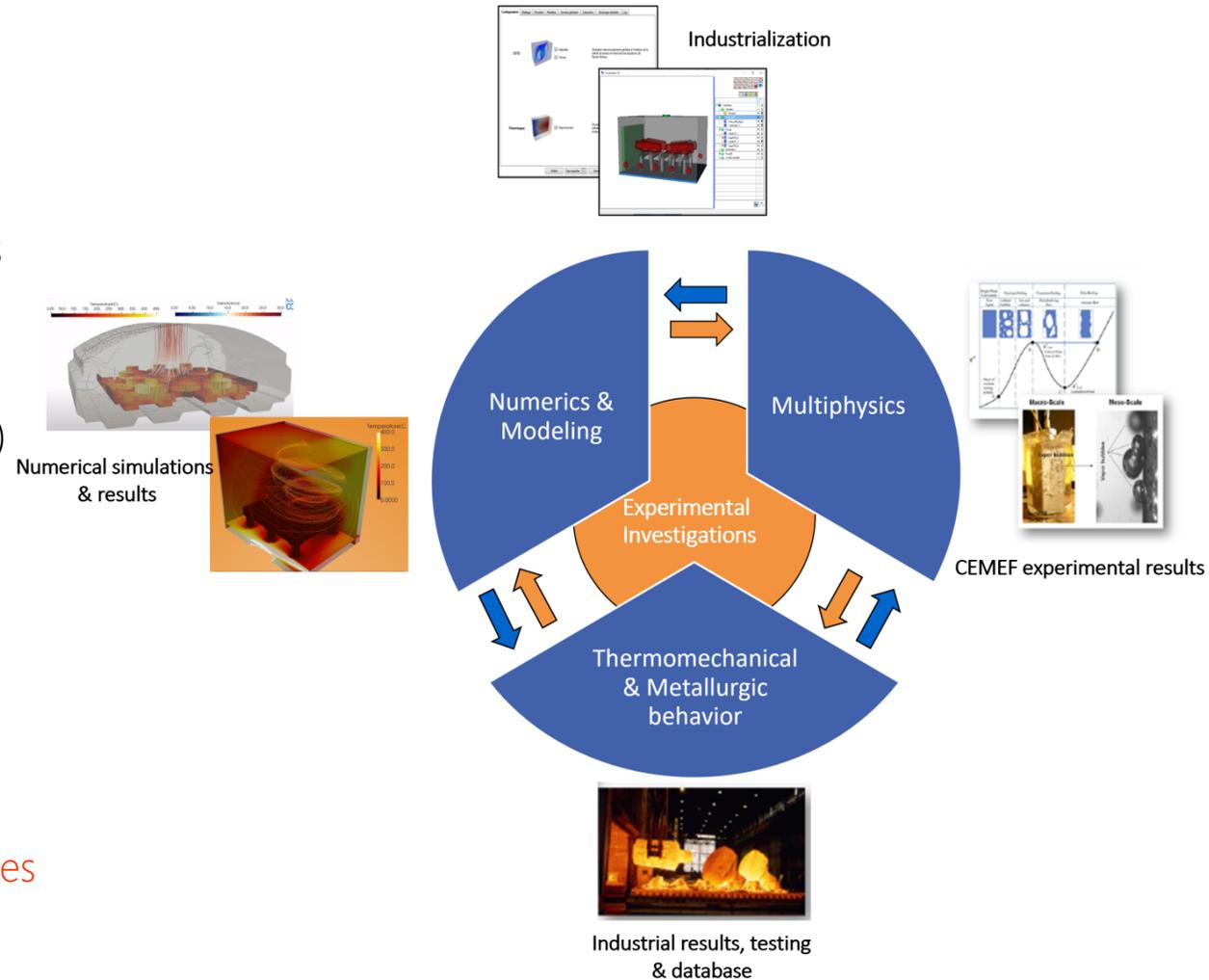


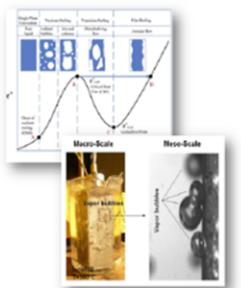
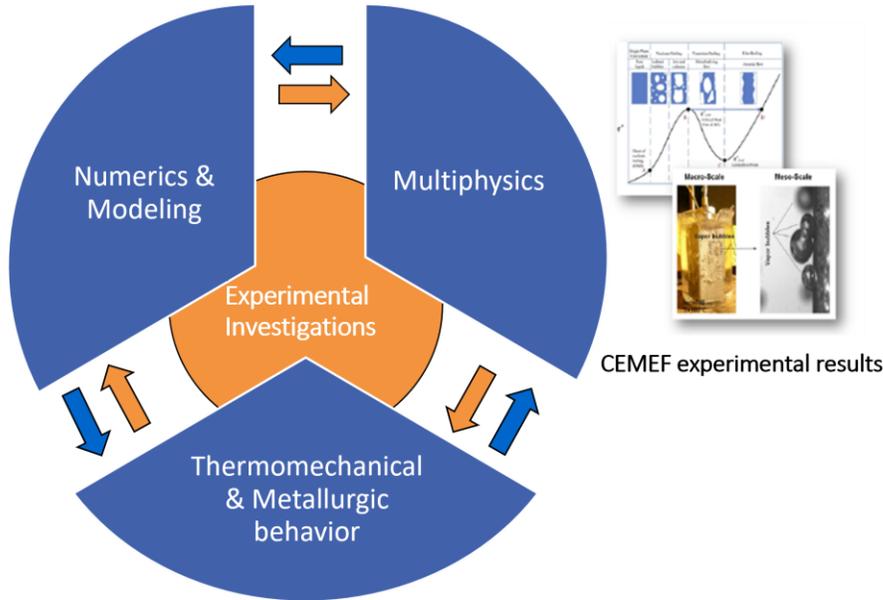
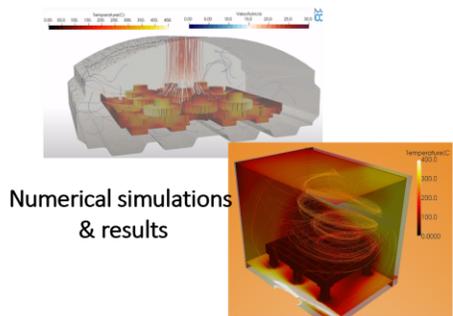
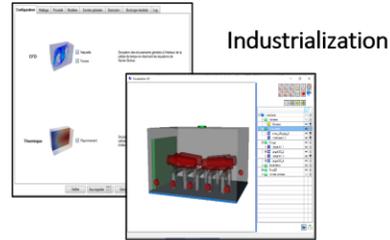
Hexagon The qobeo® project



◆ The qobeo® project

- ◆ Based on
 - ◆ Research work
 - ◆ Characterizations and experimental validations
- ◆ Continuous developments
 - ◆ Private fundings: the initial framework (~2006)
 - ◆ ADEME: Ademe call
 - ◆ ANR-Cosinus (**REALisTIC**):
 - ◆ Large Scale and Long Time Computation
 - ◆ ANR (**HECO**):
 - ◆ Moving mesh for heating and cooling processes
 - ◆ ANR Industrial Chair (**INFINITY**):
 - ◆ Numerical and physical framework for boiling and quenching applications





S2S Radiation "Surface to Surface"
(Four électrique, déplacement, ...)

AIR Heating & Air quenching

BUB Water/oil quenching

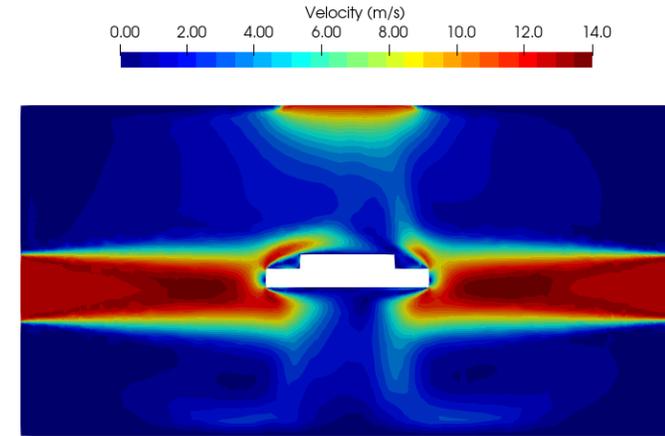


qobeo-AIR® : software principles

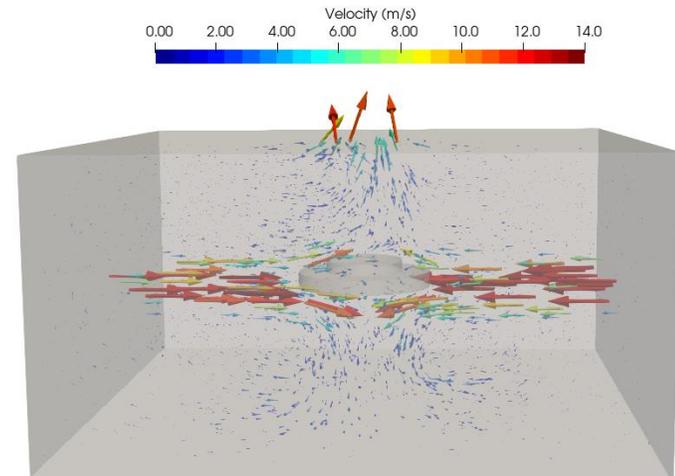


- ◆ Simulates air heating / cooling
- ◆ Tetrahedral Meshes, Anisotropic meshes
- ◆ Finite Element method
- ◆ **CFD Solver** applied on the hollow mesh
 - ◆ Natural Convection
 - ◆ Forced Convection
 - ◆ Choice of turbulent model:
 - ◆ K-epsilon
 - ◆ Smagorinsky
 - ◆ PID Regulation for nozzles flow rate

Cooling of a load from 2 nozzles



Velocity field evolution

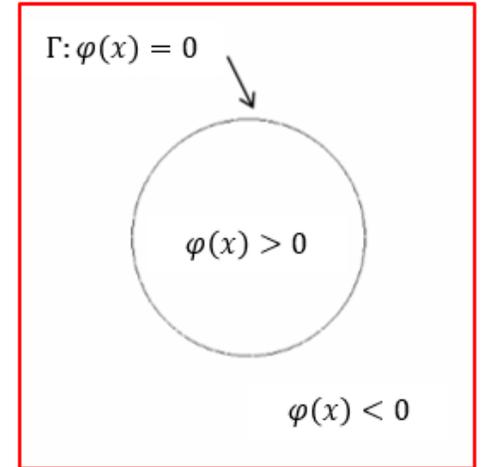
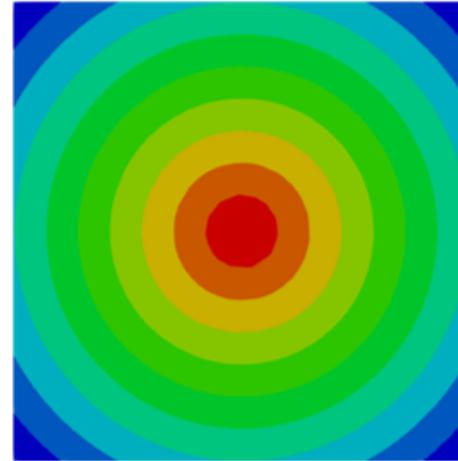




Immersed Volume Method

- ◆ Only one domain considered (the furnace)
 - ◆ Loads immersed in domain
 - ◆ LevelSet Function
 - ◆ Signed distance function to the interface Γ
 - ◆ Localize interface of immersed body

$$\phi(x) = \begin{cases} \text{dist}(x, \Gamma) & \text{if } x \in \Omega_s \\ 0 & \text{if } x \in \Gamma \\ -\text{dist}(x, \Gamma) & \text{if } x \in \Omega_f \end{cases}$$



Level set function

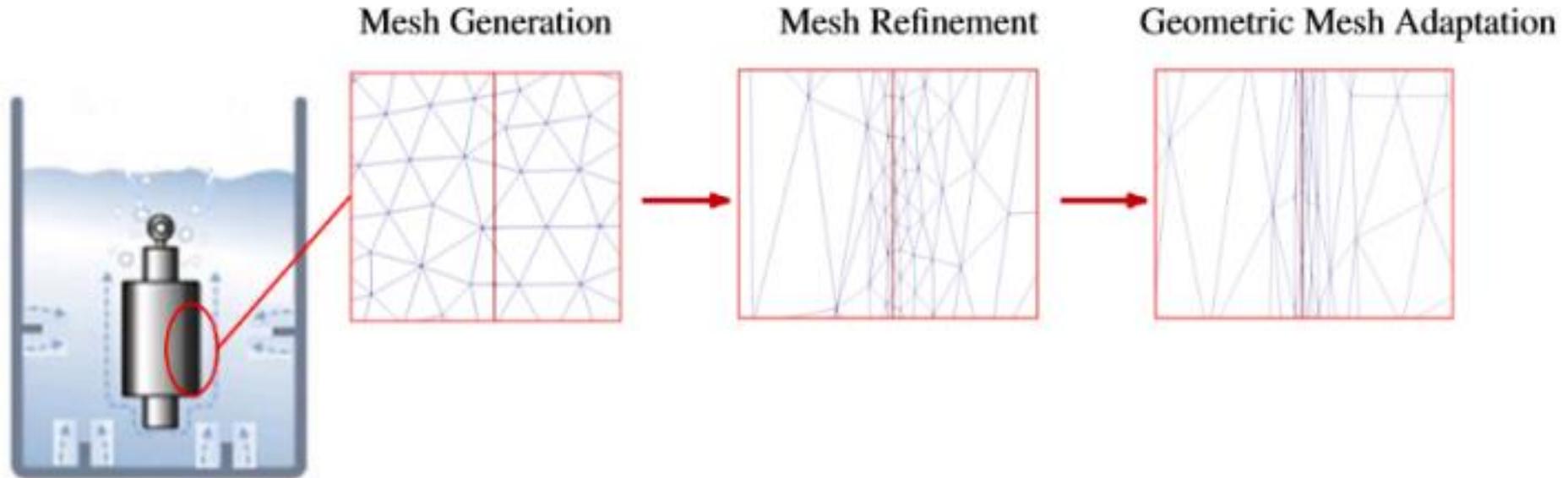
S. Osher, J. A. Sethian, Fronts propagating with curvature-dependent speed: Algorithms based on Hamilton -Jacobi formulations, Journal of computational physics 79 (1) (1988) 12-49

O. Desjardins, V. Moureau, H. Pitsch, An accurate conservative level set/ghost fluid method for simulating turbulent atomization, Journal of computational physics 227 (18) (2008) 8395-8416.



Immersed Volume Method / Meshing

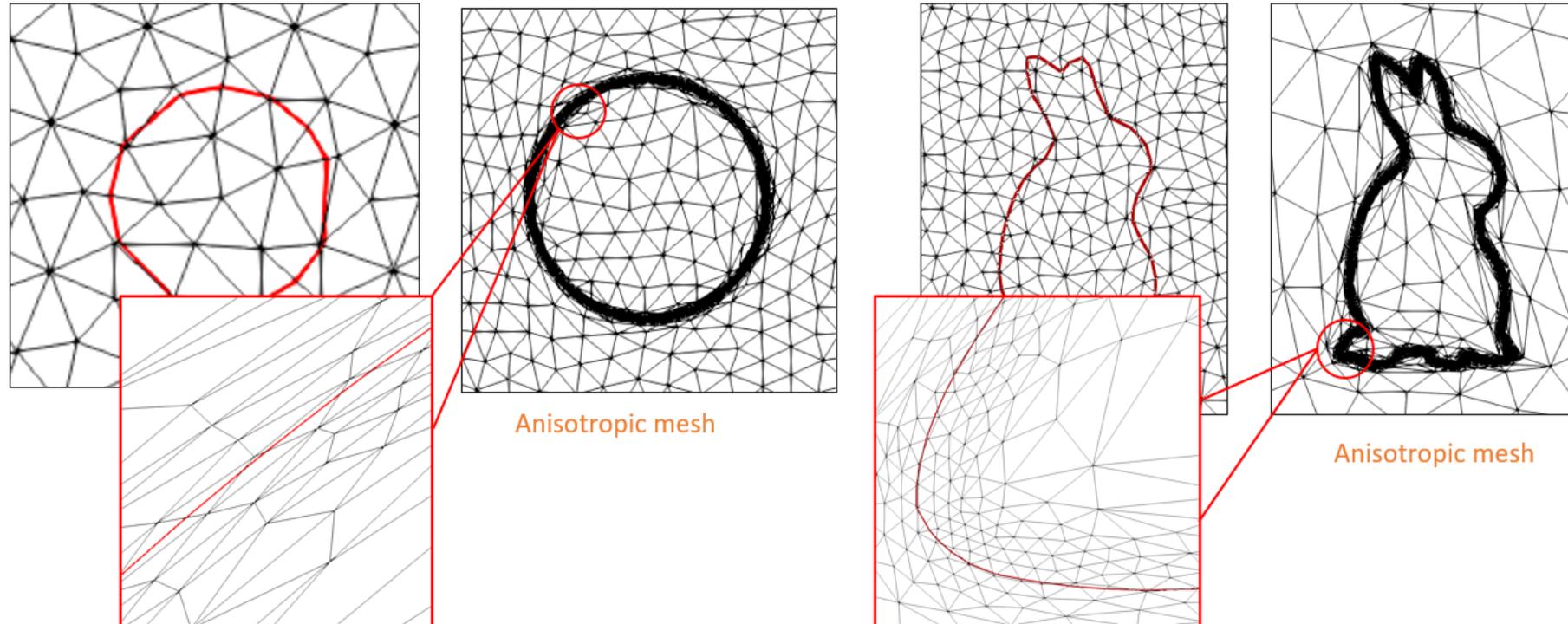
- ◆ Only one domain considered
 - ◆ Mesh refinement close to the edges
 - ◆ Anisotropic meshes (tetrahedral meshes)





Immersed Volume Method / Meshing

- ◆ Only one domain considered
 - ◆ Mesh refinement close to the edges
 - ◆ Anisotropic meshes

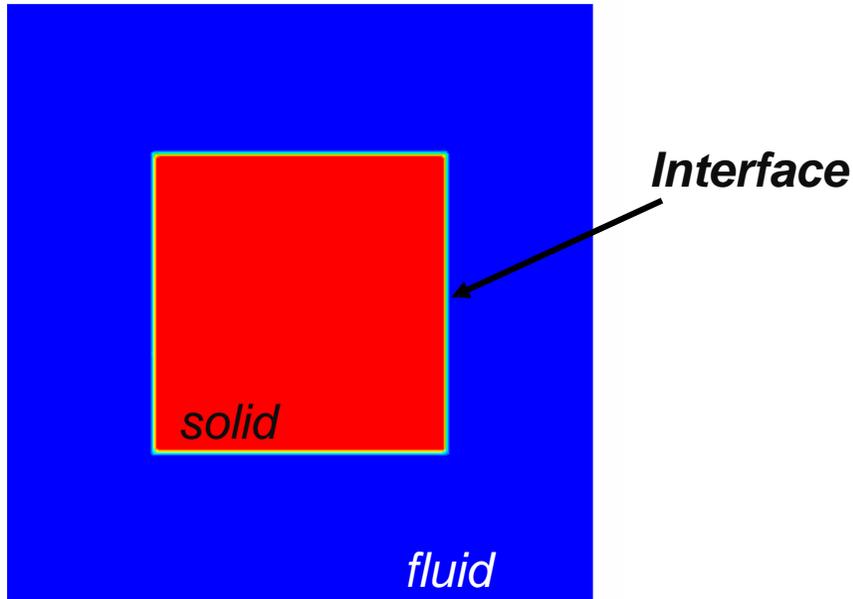




Immersed Volume Method

- ◆ Mixing laws for thermal properties
 - ◆ Smoothed Heaviside function on a narrow band region ϵ

$$H_\epsilon(x) = \begin{cases} 1 & \text{for } \phi(x) > \epsilon \\ \frac{1}{2} \left(1 + \frac{\phi(x)}{\epsilon} + \frac{1}{\pi} \sin\left(\frac{\pi\phi(x)}{\epsilon}\right) \right) & \text{for } |\phi(x)| \leq \epsilon \\ 0 & \text{for } \phi(x) < -\epsilon \end{cases}$$



$$\rho = \rho_{fluid} H_\epsilon(\phi(x)) + \rho_{solid} (1 - H_\epsilon(\phi(x)))$$

$$\eta = \eta_{fluid} H_\epsilon(\phi(x)) + \eta_{solid} (1 - H_\epsilon(\phi(x)))$$

$$k = \left(\frac{H_\epsilon(\phi(x))}{k_{fluid}} + \frac{1 - H_\epsilon(\phi(x))}{k_{solid}} \right)^{-1}$$



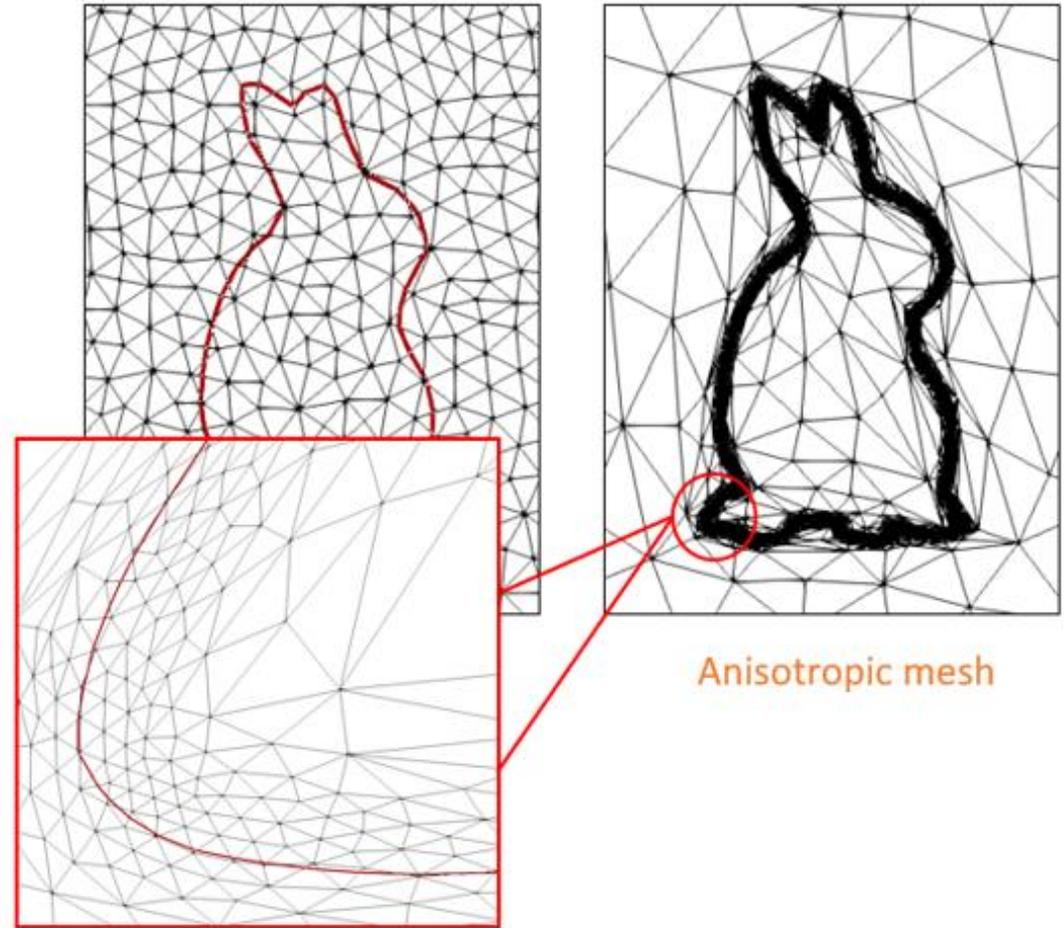
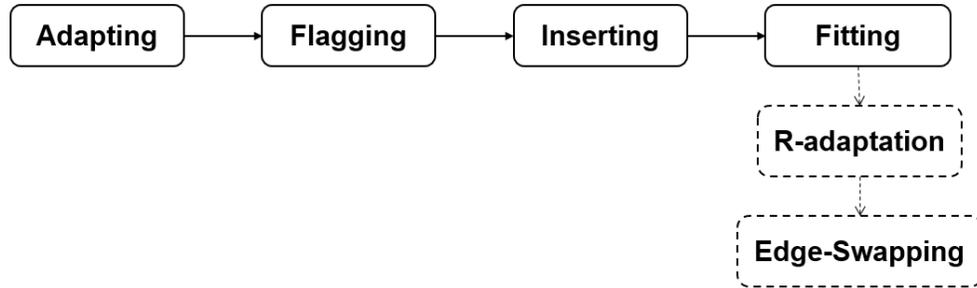
◆ Immersed Volume Method

- ◆ Mixing laws for thermal properties
 - ◆ Advantages : Suitable for most of tetrahedral meshes
 - ◆ Drawbacks : the ε parameters should be related to the mesh size.
 - ◆ What to choose when you have different mesh sizes ?
 - ◆ High impact on the results if not well defined



How to remove mixing laws

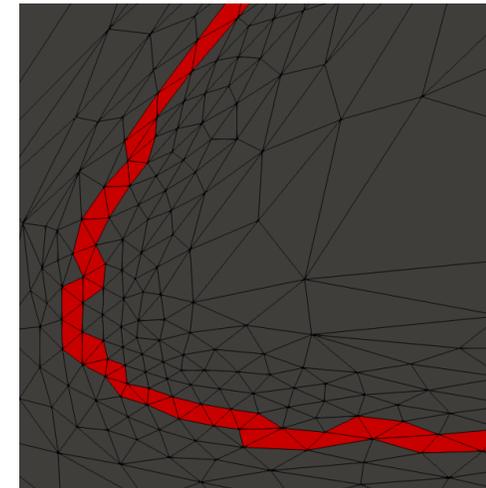
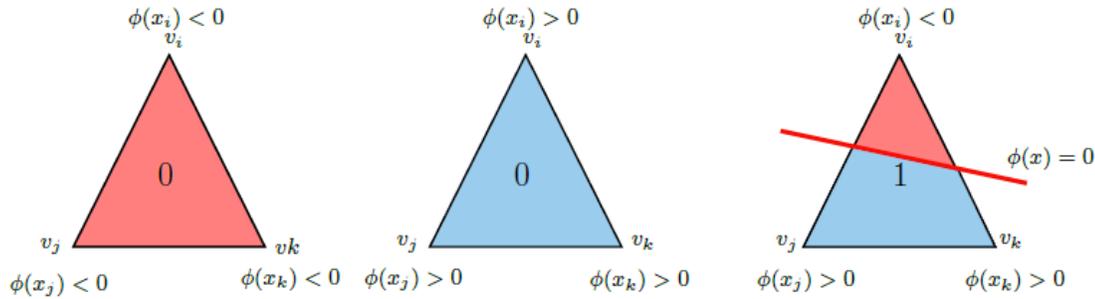
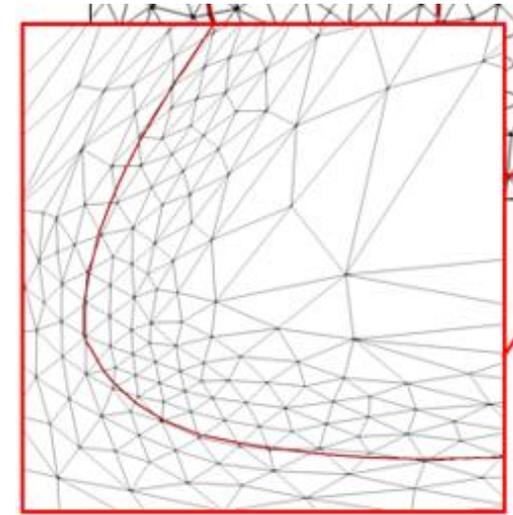
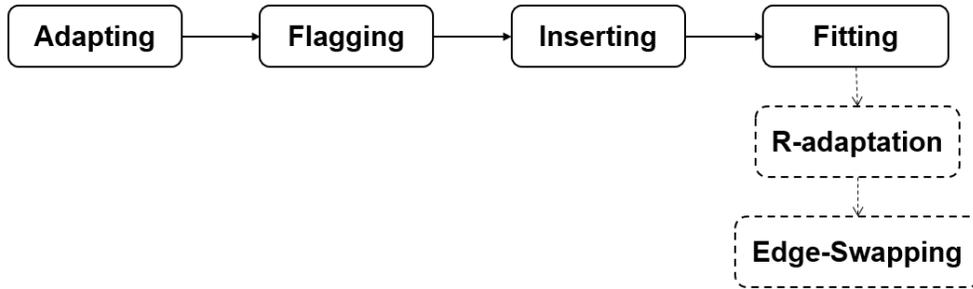
- ◆ Fitted Meshes
 - ◆ Mesh elements are only in one region (fluid, loads, ...)
 - ◆ No need to mix thermo-physical properties
- ◆ How to obtain fitted meshes





How to remove mixing laws

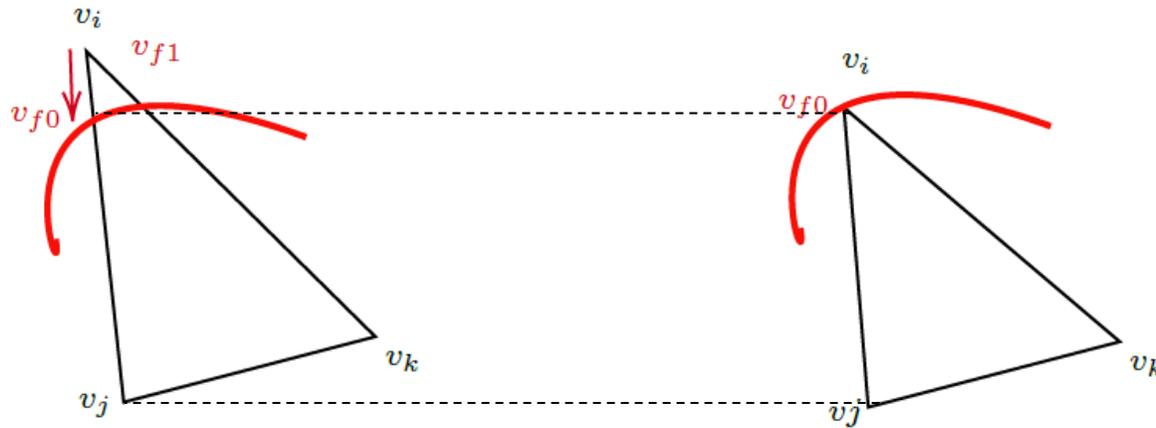
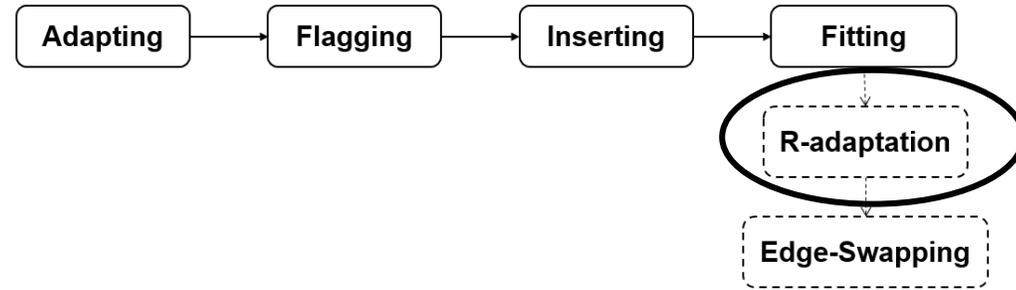
How to obtain fitted meshes





How to remove mixing laws

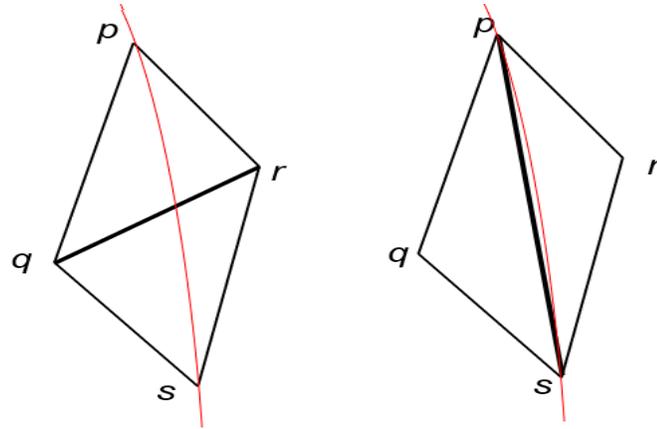
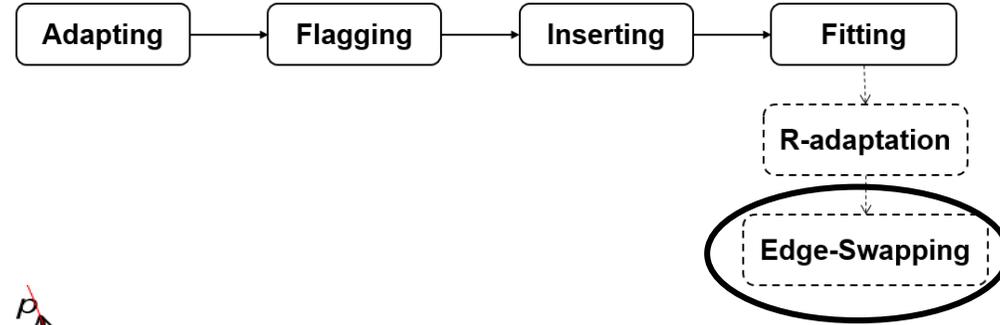
- How to obtain fitted meshes





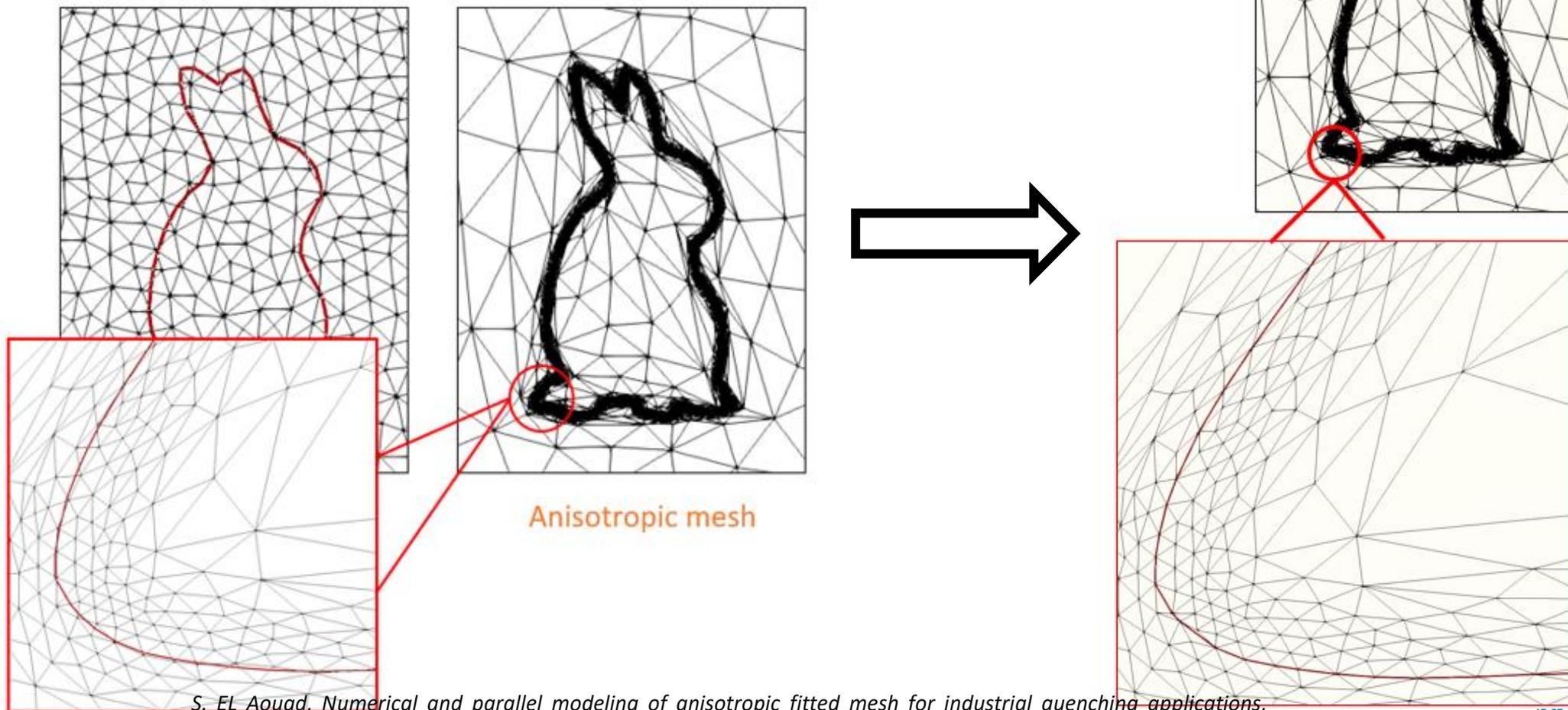
How to remove mixing laws

How to obtain fitted meshes



How to remove mixing laws

Fitted Meshes



◆ qobeo-AIR® : software principles



◆ Thermal Computation

◆ With or without Radiation
by activating **S2S model**

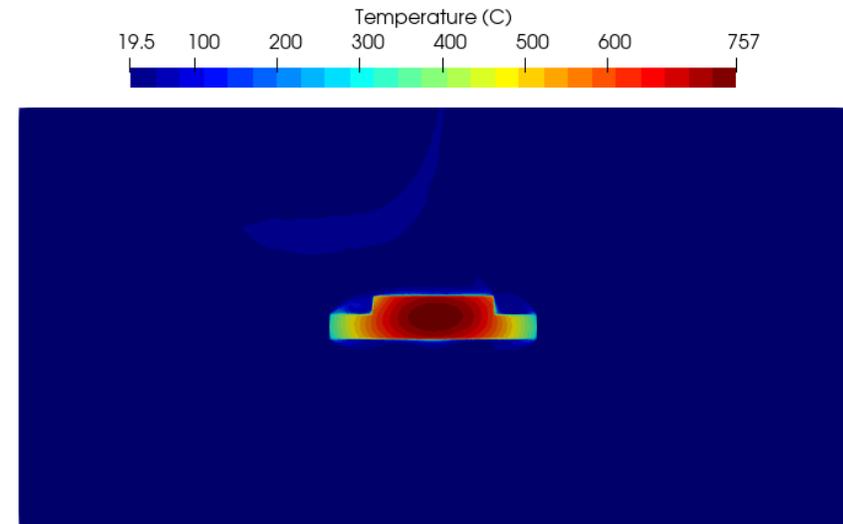
◆ S2S model take into account "obstruction"

◆ PID regulation of the thermal resistance

◆ Thermal solver applied on the whole domain
(gas + load(s))

◆ **Contact** between different loads is also considered

Temperature evolution





gobeo-AIR[®] : software principles

- ◆ S2S radiation model : Surface to Surface
- ◆ Radiosity equation

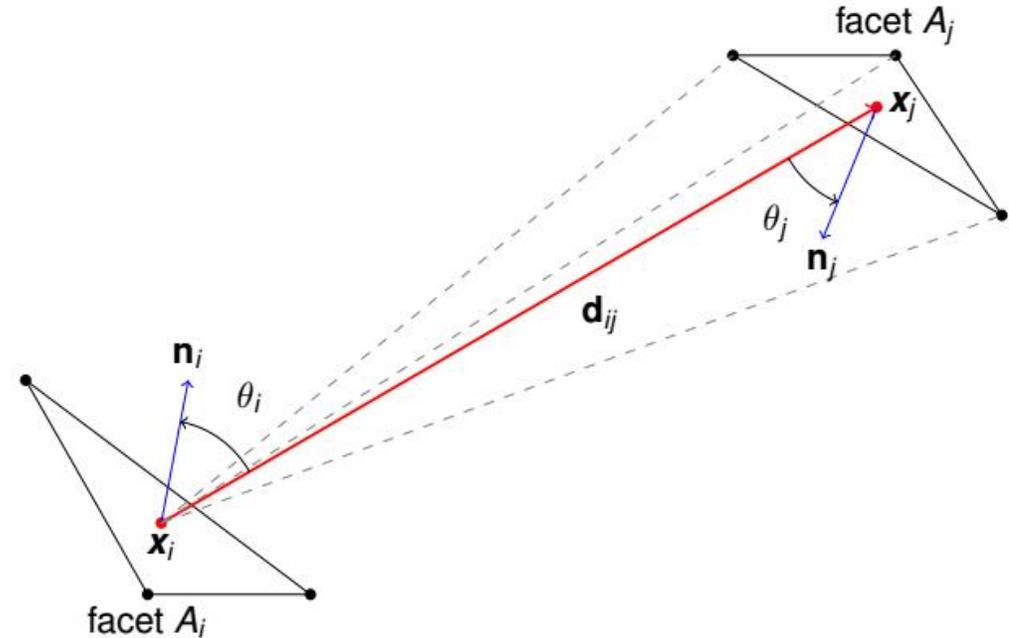
$$J_i = \underset{\substack{\downarrow \\ \text{emitted}}}{\varepsilon_i \sigma T_i^4} + (1 - \varepsilon_i) \sum_j \underset{\substack{\downarrow \\ \text{reflected}}}{F_{ji} J_j}$$

- ◆ The view factors

$$F_{ij} = |A_j| \frac{\cos(\vartheta_i) \cos(\vartheta_j)}{\pi |d_{ij}|^2}$$

- ◆ Radiative flux

$$Q_i = |A_i| \frac{\varepsilon_i}{1 - \varepsilon_i} (\sigma T_i^4 - J_i)$$

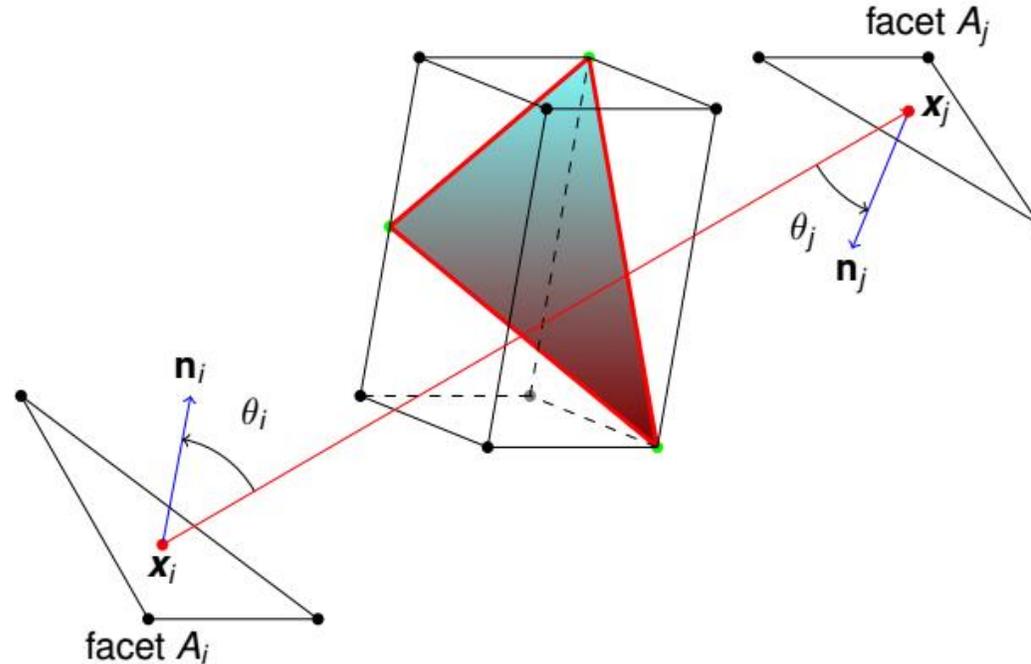


R. Gerard, Fast, flexible Surface to Surface thermal radiative transfer for sintering applications, Thèse de doctorat Mathématiques numériques, Calcul intensif et Données Université Paris sciences et lettres 2022



qobeo-AIR® : software principles

- ◆ S2S radiation model : Surface to Surface
 - ◆ Compute the a priori visibility of pairs of facets (A_i, A_j) by half-plane criterion
 - ◆ Compute the obstruction for pairs of facets (A_i, A_j) of the reduced subset



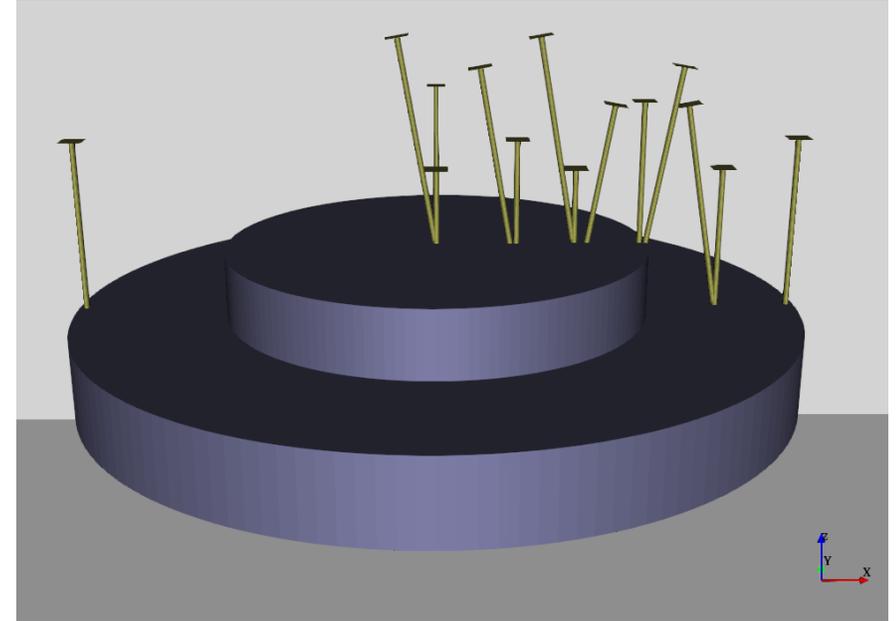
- ◆ Complexity issues : segment-tree for ray-box collisions.



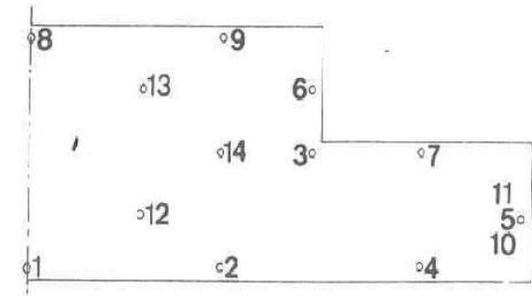
gobeo-AIR® : software principles

- ◆ S2S radiation model : Experimental validation test case
 - ◆ Provided by Safran

Resources	Type	Parameters	Position
Load	Disque Chapeau	$R_{small}=0.145m,$ $H_{small}=0.05m,$ $R_{large}=0.25m,$ $H_{large}=0.055m,$	(0,0,0)



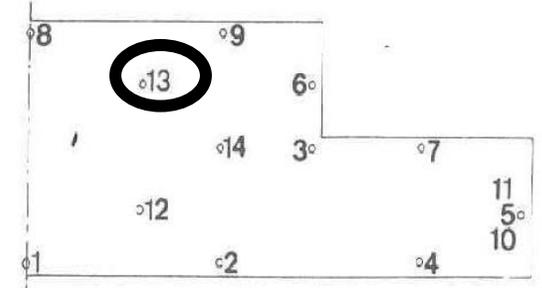
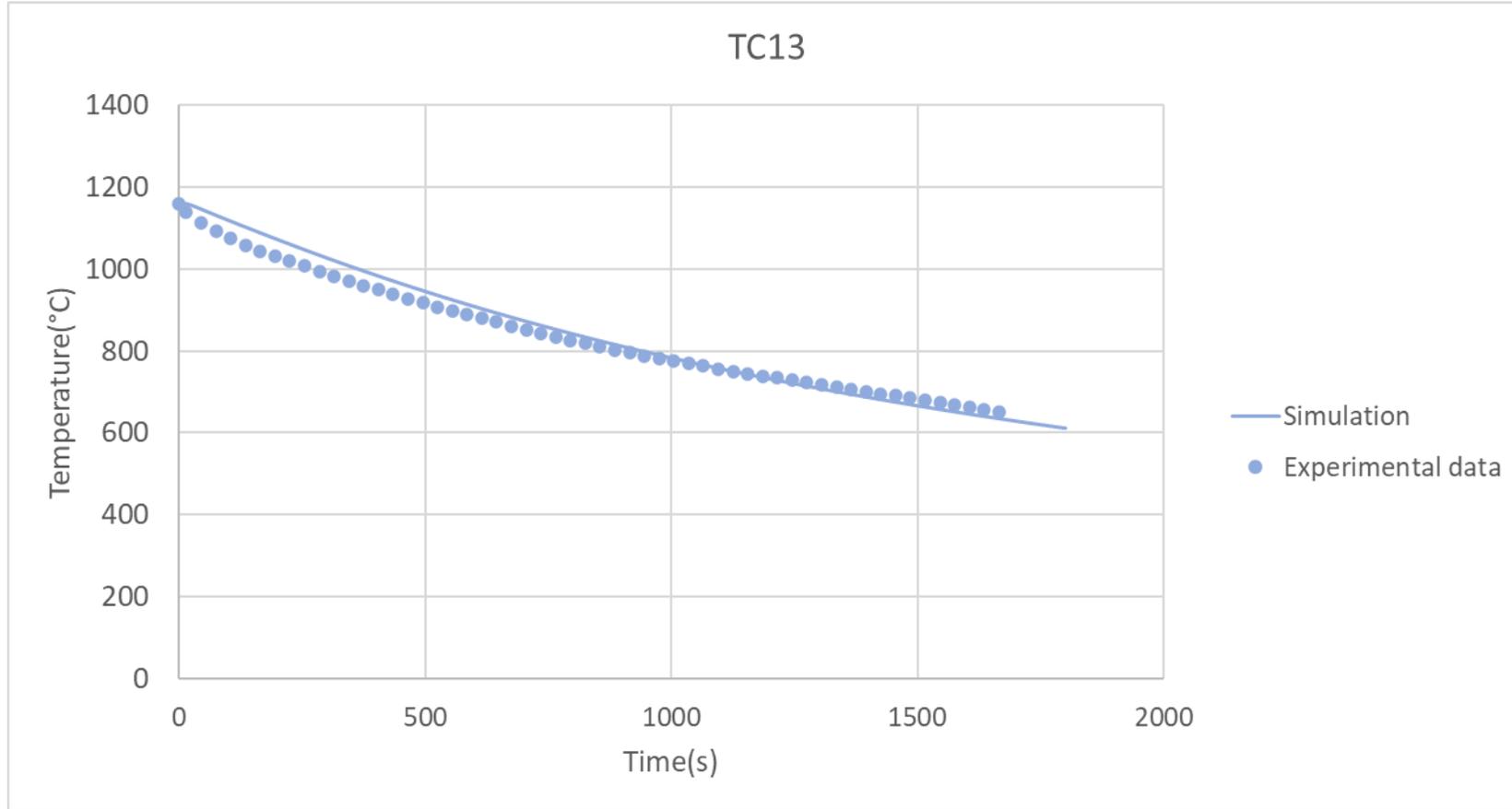
	Load
Material	Inco718
Temperature	1160°C
Emissivity	0,717





gobeo-AIR® : software principles

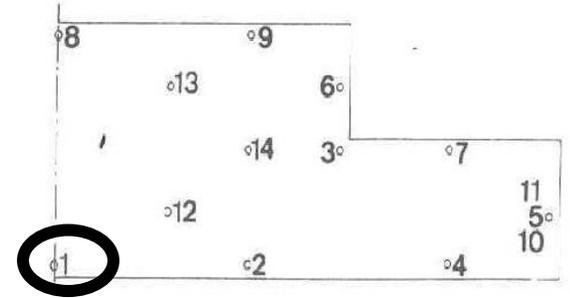
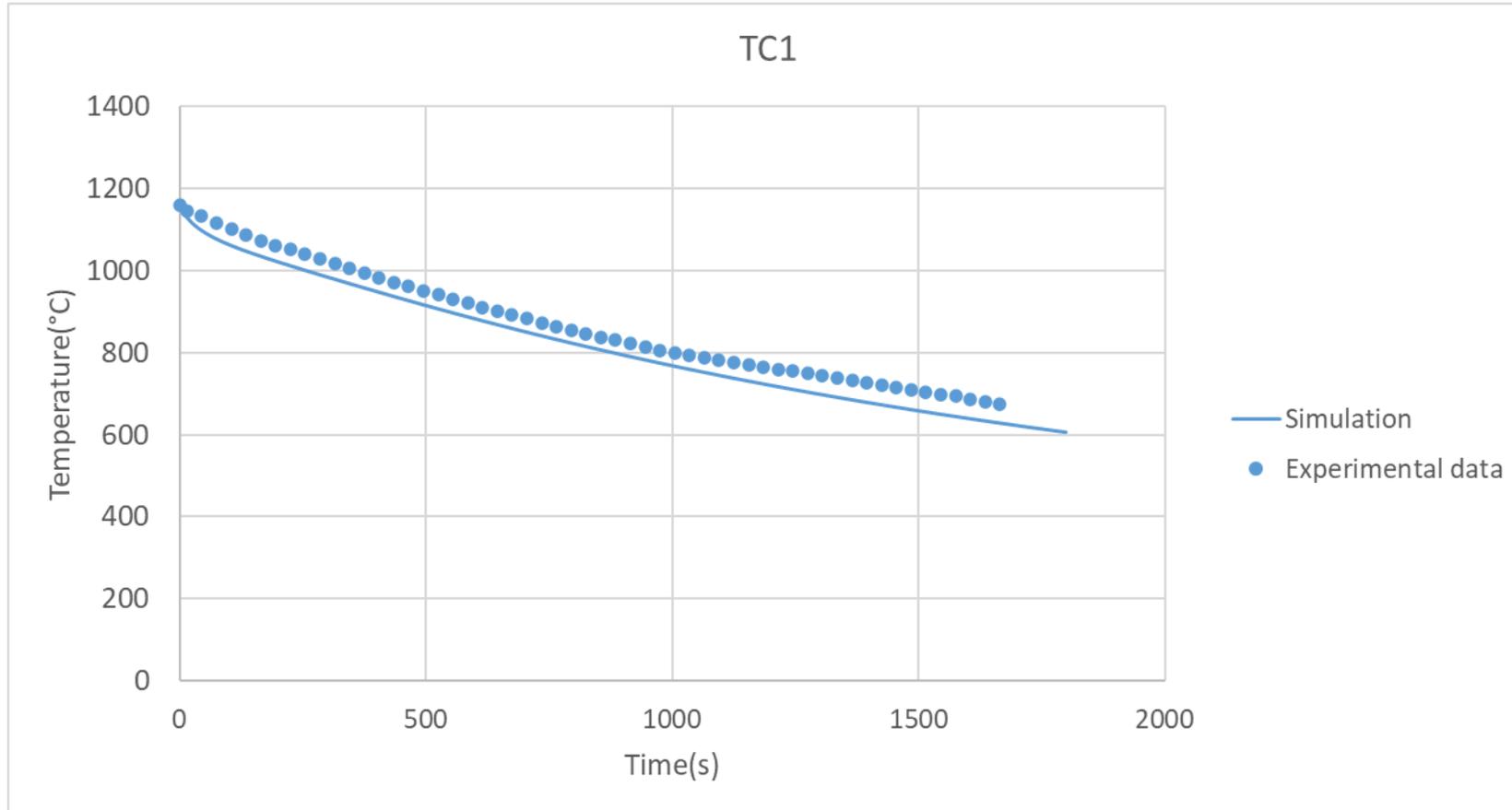
- ◆ S2S radiation model : Experimental validation test case





gobeo-AIR[®] : software principles

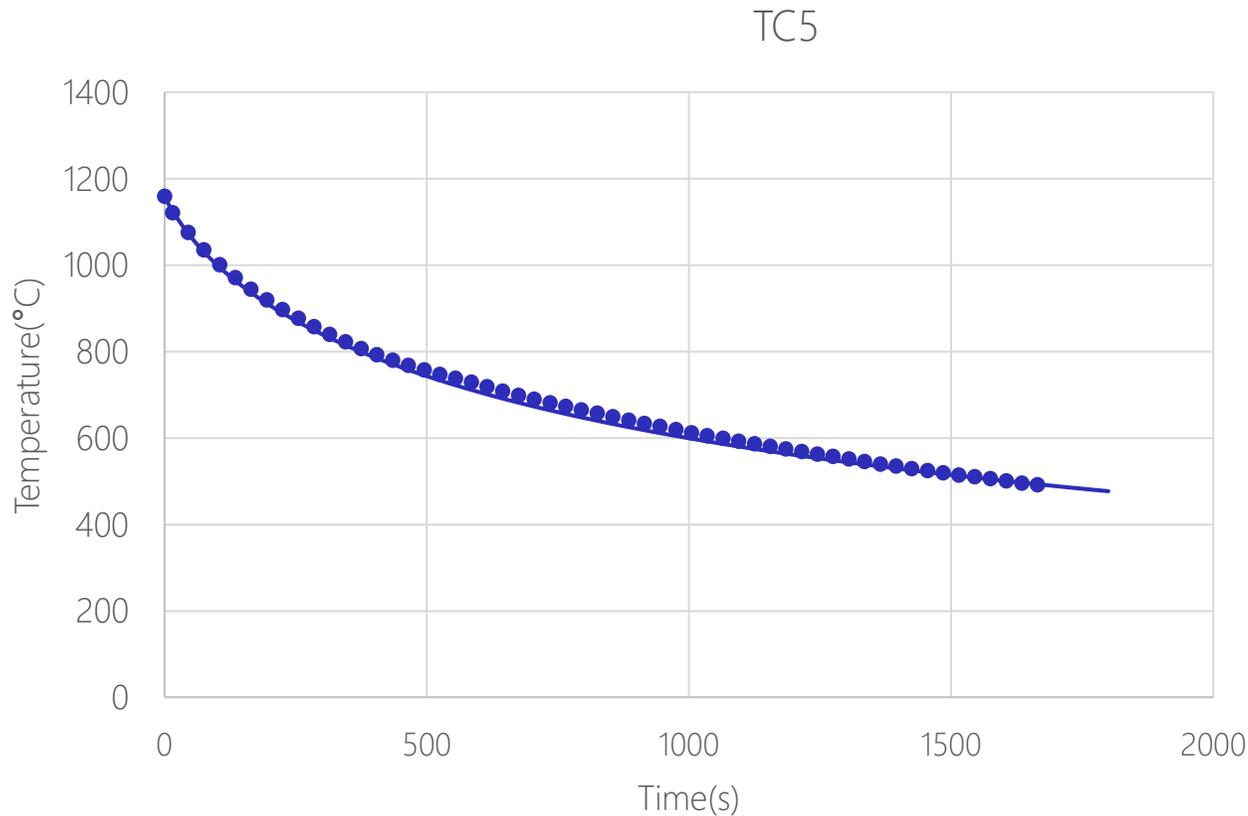
- ◆ S2S radiation model : Experimental validation test case
 - ◆ The “support” is neglected in the simulation.



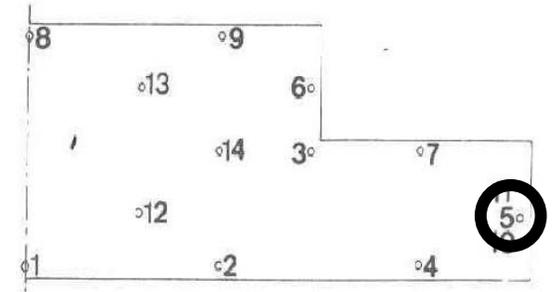
gobeo-AIR[®] : software principles



- ◆ S2S radiation model : Experimental validation test case



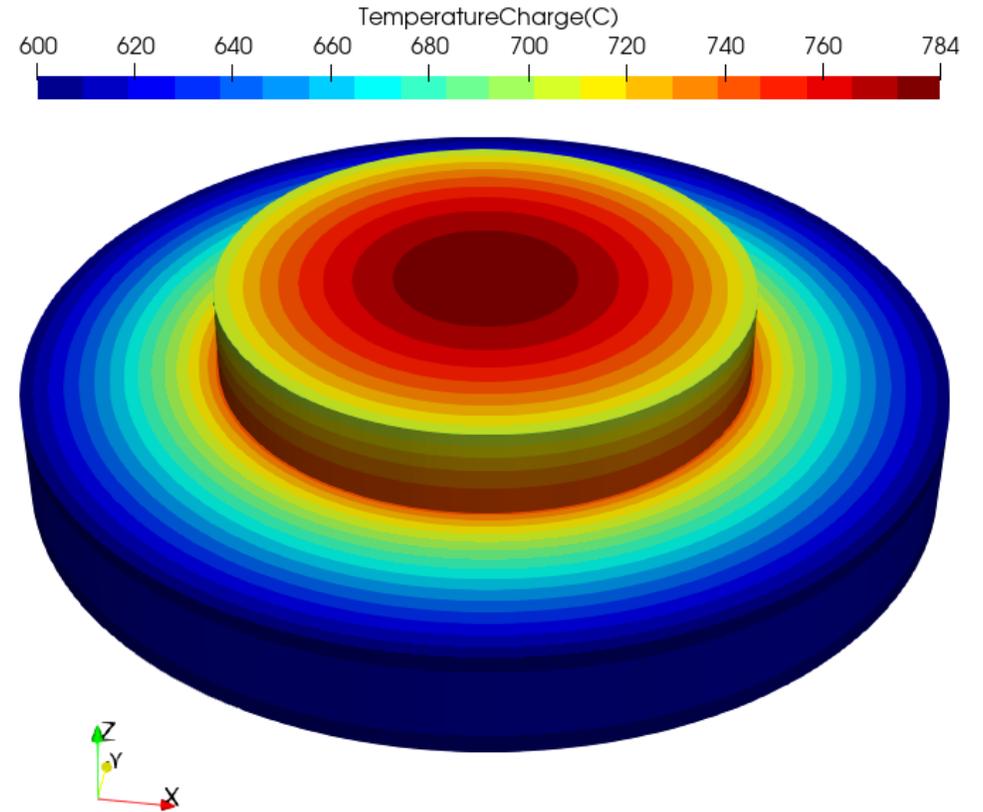
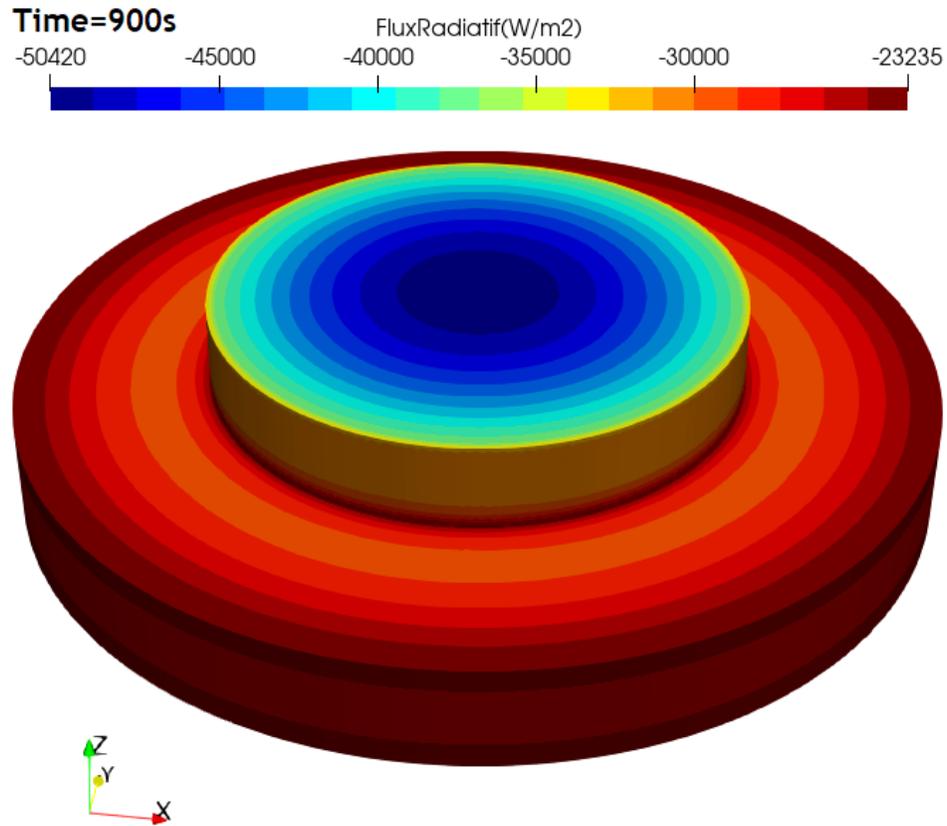
— Simulation
● Experimental data





gobeo-AIR® : software principles

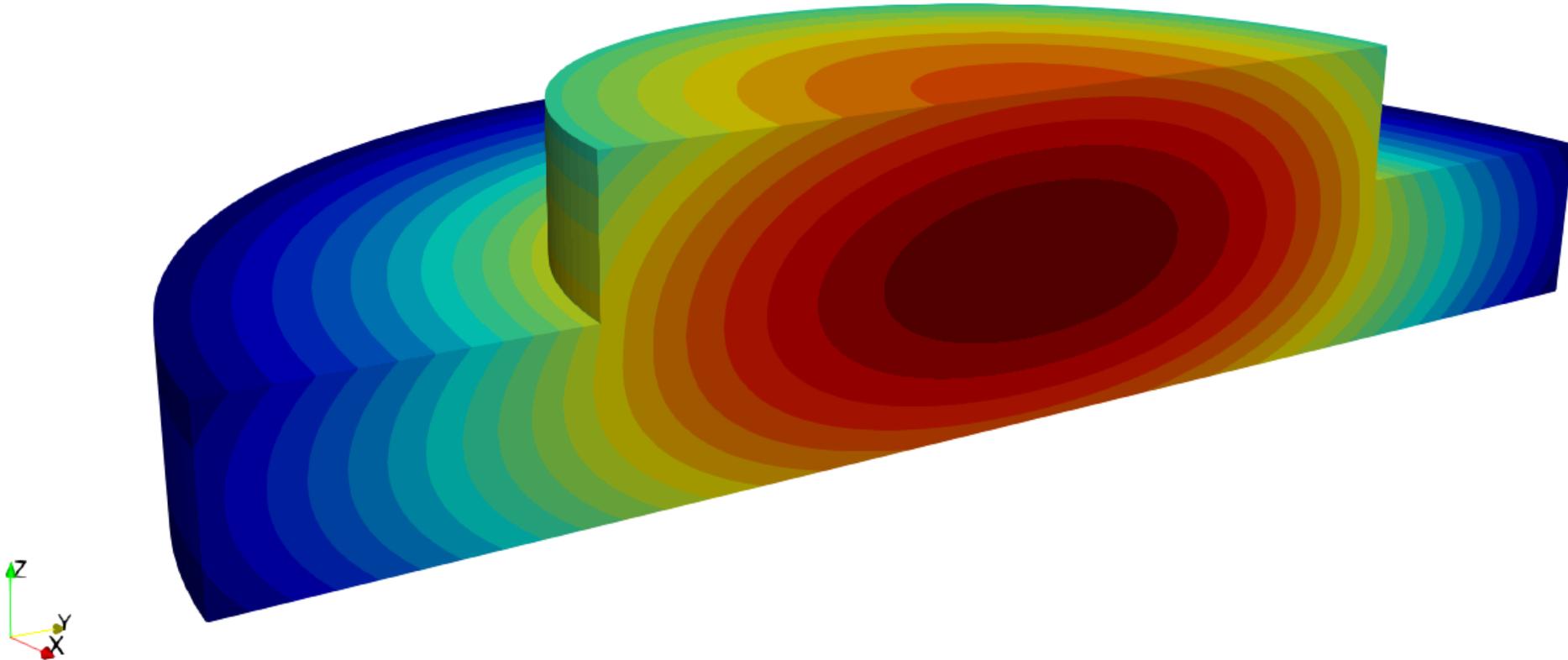
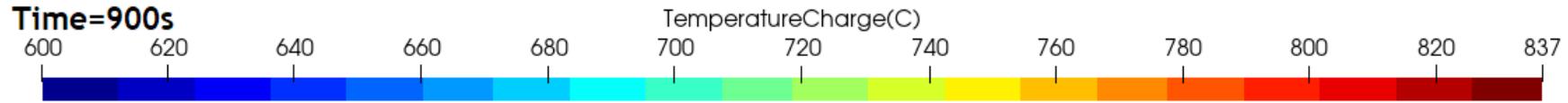
- ◆ S2S radiation model : Experimental validation test case





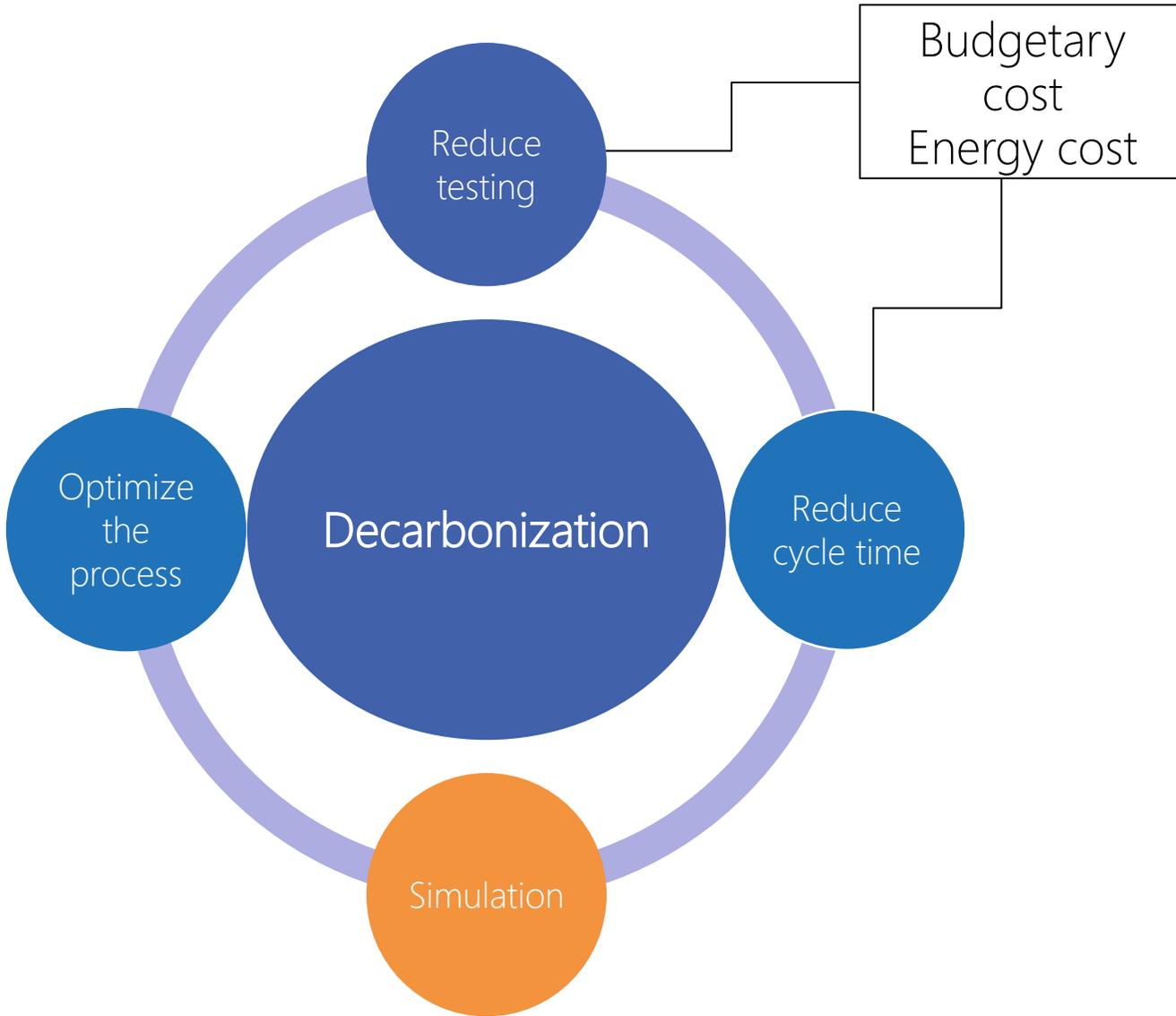
qobeo-AIR[®] : software principles

- ◆ S2S radiation model : Experimental validation test case



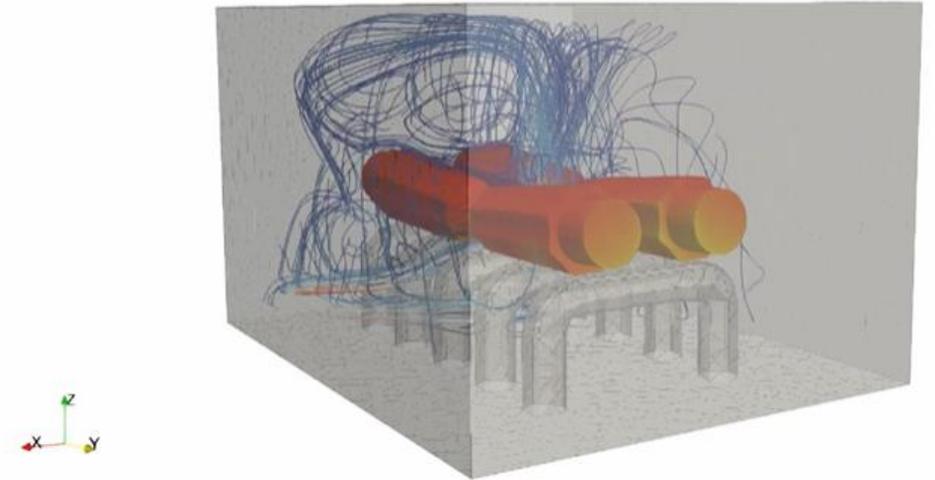


Context & objectives



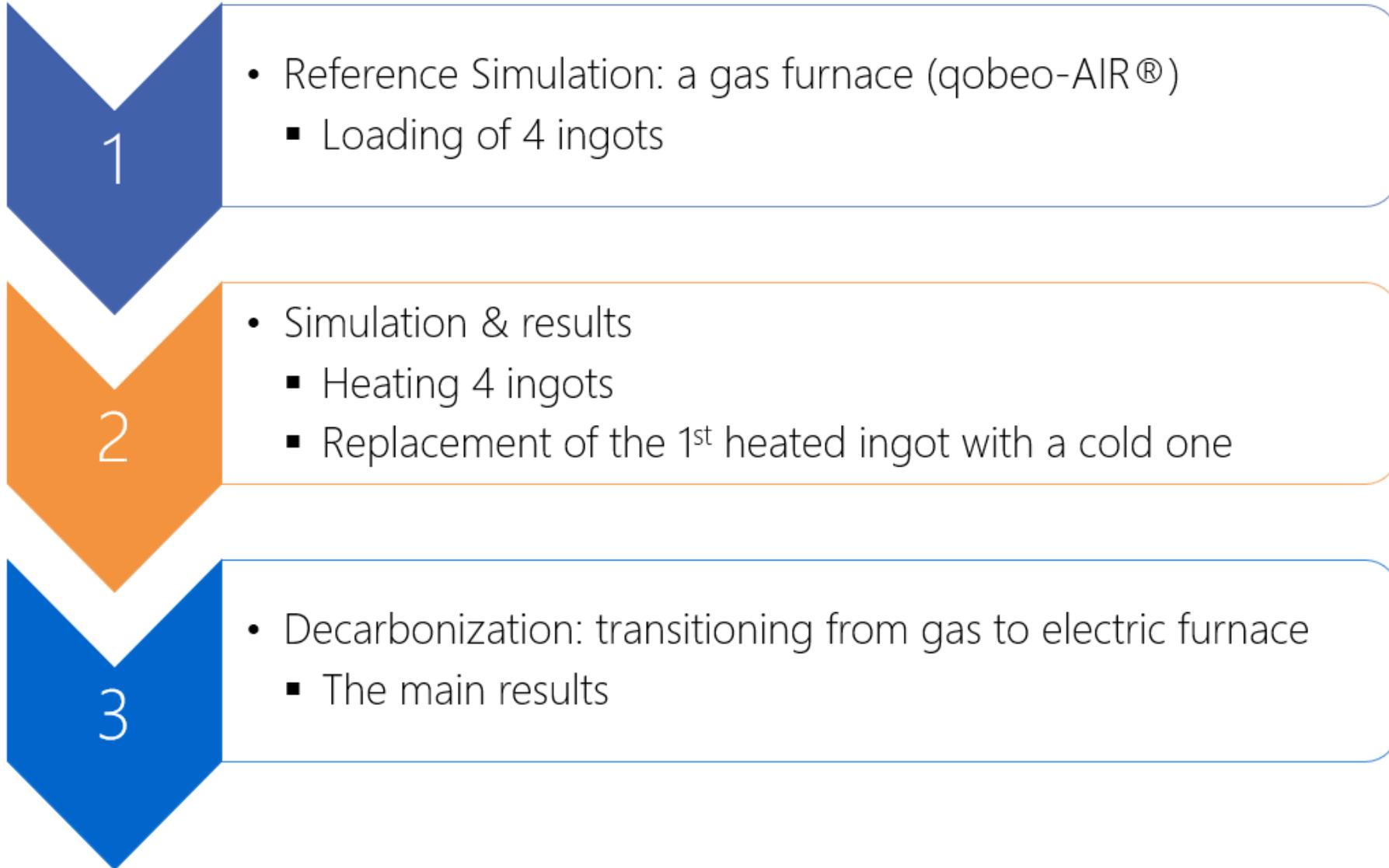
Analysis of a furnace with ingot loading

- How to anticipate heating times?
- How to consider the impact of different process scenarios on the heating of the pieces?





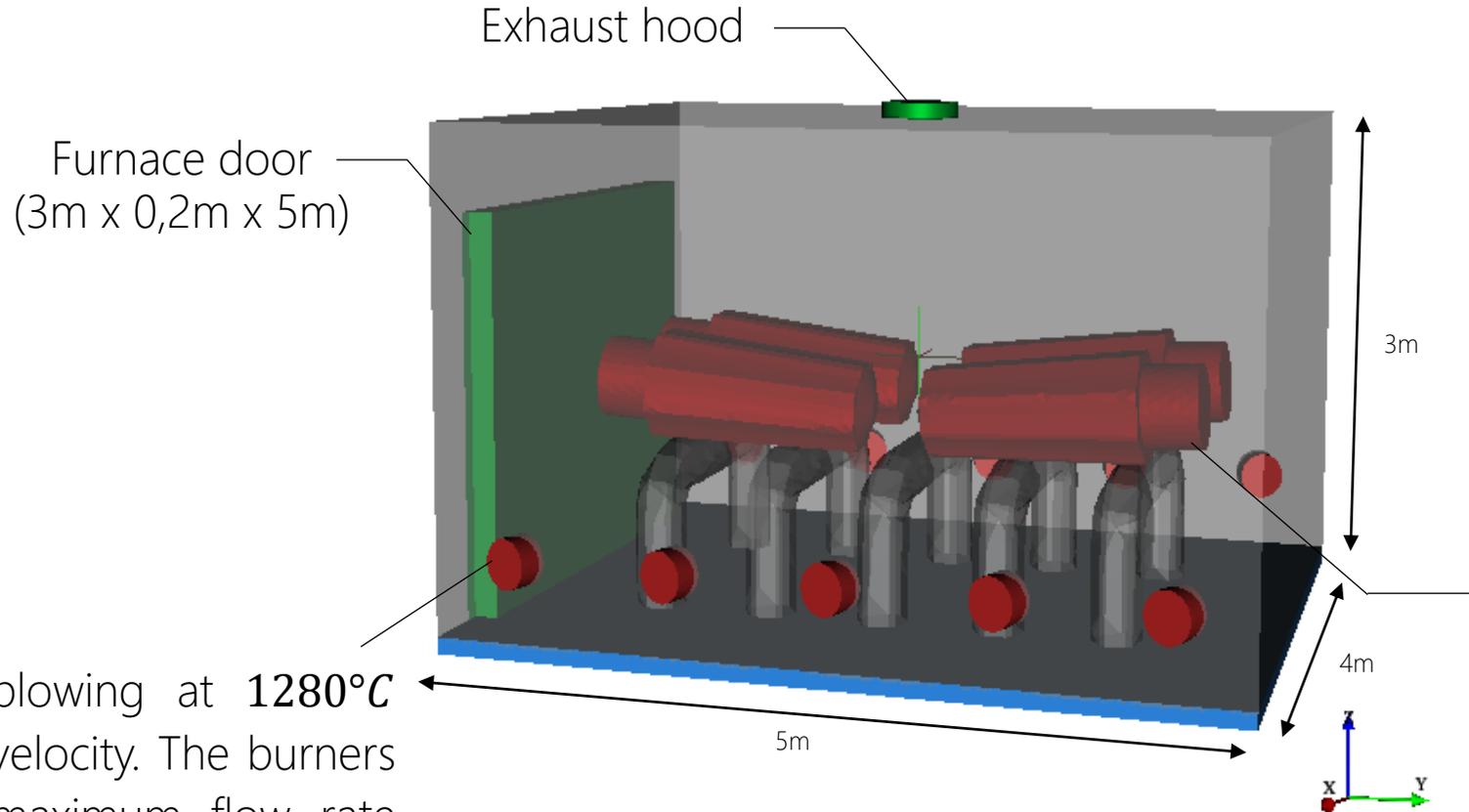
Our approach





Case Implementation

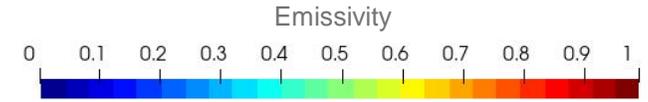
- Gas furnace (4m x 5m x 3m) with air flow heating.



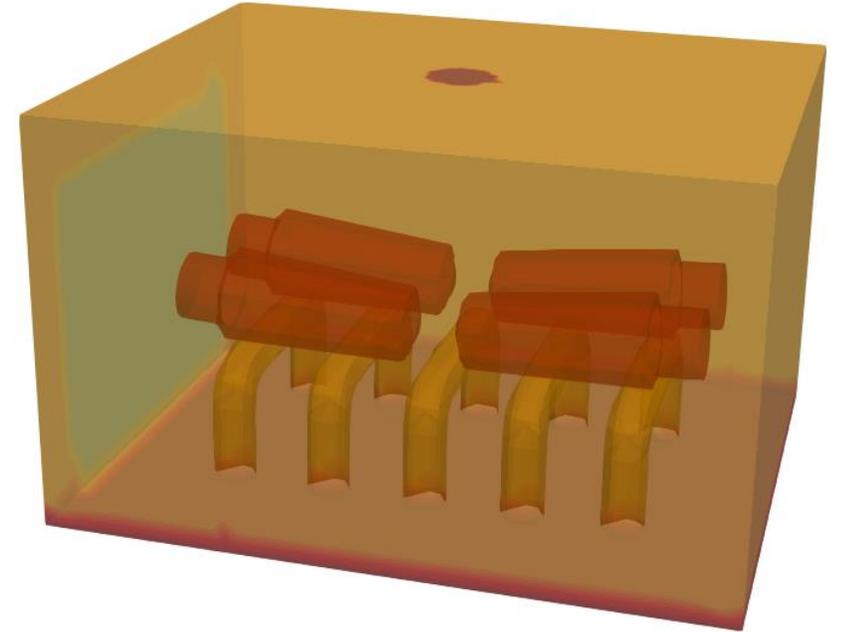
4 ingots weighing 3.3 tons each, with $T_{initial} = 400^{\circ}C$

10 burners blowing at $1280^{\circ}C$ with $17m/s$ velocity. The burners reach their maximum flow rate after 1 minute from their start-up.

Case Implementation



- ◆ Fourier boundary conditions are applied on:
 - ◆ Furnace walls: $h = 5 \frac{W}{m^2.K}$; $T_{outside} = 50^\circ$; $\epsilon_{walls} = 0,7$.
 - ◆ Closed door: $h = 5 \frac{W}{m^2.K}$; $T_{outside} = 50^\circ$; $\epsilon_{door} = 0,4$.
- ◆ After 4,4 hours, the 1st ingot is replaced with a "cold" one at **400°C**.

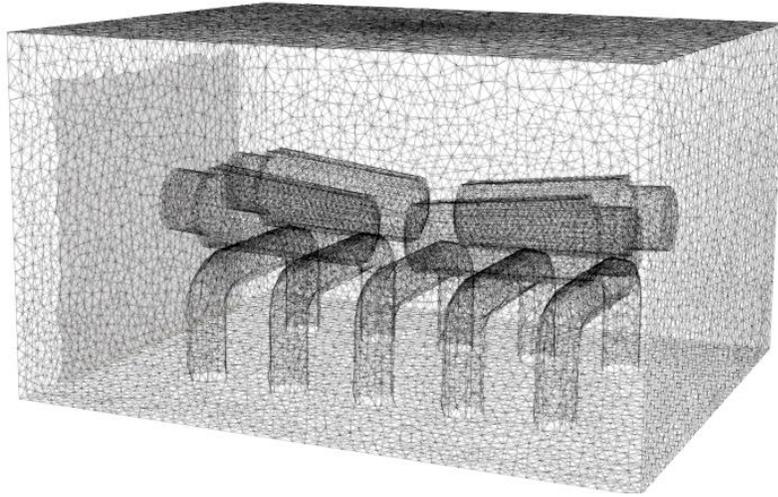


Visualization of different fields & thermal properties

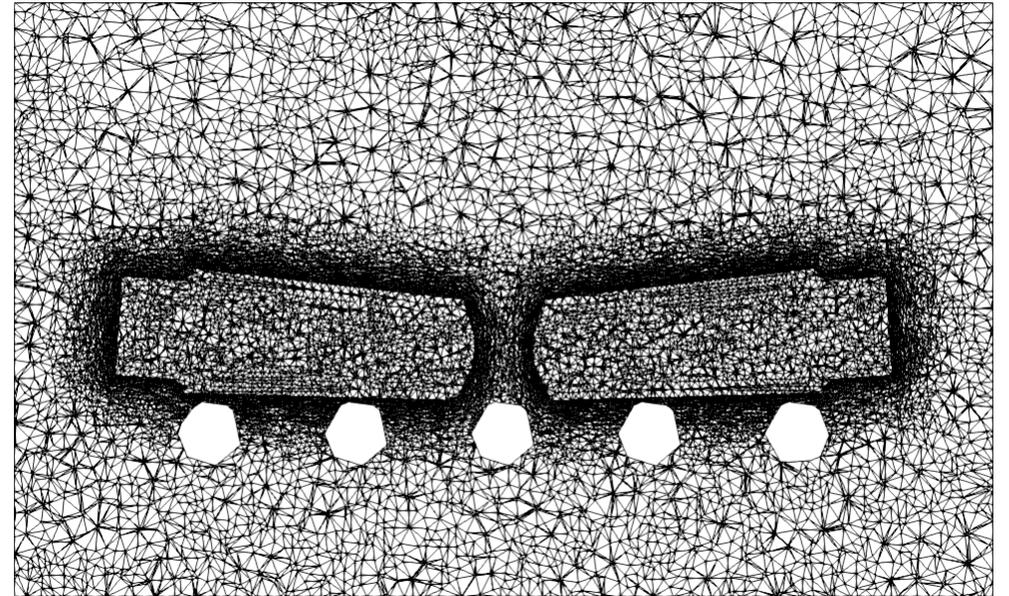


Case Implementation

- ◆ Automatic generation of surface and volume mesh for enclosures and loads.

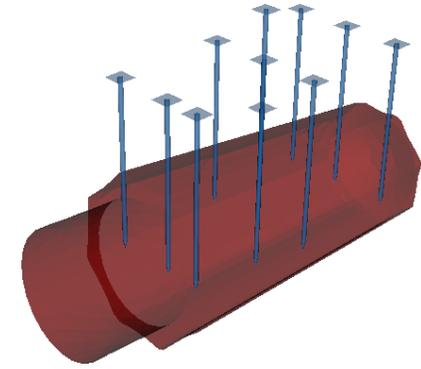


- ◆ Option to create a refined anisotropic mesh around the interfaces of the loads.



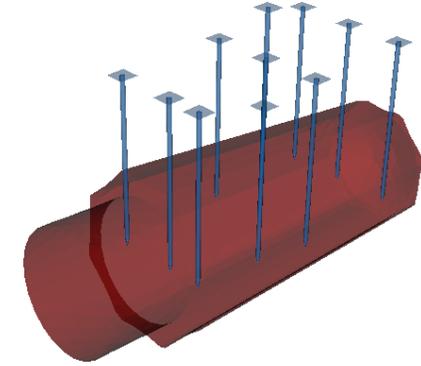
Simulation

- ◆ Numerical sensors are added in each ingot.
- ◆ For each step, we solve:
 - ◆ For the first **30 s**, with $\Delta t = 0,08 \text{ s}$:
 - ◆ Natural convection
 - ◆ Forced convection
 - ◆ Thermal calculation
- ◆ Computational cost (~1 month).



Simulation

- Numerical sensors are added in each ingot.



- Stabilized CFD

- After the first seconds, we computed an averaged velocity on 25s of physical time: $\mathbf{V}_m = \frac{1}{nb_incr} \sum_{i=1}^{nb_incr} \mathbf{v}_i$ and we imposed this velocity field as constant for the remaining time.

- For each step, we solve:

- For the first 30 s, with $\Delta t = 0,08$ s:

- Natural convection
- Forced convection
- Thermal calculation

- For the remaining calculation, thermal analysis is performed using the average velocity with $\Delta t = 0,25$ s, which helps reduce drastically the computational cost (~1 day vs 1 month).



Results – Heating 4 ingots

◆ Computational cost

◆ Full simulation

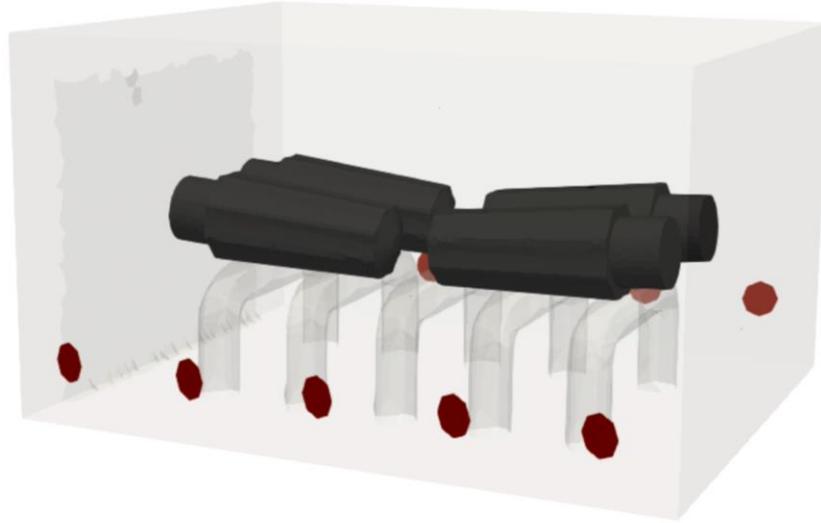
	Time step	Number of cores	Calculation duration
From 0 to 4,4 hours:	0,08 s	32 cores (@3,6 GHz, 2022)	~ 1 month

◆ Simulation with stabilized CFD

	Time step	Number of cores	Calculation duration
From 0 to 30 s:	0,08 s	32 cores (@3,6 GHz, 2022)	~ 8 hours
From 30 s to 4,4hours:	0,25 s	32 cores (@3,6 GHz, 2022)	~ 20 hours



Results – Heating 4 ingots



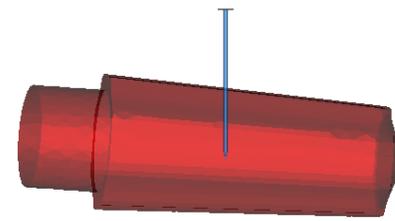
Time: 0.000 s

Obtained velocity & temperature fields.

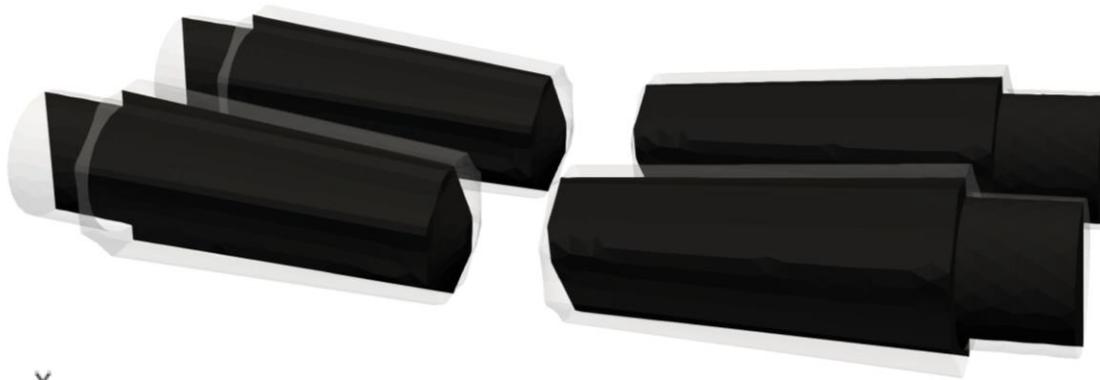
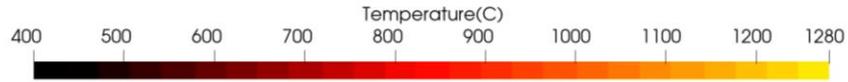
- ◆ Resolution of natural convection, forced convection, and radiation.
- ◆ The burners reach their maximal flow rate after 5 seconds.
- ◆ Stable velocity field is achieved after 30 seconds.

Results – Heating 4 ingots

Temperature evolution

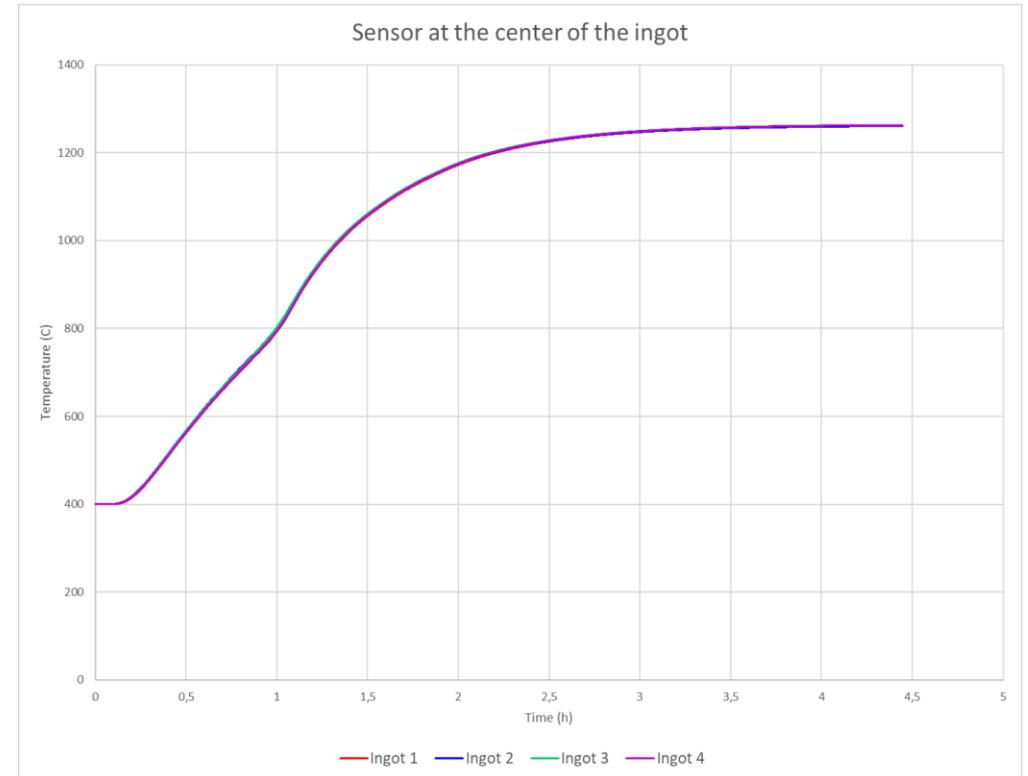


Position of the numerical sensor at the center of each ingot



Temperature evolution in the ingots

Time: 0.000 s



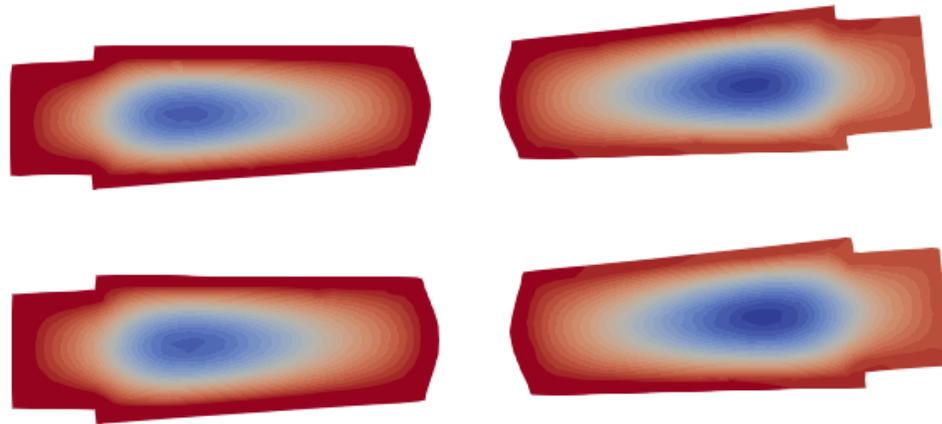
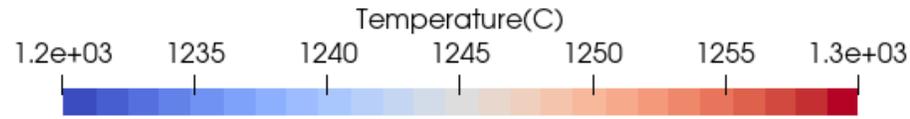
Optimization of cycle time: Homogeneous temperature achieved in 3.5 hours instead of 4.4 hours.



Results – Heating 4 ingots

Temperature evolution

Time: 2h45



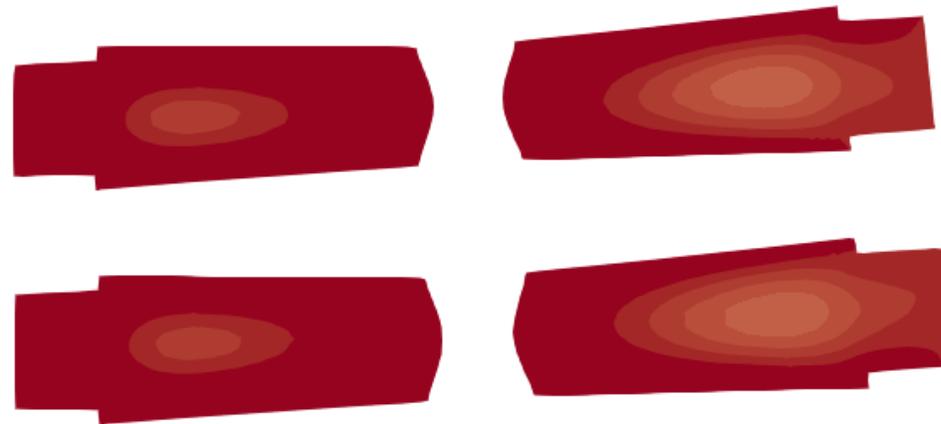
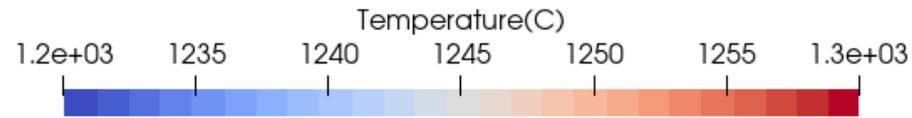
~30 °C between minimum and maximum



Results – Heating 4 ingots

- ◆ Temperature evolution

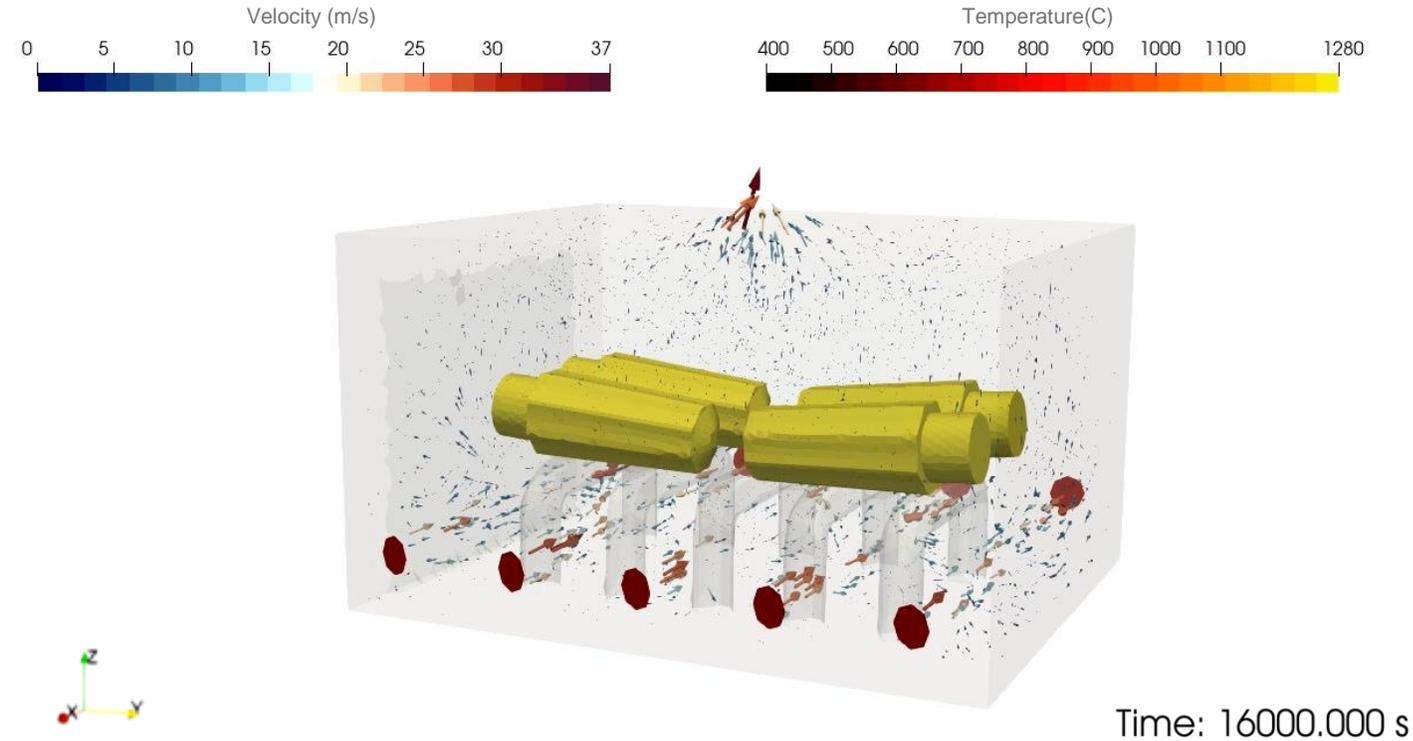
Time: 3h30



~5 °C between minimum and maximum



Results – Replacement of the 1st ingot



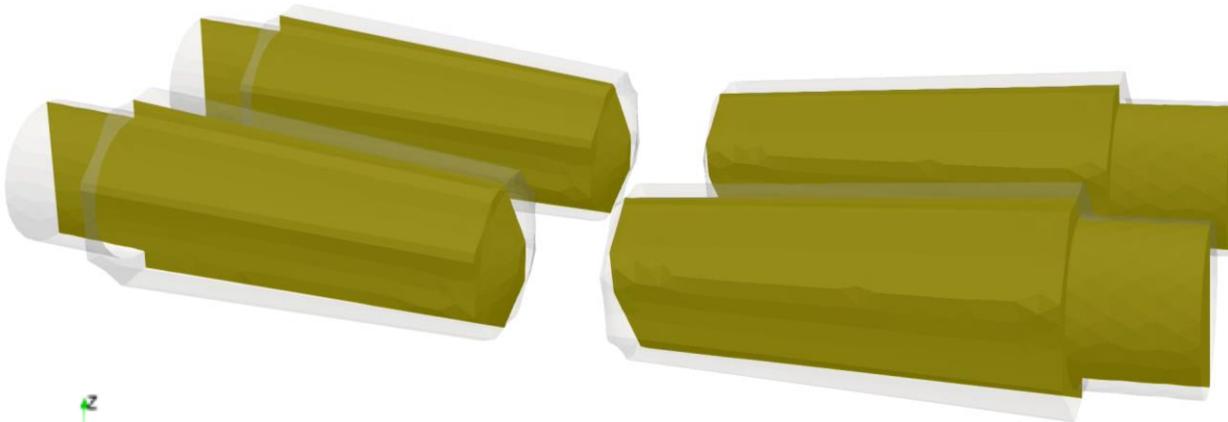
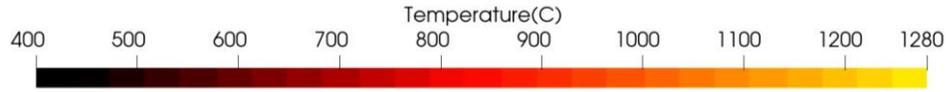
Obtained velocity & temperature fields.

- ◆ A heated ingot is replaced by another one at 400°C.
- ◆ Resolution of natural convection, forced convection, and radiation.
- ◆ The average velocity field is calculated for 100 increments after 10 seconds after the replacement.
- ◆ Resolution of the thermal solver for the remainder of the simulation.



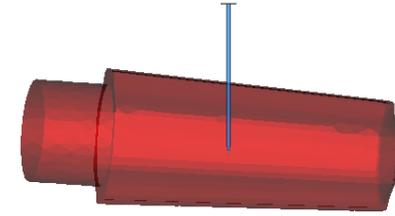
Results – Replacement of the 1st ingot

Temperature Evolution

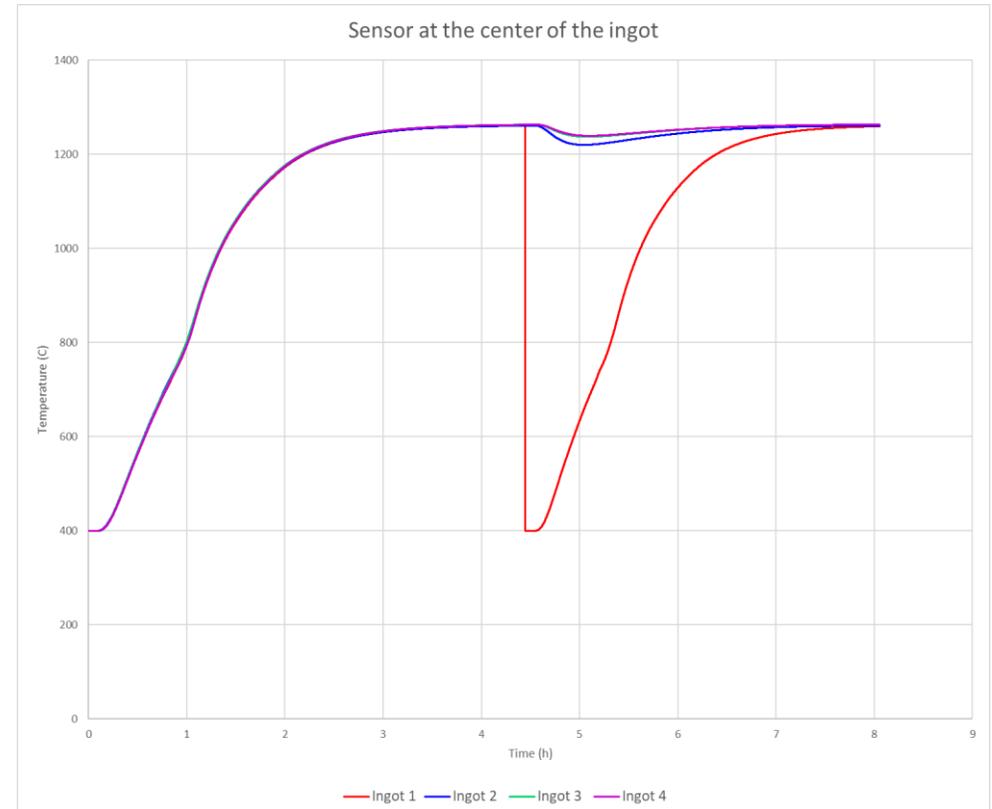


Time: 16000.000 s

Temperature evolution in the ingots

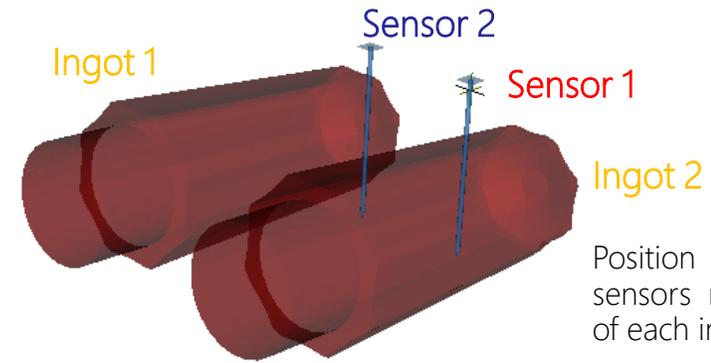


Position of the numerical sensor at the center of each ingot

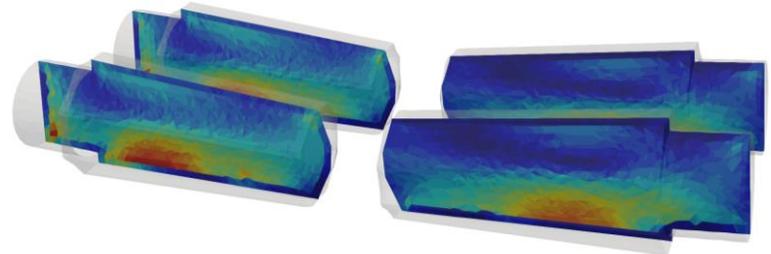
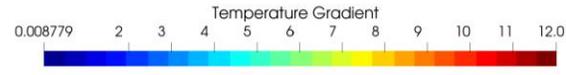


Results – Replacement of the 1st ingot

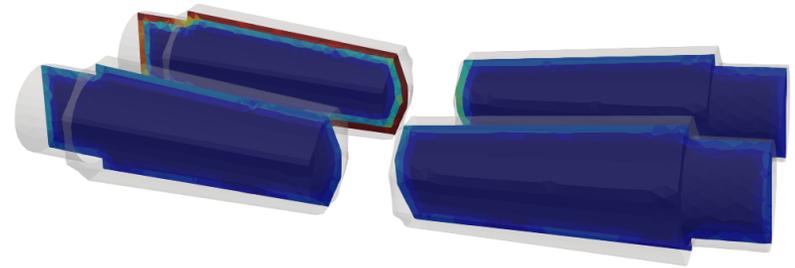
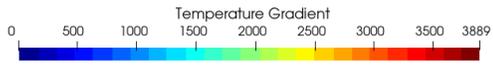
The replacement effect on the adjacent ingots



Position of the numerical sensors near the interface of each ingot



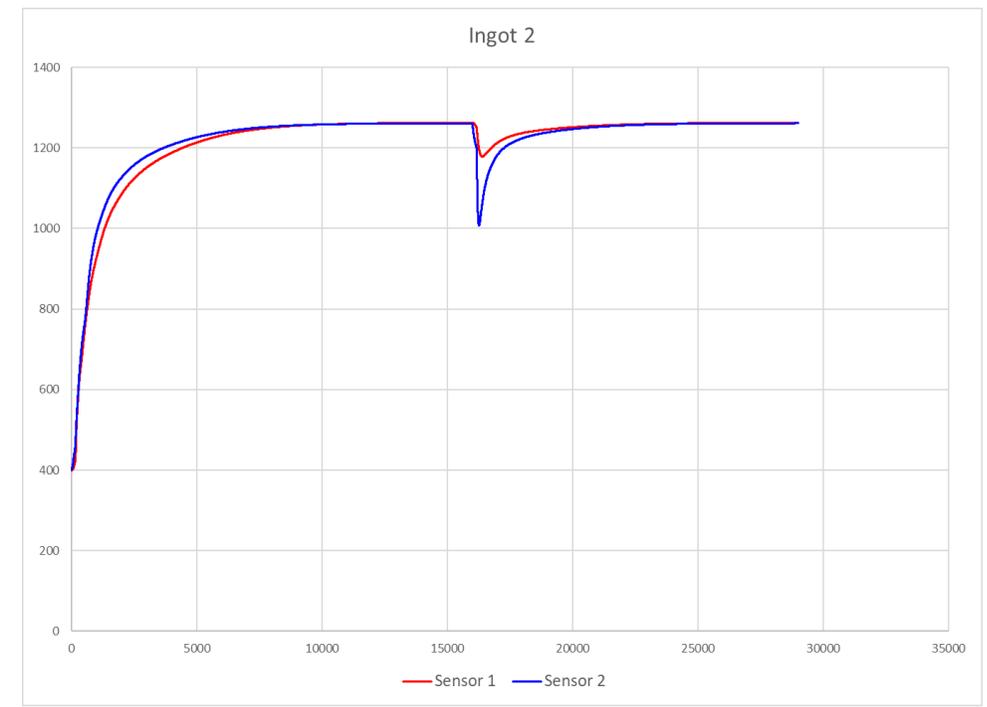
The temperature gradient in the ingots just before replacement at 4.4 hours.



The temperature gradient* in the ingots after replacement at 4.5 hours.

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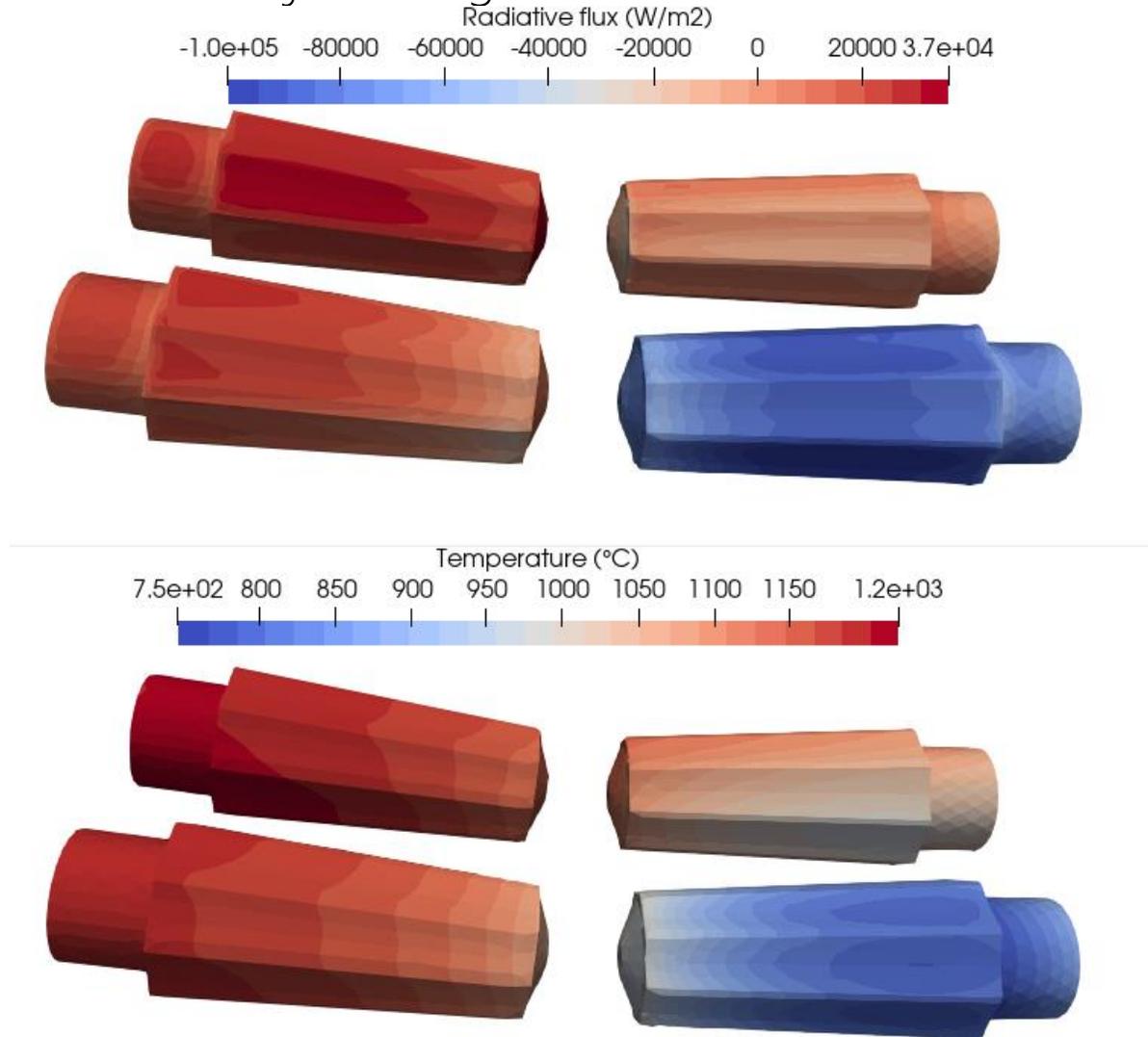
* Change of scale





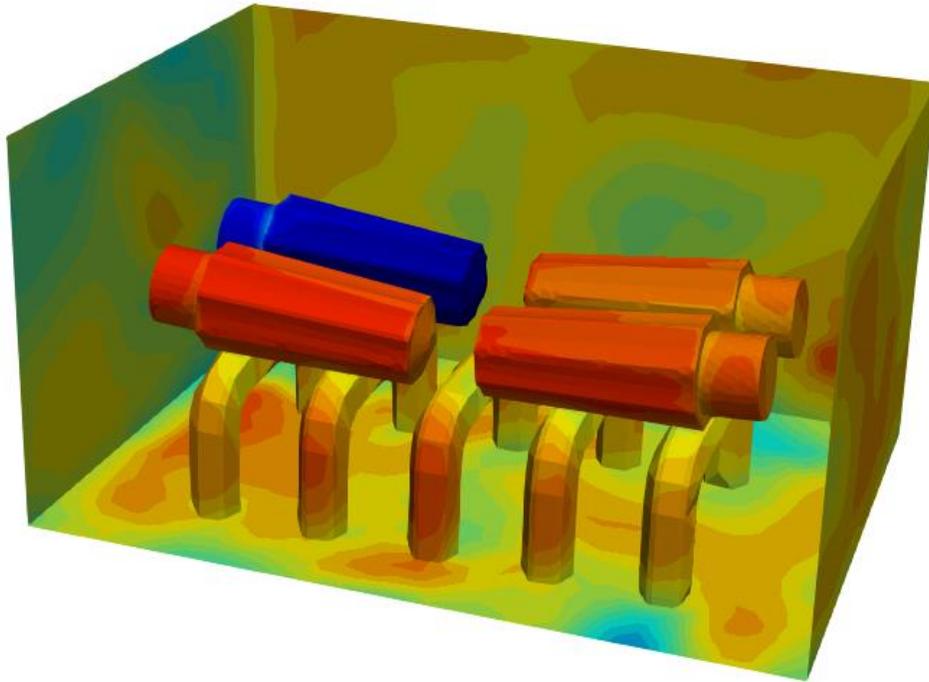
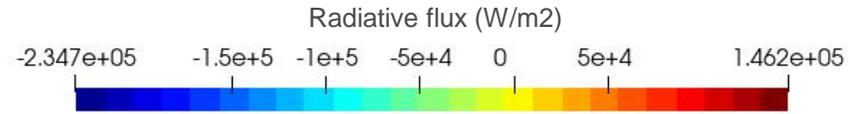
Results – Replacement of the 1st ingot

- ◆ The replacement effect on the adjacent ingots





Results – Replacement of the 1st ingot

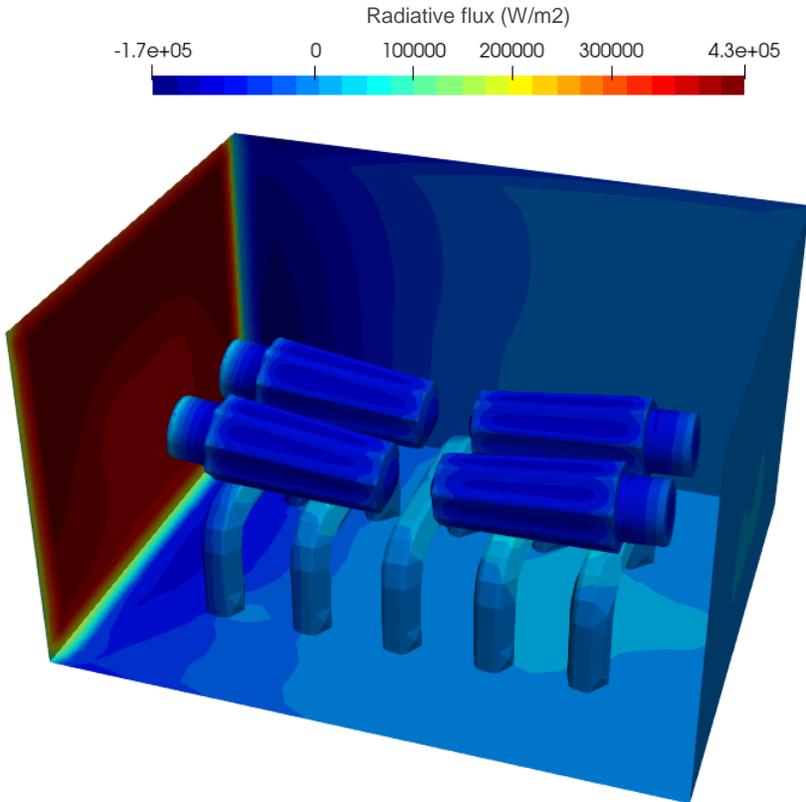


- ◆ Calculation of radiative flux and its evolution during the simulation.



Decarbonization: transitioning from gas to electric furnace

- ◆ With qobeo[®], the simulation of ingots in an electric furnace can also be performed :



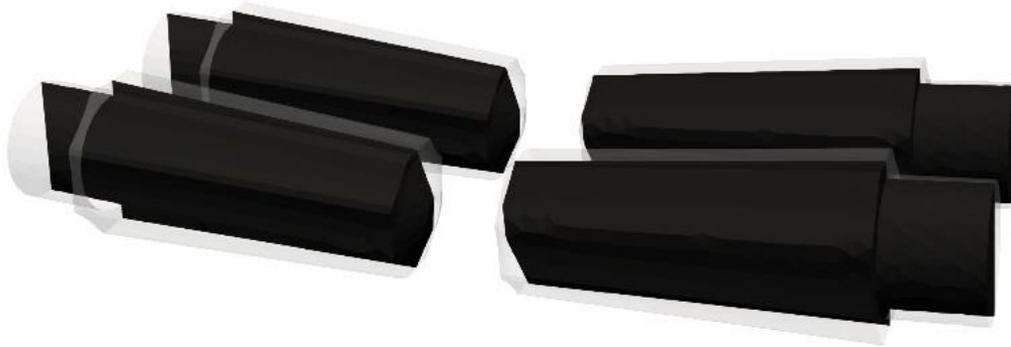
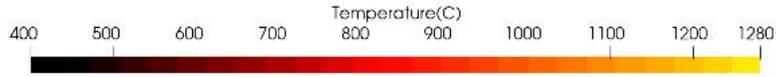
- ◆ Electric thermal resistances can be considered.
 - ◆ Furnace walls: $T_{walls} = 1200^{\circ}C$; $\epsilon_{walls} = 0,7$.
 - ◆ Fourier boundary conditions are applied on:
 - ◆ Closed door: $h = 5 \frac{W}{m^2.K}$; $T_{outside} = 50^{\circ}$; $\epsilon_{door} = 0,4$.
- ◆ After 4,4 hours, the 1st ingot is replaced with a "cold" one at $400^{\circ}C$.
- ◆ The radiative flux and its evolution can be calculated during the simulation.



Decarbonization: transitioning from gas to electric furnace

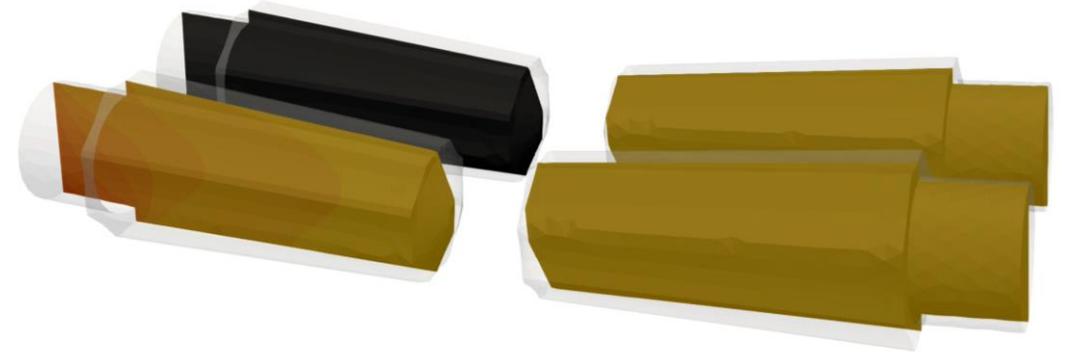
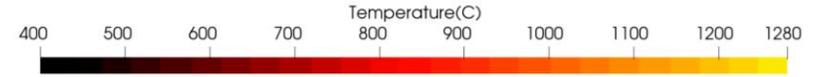
Temperature Evolution

From 0 to 4,4 hours



Time: 0.000 s

From 4,4 hours to 8 hours



Time: 16000.000 s

Fast Computation

Physical time	Number of cores	Calculation duration
8 hours	32 cores (@3,7 GHz)	~ 30 hours



Conclusion

- ◆ Simulate complex installations ✓
- ◆ Consider the physics' complexity ✓
- ◆ Achieve optimal heating cycles ✓
- ◆ Reliable and fast numerical analysis ✓
- ◆ With qobeo[®], the transition to decarbonization can be done in a smoother manner.

"It's always worth exploring numerical simulation to support your process optimization."



For any questions,
feel free to contact
our team



Sciences Computers Consultants
(Headquarters)

10 rue du plateau des Glières

F-42000 Saint Etienne

+33 (0)4 77 49 75 80

scc@scconsultants.com

<https://qobeo.scconsultants.com>

<https://www.scconsultants.com>